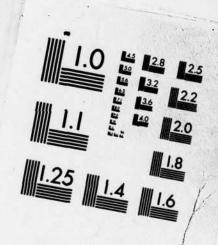
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NONCONTACT NONDESTRUCTIVE DETERMINATION OF PAVEMENT DEFLECTION UNDER MOVING LOADS M.E., Harr

N.T. Ng-A-Qui



Final Report August 1977



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PREFACE

This work was conducted under sponsorship of the Federal Aviation

Administration (FAA) as part of the runway roughness study to determine the effect of the loaded pavement profile on aircraft operations. An outgrowth of this study is a noncontact system to measure the pavement deflection basin under a moving load. This report describes the hardware developed and associated transfer function theory which forms the basis for a pavement evaluation and response scheme. (The effect of the loaded pavement profile will be the subject of a separate report to be published in September 1977.) Continued development of the noncontact system reported herein is being sponsored by the FAA under separate contract.

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LIST OF SYMBOLS

- equivalent mass
- equivalent damping
- equivalent stiffness
- pavement deflection (or signature),
velocity and acceleration, respectively
- loading function
- lateral distance from edge of tire
print
- signature
- maximum deflection at edge of wheel,
parameter of the maximum lateral dynamic
deflection basin
- parameter of the maximum lateral dynamic
deflection profile
- parameter of the maximum lateral dynamic
deflection profile
- maximum deflection at gage j
- modified input
- the reduced transfer function
- a complex variable
- time
- output
- time increment
- equivalency function
- time to peak deflection

LIST OF SYMBOLS (Concluded)

W(X)	- subgrade deflection at x from tire print		
p	- pressure on subgrade		
k	- modulus of subgrade reaction		
СН	- shear stiffness parameter		
Es	- subgrade modulus of elasticity		
CBR	- California Bearing Ratio		

SECTION I

INTRODUCTION

Today, personnel concerned with the evaluation of airfield pavements are confronted with three major problems. First, many primary pavements are old and/or are approaching early stages of deterioration. Second, the gross weight of aircraft is increasing continually, thus the demands on airfield pavements are also increasing. Third, procedures (References 1, 2, 3 and 4) used to evaluate airfield pavements are; either, destructive in nature, or, apply to only small areas of the pavement: in addition, they require considerable performance time and are therefore costly with respect to the flow of traffic.

The problem faced by the Air Force in this context has been stated by one author (Reference 5) who wrote,

"The Air Force alone owns enough pavement to be able to provide a 200-foot wide runway stretching from the state of Washington to the Southern tip of Florida. The problem of maintenance and rehabilitation of this inventory becomes more complex and critical each year. Most of the systems are over 20 years old . . . coupled with these aging pavements is the rapid growth of aircraft traffic and weights."

The need exists today for a method of pavement evaluation which would be able to quantify the stage of the aging process of a pavement and provide guidance with respect to its rehabilitation. It should be

able to do so considering the action of varying magnitudes and configurations of loads, and under varying ambient conditions.

Ideally, it should be nondestructive, rapid, permit the evaluation of the entire pavement with a minimum interruption to air traffic, and should be inexpensive and easy to use.

The problem of evaluating a pavement is a complex one. The pavement section consists of various materials. These materials are far from the ideal models of classical mechanics; and they vary diurnally, seasonally and with repetitions of loading. In addition, both vehicular and non-vehicular loads applied to the pavement-subgrade system vary in magnitude, intensity, and frequency.

Recognizing the nature and scope of the problem, and the need that exists today to overcome it, research activities were initiated at Purdue University toward the development of a nondestructive pavement evaluation capability to account for the complexity and variability of the pavement-subgrade system, and simultaneously satisfy the demands imposed by practical considerations; such as mobility and ease of operation of equipment. Research in this endeavor is predicated on the use of transfer function theory as the key to pavement evaluation. This methodology has evolved over a 15 year period from concept, through theory and laboratory studies to field investigations using fixed installations (References 6, 7, 8, 9, 10, 11, 12 and 13).

SECTION II

LITERATURE REVIEW

The need for improving methods and developing theories to quantify adequately the complex mechanism of pavement-subgrade interaction has generated considerable activity in the area of pavement analysis and evaluation in general, and nondestructive testing in particular. In evaluation procedures, efforts are being made to eliminate or reduce destructive testing. In nondestructive testing emphasis is currently being put on the use of surface deflection measurements. In analysis, attempts are being made to deviate from the classical approaches and use probabilistic and stochastic models. Because in the recently published literature several excellent state of the art papers and literature reviews already exist (References 14, 15, 16 and 17), no attempt will be made here to repeat these works, however, a synoptic overview of the principal works is presented.

2.1 Airfield Pavement Evaluation

The evaluation of airfield pavements can be categorized as destructive or nondestructive.

2.1.1 Destructive Evaluation

These procedures, as the name implies, destroy parts of the system; and, generally, involve the use of test pits, laboratory and/or in situ testing. Five procedures, with minor modifications are currently in use. These are the procedures of the U. S. Federal Aviation Administration (FAA) (Reference 1); U. S. Navy Bureau of

Yards and Docks (Reference 2); the Canadian Department of Transportation (Reference 3); the British Department of the Environment (Reference 4) and the U. S. Department of the Army and the Air Force (Reference 18). Basically, the five procedures involve the use of field tests in conjunction with sampling and laboratory testing.

The FAA procedure uses California Bearing Ratio (CBR) tests conducted in accordance with procedures in MIL-STD 621A Method 101.

The Navy method uses non-repetitive plate load tests conducted according to specifications of American Society for Testing and Materials (ASTM) Test Method-Designation D-1196 (Reference 19). The Canadian practice uses repetitive plate bearing tests specified in ASTM

Test Method Designation D-1195 (Reference 20). The British procedure also uses repetitive plate bearing tests but these are conducted in a slightly different manner from the ASTM D-1195 Test Method.

The Air Force evaluation procedure is summarized in the report of the testing program conducted in a recent (1975) evaluation. It was reported (Reference 21) that,

"Field testing consisted of 28 exploratory test pits . . . Thickness measurements were made on each pavement component, in situ density and moisture contents were determined, values of Modulus of Subgrade Reaction and CBRs were obtained and bulk samples of soil layers were taken. Core samples were extracted at 184 locations to ascertain pavement thickness and subsurface profile. Laboratory testing included classification of soils;

development of soil moisture-density relationships;

Marshall testing of asphaltic concrete; and tensile

splitting of Portland cement concrete cores."

The period of the evaluation was about 8 days.

The prevalent feeling concerning the use of test pits, and current destructive evaluation procedures was quite aptly put when one author (Reference 22) wrote,

"Yet most civil engineers still cannot scientifically evaluate the load carrying capacity of an airport pavement without first rendering it unserviceable in a way no aircraft could have accomplished. The resulting test pits when filled and resurfaced remain as abrupt discontinuities to uniform pavement performance."

This general feeling coupled with the many other disadvantages of destructive testing has directed research towards the development of rapid, nondestructive evaluation equipment and procedures.

2.1.2 Nondestructive Evaluation

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Nondestructive procedures are based on the analysis of the measured surface deflection response of a pavement-subgrade system to loads applied at the surface. The main differences among existing methods are: (1) the magnitude and manner in which the load is applied to the surface, and (2) the particular aspect of the surface disturbance that is measured, i.e., deflection magnitude, wave length, frequency. Table 1 shows a summary of the major load classes and their

TABLE 1 SUMMARY OF NONDESTRUCTIVE TESTING EQUIPMENT AND
PERTINENT CHARACTERISTICS

Category	Load Class	Device	Deflection Measurement Made
1	Static	Plate (19) ^a	Magnitude
		Plate (20)	Magnitude
2	Impact	Washington State (58)	Magnitude
		Paris (58)	Magnitude
		German (58)	Magnitude
3	Vibratory	Shell (23)	Wave Length
		Road Rater (25)	Magnitude
		Dynaflect (23)	Magnitude
		Corp of Engr. (25)	Magnitude
		AFCEL (68)	Wave Length
4	Vehicular	Benkleman (69)	Magnitude
		California (55)	Magnitude
		Lacroix (70)	Magnitude
		TRRL (71)	Magnitude
		South Africa (23)	Curvature

a Numbers refer to references

characteristics, devices being developed and in use, and the particular aspect of the surface disturbance that is measured.

In the static load class repetitive or non-repetitive loads are applied to plates of varying sizes on the surface of the pavement. In the impact load class, weights are allowed to free-fall and impact a plate on the pavement surface. The weights may range in size from 20 lbs (Reference 23) to 500 lbs, as was used by Isaada (Reference 24). The vibratory procedures also show a considerable range in magnitude of vibratory loads and frequency of vibration. Peak vibratory loads may range from 0.75 to 25 kips and frequencies may range from 1 to 5000 hertz. Green and Hall (Reference 25) conducted a thorough study of vibratory equipment. Their work and the references cited in Table 1 provide additional details. Of the devices that use vehicular loads, the Benkleman beam measures rebound deflection, and the LaCroix and California deflectometer are somewhat automated Benkleman beams. The TRRL device employs an optical (non contact) displacement transducer for measuring deflections, and the South African apparatus measures curvature of the pavement as the wheel approaches.

2.2 Analysis

Numerous analytical solutions exist, employing various models, representing a pavement-subgrade system. These models are based on the theories of elasticity and viscoelasticity. Numerous equations representing their behavior under static and dynamic loading have also been solved (References 26 through 40). In 1972, Boyer (References 11) presented a rather definitive review of these solutions. In

addition, he employed the concept of transfer function theory along with the results of full scale field testing, using gages installed in the pavement. He proposed an "a posteriori" modelling concept to develop transfer functions and predict pavement response. Highter (Reference 13) reviewed the use of energy concepts and methods, and failure criteria relative to pavement evaluation. Using energy concepts he was able to substantiate that "a functional relationship exists between cumulative energy as measured by cumulative peak deflections imparted to a given pavement system and the condition of the system". Three sources of data were studied to verify his hypothesis. AASHO Road Test data, the Air Force base traffic records from Pease and Castle Air Force Bases, and data from field testing of overlays specifically constructed for his investigation.

A theoretical study of dynamic stiffness and its application to vibratory nondestructive testing of pavements was conducted by Weiss (Reference 41) in conjunction with the study by Green and Hall (Reference 25). Weiss developed a nonlinear vibration theory for pavements and gave a method of obtaining the shear modulus and thickness of each pavement layer. In this work a vibrator is used to generate the deflections from which the dynamic stiffness is calculated.

Wiseman (Reference 42) used the Hogg and the Hertz models to estimate pavement characteristics. The Hogg model provided the required pavement stiffness for full-depth asphalt airfield pavements. He then related the computed stiffness to projected traffic and a characteristic length using published Asphalt Institute procedures.

Wiseman also used the Hertz model to compute the single wheel load on a rigid pavement as a function of measured stiffness.

A probabilistic analysis of the response of pavement structures was undertaken by McCullough (Reference 17). He cited the application of stochastic process, in this regard by the Texas Highway Department (Reference 43), California Division of Highways (Reference 44) and the Asphalt Institute (Reference 45). He also illustrated the application of the Markov process and its theoretical relation to states of a pavement. Reliability techniques have also been reported by Darter and Hudson (Reference 46). In 1976, Moavenzadeh (Reference 47) presented a stochastic simulation model for predicting pavement performance. Recognizing the limitations of his model, he concluded that it can be used to compare various design alternatives. It also accounts for geometry, load area and intensity, random inter-arrival times, statistical variation in material properties and temperature histories.

Inherent in all analytical and stochastic models is the notion that parameters can be obtained that reflect the actual vehicular load experienced by a pavement subgrade system. At the present time, the literature shows that parameters used in analysis are obtained by static field test, laboratory testing, vibrations, impact and measurement of pavement rebound characteristics. The actual time dependent response of the pavement-subgrade system subject to one complete vehicular pass is not considered.

SECTION III

THEORY AND METHODOLOGY

There exist today numerous elastic and viscoelastic solutions

(References 26 through 40) to the overall vehicle-pavement-subgrade

interaction problem, in addition to the results of several full scale

field testing programs (References 48 through 50). Accumulated

theory and prototype testing point to the following fundamental

observations:

- 1. The characteristics of the component materials of a pavement-subgrade system vary locally, even within the small volume of a representative "homogeneous" sample.
- Pavement-subgrade characteristics and response to loadings change with seasonal and ambient conditions.
- Load repetitions affect the various components of the system differently.
- 4. A nondestructive field test conducted in situ using prototype loading better reflects the action of a pavement-subgrade system than destructive tests on samples, or tests conducted in situ using non-representative loads.
- 5. The pavement subgrade system exhibits three basic behavioral characteristics; namely,
 - (a) inertial characteristics,

- (b) elastic characteristics, i.e. linear and independent of time, and
- (c) viscoelastic characteristics that account for the memory and the time dependent nature of the system response.

Recognition of the complexity and variability of the pavement-subgrade system and the nature of the wheel loading-pavement-subgrade interaction problem, has led to the development of global methods of characterizing the pavement-subgrade system under prototype conditions of loadings (References 38, 51, and 52). The global concept provides a quantitative spatial evaluation rather than one limited to the very near vicinity of a point and the subsequent assumptions of homogeneity and isotropy in the case of the classical theories (References 11 and 12).

3.1 Hypotheses

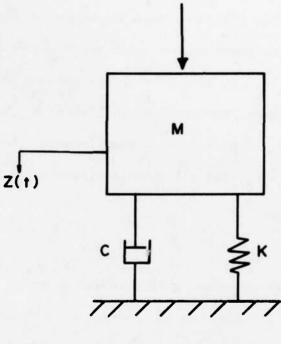
The principal hypotheses embodied in this research are:

1. The response of a point on the surface of a pavement-subgrade system when subjected to vehicular loadings can be characterized by a (Kelvin) lumped parameter model. The three parameters of the model are m, c, and k, as shown in Figure 1. The equation describing the hypothesized lumped parameter system is

$$m\ddot{z}(t) + c\dot{z}(t) + kz(t) = F(t)$$
 (1)

where m represents the equivalent mass (the interacting

F(t) Equivalent Loading



 $m\ddot{z}(t)+c\dot{z}(t)+kz(t)=F(t)$

Figure 1. Kelvin Model of the Pavement Subgrade System

inertial components of the pavement-subgrade system), c is the damping (the time dependent viscoelastic or energy dissipating component), and k is the stiffness (elastic component); z(t), z(t) and z(t) are the equivalent acceleration, velocity and deflection, respectively. F(t) is the equivalent input force due to the wheel load, (its maximum value for the F-4 aircraft, for example, is approximately 25 kips). Figure 1 shows the hypothesized model.

- 2. The salient characteristics of the time dependent response of the pavement-subgrade system to a complete vehicular pass can be obtained in a nondestructive manner.
- 3. Equivalent parameters describing the pavement-subgrade system stiffness (k), energy dissipation (c) and inertial (m) characteristics can be extracted from the measured response.
- 4. The extracted parameters accomplish the following:
 - (a) they reflect the global structural
 state of the pavement-subgrade
 system and; whereas, they are responsive
 to seasonal and local variations, they
 can be used as the basis of a
 nondestructive evaluation scheme.

- (b) they can be used to predict the deflection response to varying magnitudes and configurations of wheel loads.
- 5. Many current parameters used in conventional design and analysis can be obtained from the noted response of a pavement-subgrade system to actual vehicular loading.

The developed theory and methodology provide the following: (1) the theoretical basis of a pavement evaluation scheme. This includes the determination of the deflection response of a pavement-subgrade system to prototype loading, and the methodology of extracting parameters describing the equivalent mass, stiffness and energy dissipation characteristics of the pavement-subgrade system. (2) The methodology for predicting the deflection response of flexible pavements to prototype loadings of varying magnitudes and configurations.

3.2 Evaluation

The pavement evaluation scheme consists of determining the parameters inherent in the transfer function of the pavement-subgrade system. The transfer function is a mathematical expression which represents the mechanism that converts the equivalent input to an output. Figure 2 shows an overview of the input-transfer function-output interrelation as it applies to wheel load-pavement-subgrade interaction.

The basic elements of the evaluation scheme are:

(a) the signature determination, and

(b) the parameter determination.

3.2.1 Signature Determination

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The signature is the pavement deflection-time plot corresponding to the pavement deflection at the outer edge of the tire print as shown in Figure 3. Because the mechanical setup of the measuring equipment precludes the measurement of deflections closer than 3 inches from the outer edge of the tire print, the signature is calculated from deflections measured in the field using the "stationary beam-moving load" procedure.

In this procedure (Figures 3 and 4) a cantilever beam with its pivot outside the deflection basin supports sensing elements within the deflection basin. The sensing elements measure relative motion between the pavement surface and the beam as the load vehicle travels in a direction perpendicular to the beam position. The sensing elements are positioned at different lateral distances from the load wheel and therefore record the deflection time history for one complete pass of the load vehicle at the points of measurement. The location of the tire print is determined from the impression the tire makes on silver adhesive tape placed on the pavement in its path in front of the beam. The distance from the outer edge of the tire print to the first gage is measured. From this initial measurement and the fixed intergage distances, the distance of each gage from the outer edge of the tire print is calculated [Figure 3(b)]. Figure 4 shows an Air Force P-2 fire truck about to make a pass by two stationary measurement systems.

The method of calculating the signature is as follows:

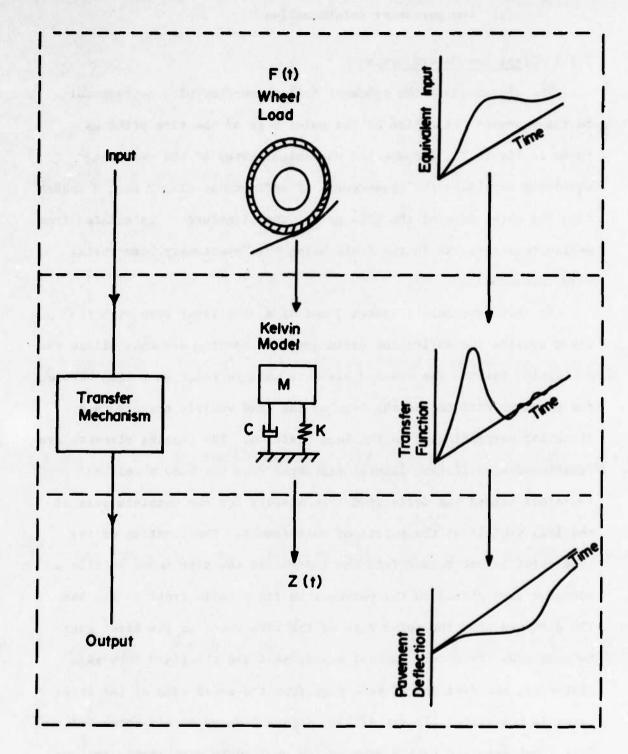
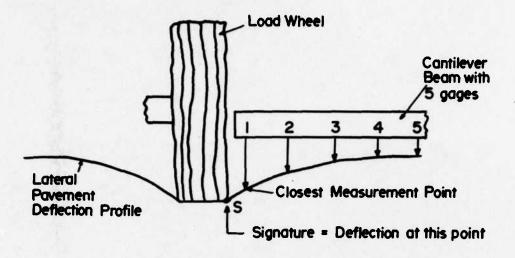
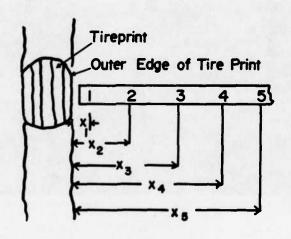


Figure 2. An Overview of the Transfer Function Concept as Applied to Wheel-Pavement Subgrade Interaction



(a) Profile



(b) Plan View

Figure 3. Location of Signature and Relative Gage Location and Setup



Figure 4. Setup During Stationary Beam-Moving Load Operation - P-2 Fire Truck

- (a) The maximum value of the deflection, z_{max} , at each particular gage (j) is obtained from each of the deflection-time plots recorded at said gage (j) during one pass of the vehicle. V and W of Figure 5 show z_{max_1} and z_{max_2} (z_{max_3} , z_{max_4} and z_{max_5} are not shown), in perspective.
- (b) The distances x_j [Figure 3(b)] from the outer edge of the tire print to each gage as recorded in the field, and the corresponding maximum gage readings, z_{max_j}, are then fitted with a curve given by the equation

$$z_{\text{max}_{j}} = A_{\text{peak}} \exp (\beta_{\text{peak}} x_{j}^{\text{r}_{\text{peak}}})$$
 (2)

where $A_{\rm peak}$, $\beta_{\rm peak}$ and $r_{\rm peak}$ are parameters describing the attenuation of the maximum values of the deflection-time plots recorded on each gage. Equation (2) is similar in form to those used by Boyer (Reference 11) and Highter (Reference 12), but has three parameters, $A_{\rm peak}$, $\beta_{\rm peak}$ and $r_{\rm peak}$, instead of two that they used. Baladi (Reference 52) also uses a three-parameter equation to describe the attenuation of the maximum lateral deflection profile caused by vehicular loads on highways.

The curve fitting calculations are performed by the computer subroutine EXACFIT listed in Appendix C. The equations used are derived as follows:

Given the general equation of the curve in the form

$$z = l \exp(\beta x^{T})$$
 (2a)

This equation has three unknowns, A, β and r. Using this basic equation and values of (x_1, z_{max_1}) , (x_2, z_{max_2}) and (x_3, z_{max_3}) , (where the numbers in the subscripts refer to gages numbered from 1, the closest to the wheel, to 5 the farthermost from the wheel), the following are had:

$$z_{\text{max}_1} = A'_{\text{peak}} \exp (\beta'_{\text{peak}} x_1^{\text{r}_{\text{peak}}})$$
 (3a)

$$z_{\text{max}_2} = \Lambda^{\dagger}_{\text{peak}} \exp (\beta^{\dagger}_{\text{peak}} x_2^{\text{peak}})$$
 (3b)

$$z_{\text{max}_3} = A_{\text{peak}}^{\dagger} \exp (\beta_{\text{peak}}^{\dagger} x_3^{\text{rpeak}})$$
 (3c)

Dividing equation (3a) by equation (3b) gives

$$z_{\text{max}_1}/z_{\text{max}_2} = \exp (\beta_{\text{peak}}' x_1^{\text{r'peak}} - \beta_{\text{peak}}' x_2^{\text{r'peak}})$$

Taking the natural logarithm of both sides produces

$$\ln z_{\text{max}_1} - \ln z_{\text{max}_2} = \beta'_{\text{peak}} (x_1^{\text{r'peak}} - x_2^{\text{r'peak}}) \qquad (4)$$

Similarly from equation (3a) and (3c)

$$\ln z_{\text{max}_1} - \ln z_{\text{max}_3} = \beta_{\text{peak}}' \left(x_1^{\text{r'peak}} - x_3^{\text{r'peak}} \right) \tag{5}$$

Dividing equation (4) by equation (5) and rearranging terms gives

$$(x_1^{r_{peak}} - x_3^{r_{peak}}) \left[\frac{\ln z_{max_1} - \ln z_{max_2}}{\ln z_{max_1} - \ln z_{max_3}}\right] = x_1^{r_{peak}} - x_2^{r_{peak}}$$
 (6a)

Equation (6a) is solved by an iteration-interpolation procedure detailed in Appendix C, and r'_{peak} is found.

 β'_{peak} is then calculated using equation (5), where

$$\beta_{\text{peak}}^{\prime} = \frac{\ln z_{\text{max}_{1}} - \ln z_{\text{max}_{3}}}{x_{1}^{\prime} + \sum_{\text{peak}_{1}}^{\prime} - x_{3}^{\prime}}$$
(5a)

Then A' is calculated using equation (3a) where

$$A_{\text{peak}}^{\prime} = z_{\text{max}_{1}}/\exp \left(\beta_{\text{peak}}^{\prime} \times_{1}^{r'}\right). \qquad (3a)$$

Using (x_1, z_{max_1}) , (x_3, z_{max_3}) and (x_5, z_{max_5}) instead of (x_1, z_{max_1}) , (x_2, z_{max_2}) and (x_3, z_{max_3}) , equations (3a), (3b) and (3c) are again solved to find r''_{peak} , β''_{peak} and A''_{peak} . The desired r_{peak} , β_{peak} and A_{peak} are found using the equations

$$r_{peak} = \left(r_{peak}' + r_{peak}''\right)/2 \tag{7}$$

$$\beta_{\text{peak}} = \left(\beta_{\text{peak}}^{1} + \beta_{\text{peak}}^{"}\right)/2 \tag{8}$$

$$A_{\text{peak}} = \left(A_{\text{peak}}^{1} + A_{\text{peak}}^{1}\right)/2 \tag{9}$$

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r peak, β peak and A peak are the parameters which describe the maximum lateral deflection profile, partly shown in Figure 5 as the curve V'VW.

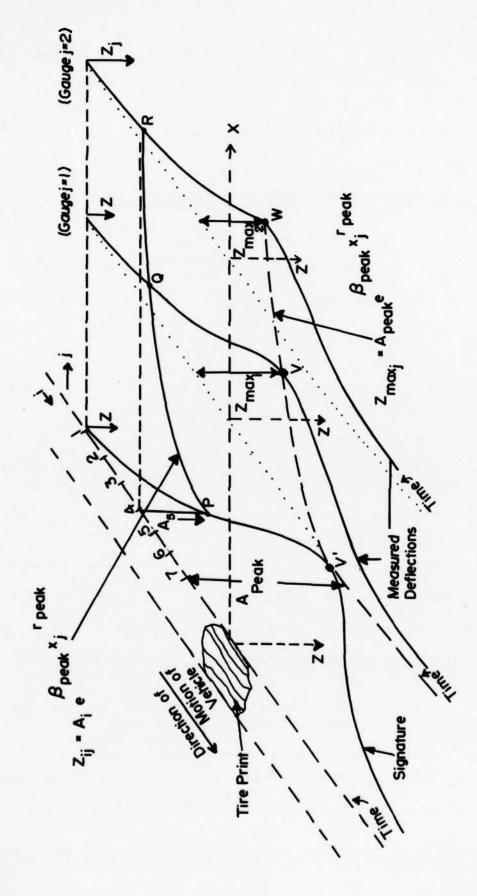


Figure 5. Overview of Method of Calculation of Signature Due to Prototype Loading

(c) The signature is calculated using the r_{peak} and β_{peak} parameters and the basic equation, equation (2a), in the form

$$A(t) = z_1(t)/\exp (\beta_{peak} x^{peak})$$
 (10)

where $z_1(t)$ refers to the deflection at gage 1 (closest to the wheel), and t is the time shown discretized as i in Figure 5. Subroutine SIGNTR in Appendix C performs these computations.

The signature thus calculated is the pavement deflection due to prototype loading at point S in Figure 3. Using this signature as the output (Figure 2), parameters k, c, and m describing the transfer mechanism of the three-parameter model of the pavement-subgrade system, are calculated.

3.2.2 Parameter Determination

The objective of the theoretical procedure is to determine the equivalent m, c and k parameters of a pavement-subgrade from the signature obtained from actual field loadings using transfer function theory. These parameters define the transfer mechanism that converts wheel loadings to pavement deflections.

The wheel load-pavement-subgrade interaction, as shown in Figure 2, can be represented mathematically by the equation (1):

$$m\ddot{z}(t) + c\dot{z}(t) + kz(t) = F(t)$$
 (1)

It can also be represented by the equation

$$I(t) * G(t) = \frac{F(t)}{m} * G(t) = z(t)$$
 (11)

where F(t) is the equivalent loading (input) function, m is the equivalent mass, G(t) is the <u>reduced transfer function</u>, z(t) is the response and the symbol (*) represents the mathematical operation of convolution (Reference 53). In its integral form equation (11) becomes

$$z(t) = \int_{0}^{t} \frac{F(t)}{m} \cdot G(t-\tau) d\tau \qquad (12)$$

G(t), the reduced transfer function in equations (11) and (12) is obtained from equation (1) as follows:

Consider the basic equation of the model:

$$m\ddot{z}(t) + c\dot{z}(t) + kz(t) = F(t)$$
 (1)

taking the Laplace transform of both sides, imposing initial conditions (pavement is assumed to be at rest prior to loading) of z(0) = z(0) = 0, and transposing terms, yields:

$$\vec{z}(s) = \frac{\vec{f}(s)}{m} \cdot \left(\frac{1}{s^2 + \frac{c}{m} s + \frac{k}{m}}\right)$$

where the bars denote the transformed function. The reduced transfer function is defined as

$$\bar{G}(s) = \frac{1}{s^2 + \frac{c}{m} s + \frac{k}{m}}$$

Inverting G(s) gives,

$$G(t) = \begin{bmatrix} \frac{1}{\frac{k}{m} - \frac{c^2}{4m^2}} e^{-\frac{c}{2m} \cdot t} \end{bmatrix} \begin{bmatrix} \sin \frac{k}{m} - \frac{c^2}{4m^2} t \end{bmatrix}$$
 (13a)

if $k/m < \frac{c^2}{4m^2}$, namely, the system is overdamped, then

$$G(t) = \begin{bmatrix} \frac{1}{2m} - \frac{c}{2m} \cdot t \\ \frac{c^2}{4m^2} - \frac{k}{m} \end{bmatrix} \begin{bmatrix} \sinh \frac{c^2}{4m^2} - \frac{k}{m} t \end{bmatrix}$$
 (13b)

Two ways to obtain the equivalent input (loading function) F(t) will be pursued in this work. The first way is by the direct substitution in the equation

$$m\ddot{z}(t) + c\dot{z}(t) + kz(t) = F(t)$$
 (1)

of the calculated values of z(t), z(t) and z(t) obtained from the measured deflections, and the parameters of the transfer mechanism m, c and k. The second method is by implicit convolution using equations (11) and (12). The implicit discrete form of equation (12) is given (Reference 53) by the equation

$$I(i) = \frac{k=i-1}{0(i)/\Delta T - \sum_{m=1}^{k=i-1} I(m) \cdot G(k+2-m)}$$
(14)

where G() = the reduced transfer function

$$0(1)$$
 = the output $z(1)$

$$m \times I(i) = F(i) =$$
the equivalent loading function (15)

ΔT = time increment

and, the initial value of I(i) = I(1) is given by the equation

$$I(1) = \frac{O(1)}{G(1) \cdot \Delta T}$$

The same two arguments above lead to two ways of finding the output z(t), given the forcing function and the parameters m, c, k. The first is by the closed form solution of equation (1). The second is by explicit convolution using the equation

$$O(i) = z(i) = \sum_{j=1}^{i} G(i) \cdot I(i - j + 1) \Delta T$$
 (16)

where z(), G(), I() and ΔT are as defined in equation (14).

During field testing, only the deflections with time are measured, and from these deflections the signature A(t) in equation (10) can be obtained as discussed previously. How then are the parameters of transfer frunction and the loading functions determined?

In concept, the methodology used is as follows: The signature A(t) is taken to be equivalent to the response z(t) in equations (11) to (16) as discussed in the previous section entitled "Signature Determination". The input function F(t) is now known initially. It is assumed however that the maximum value of F(t) is the wheel load, or approximately 25 kips (Reference 54) for the F-4 aircraft, which is the main vehicle of interest in the present study. Using the signature z(t), the maximum assumed value (25 kips) of the equivalent input, and the equations developed previously in this section, the k, c, and m parameters of the pavement-subgrade system are determined.

The detailed procedure for finding k, c, and m is as follows:

- (1) The signature is differentiated numerically to yield the time history of the velocity z(t) and the acceleration z(t). Typical plots of z(t), z(t), and z(t) are shown in Figure 6.
- (2) The components of the equivalent force, due to each of the three parameters, on the left hand side of the basic equation (1), are then considered for three special conditions, labelled as sections (1), (2), and (3) in Figure 7.

At section (1), the point of inflection of the signature, the following equations hold:

$$\ddot{z}(t_1) = 0$$

$$m\ddot{z}(t_1) = 0$$

hence,

$$c\dot{z}(t_1) + k z(t_1) = F(t_1)$$
 (17a)

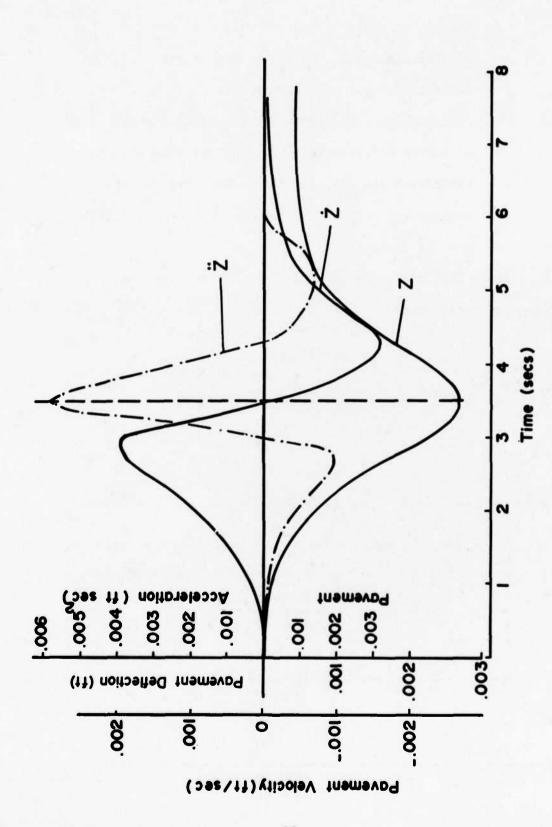
At section (3), the maximum deflection response, the following equations hold:

$$\dot{z}(t_3) = 0$$

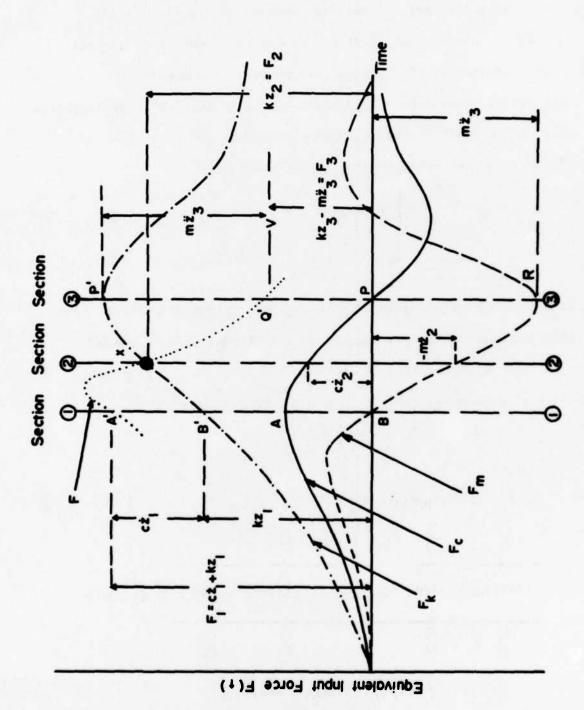
$$c\dot{z}(t_3) = 0$$

hence,

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Zero, First and Second Derivatives of the Measured Displacement Function (Typical) Figure 6.



Determination of Parameters of Transfer Function - A Graphical Overview of Methodology Figure 7.

$$m\ddot{z}(t_3) + kz(t_3) = F(t_3)$$
 (17)

The third special condition occurs, for each deflection data set, at a time between the time corresponding to the point of inflection [section (1)] and the maximum deflection [section (3)]. With reference to Figure 7, with increasing times from section (1) to section (3), the stiffness component [kz(t)] of the equivalent input force increases from B' to a maximum at P'. During this passage, the damping component [cz(t)] decreases from a maximum of A to zero (P). Similarly, the inertial component [mz(t)] decreases from zero (B) to a negative value of R. The time corresponding to the conditions,

$$c\dot{z}(t_2) = -m\ddot{z}(t_2) \tag{18}$$

and, hence,
$$kz(t_2) = F(t_2)$$
 (19)

will be called the crossing point. It provides the final special condition for determining the measure of the parameters m, c and k. This condition is represented as section (2) of Figure 7. It should be noted that between section (1) and the crossing point,

$$kz(t) \leq F(t_2)$$

and between the crossing point and section (3)

$$kz(t) > F(t_2)$$

In concept, therefore, the crossing point represents the time at which the Winkler hypothesis (Reference 55) is satisfied.

(3) Using the equation in paragraph (2) and the developed method, an iterative procedure, given in detail in Appendix C was used to determine the k, c and m parameters in this study. Briefly the procedure is as follows:

The signature of the load vehicle is obtained as described in the section entitled "Signature Determination". A first trial value of k is taken as a fraction (AA1 < 1.0) of the wheel load (25 kips) divided by the maximum value of the signature. Then it is assumed that another fraction (AA2 < 1)of the wheel load acts at the time corresponding to section (3) Figure 7. The first trial value of m is calculated for the conditions at section (3) using equation (17). The initial trial value of the c parameters is then computed using equation (18) at the first estimated crossing point in the data set, i.e., at the first discrete interval of time after section (1). Using these first estimates of k, c and m the transfer function G(t) [equation (13)] is had [EXACT 1]1. Next the loading function F(t) is obtained by implicit convolution [equations (14) and (15) and NWCONV] in the time domain using the signature deflection z(t) is output and G(t). The maximum value of F(t) is then found. A check is made to see if this (maximum) value of F(t) satisfies the necessary (500 pound) criterion that,

Capital letters in parentheses refer to computer programs or symbols used in computer programs in Appendix C.

If this criterion is not satisfied, the next crossing point is selected, c is recalculated and the same procedure is followed to obtain the maximum value of F(t), which is again examined with respect to the previously stated criterion. This process is repeated until the possible crossing points are exhausted i.e. section (3) Figure 7 is reached, or the maximum value F(t) satisfies the previously stated criterion, whichever occurs first.

If all possible crossing point times are exhausted a new value of m is calculated by incrementing the value of AA2. Using the first trial value of k and the second trial value of m a new c is calculated for the first estimated crossing point. The procedure is repeated, checking the maximum value of F(t) against the 500 pound criterion. If this criterion is not satisfied the sequence of crossing points is again stepped through. If these are exhausted a next trial value of m is assumed and the entire process is repeated. If the incremented value of AA2 reaches unity, a second trial value of k is calculated by incrementing AA1, and the entire procedure, including a new sequence of m and c values, is examined at all possible crossing points. This process is repeated until the 500 pound criterion (equation 20) is satisfied.

If the 500 pound criterion is satisfied, a new c value is calculated for conditions corresponding to the time at the inflection point of the signature (section (1), Figure 7, equation 17a); using the current k value and the force obtained by convolution, corresponding to the time of inflection. If this new c value does not correspond to the

c value which satisfied conditions at the crossing point and the 500 pound criterion, the iteration process is continued by selecting another crossing point and repeating the entire procedure. If the c values do correspond then the corresponding k, c, and m values are the desired parameters.

3.3 Prediction of Pavement Response (Signature)

The theory and methodology for signature and parameter determination developed in the previous sections were used to predict the signature of a vehicle (vehicle A) at a site prior to the actual passage of the vehicle at that location. This was done by first obtaining signatures of both a standard vehicle and vehicle A at another site, and the signature of the standard vehicle at the new site. This is accomplished by using an equivalency function.

The equivalency function is defined as the ratio of the loading (input) function of a vehicle, $F_1(t)$, and the loading function of the standard vehicle, $F_2(t)$, both obtained at the same location (at a standard site). This relationship is given by the equation.

Equivalency Function = EF(t) =
$$\frac{F_1(t)}{F_2(t)}$$
 (21)

The procedure for determining the equivalency function is as follows: with reference to Figure 8, Block I, at a standard site the signature due to a standard vehicle is obtained as discussed in the section entitled "Signature Determination". Using this signature as output, the transfer function, $G_2(t)$, of the standard site and the loading $F_2(t)$ of the standard vehicle at the standard site are determined as previously discussed in the section entitled "Parameter

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Determination". Next the signature of A (the vehicle whose response is to be predicted) is obtained at the standard site. Using the transfer function of the standard site as determined by the standard vehicle and the signature of vehicle A at the standard site, the loading function $F_1(t)$ of vehicle A is obtained by implicit convolution using equation 14 and 15, and subroutine NWCONV (Appendix C). Once the equivalency function is had from equation 21, the predicted signature is obtained as follows:

First, the signature of the standard vehicle, z(t), its loading function $F_{sA}(t)$ and the transfer function $F_{sA}(t)$ are obtained at the new site as discussed in the preceding paragraph. These functions are shown schematically in Block III Figure 9. Next the loading function $F_{1A}(t)$, of vehicle A at the new site is calculated using the equation

$$EF(t) \times F_{sA}(t) = F_{1A}(t)$$
 (22)

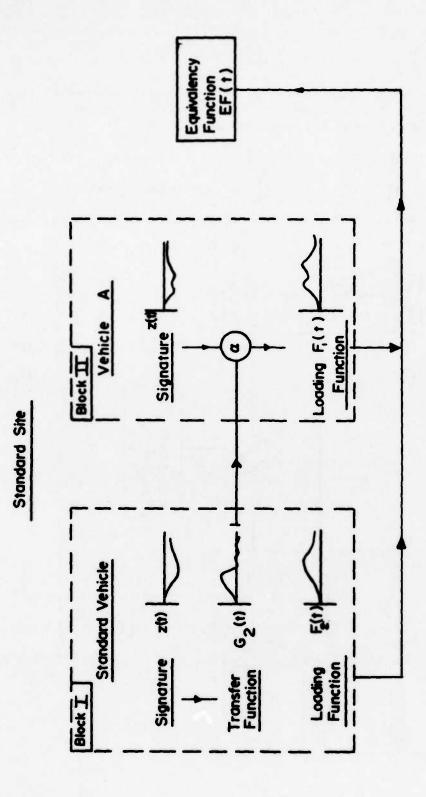
Equation 22 is represented in Figure 9 as the connection between the loading function of Block III and the equivalency function at the point labelled β .

The predicted signature of vehicle A at the new site is computed by the explicit convolution of the loading function of vehicle A at the new site and the transfer function of the new site as determined by the standard vehicle, using equation (16),

$$z(i) = \sum_{j=1}^{i} G(i) \cdot I (i - j + 1) \Delta T$$

where G(i) = transfer function of the new site

I() = the equivalent loading function of vehicle A at the new



Overview of Scheme for Determining the Equivalency Function Figure 8.

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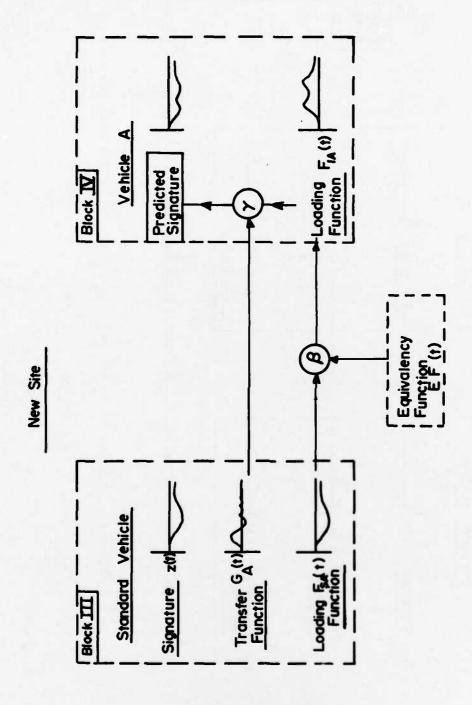


Figure 9. Overview of Scheme for Predicting the Signature

site (see equations 11 and 15)

z(i) = the predicted signature

 ΔT = the time increment

- j = dummy variable, and
- i = dummy variable indicating the time interval number

The explicit convolution is represented in Figure 9, Block IV as the connection between the equivalent loading and the transfer function at the point labelled γ of Figure 9.

The computer program PREDCT that performs the noted calculations to generate the predicted signature is given in Appendix C.

The evaluation and prediction methodologies are clearly dependent on the availability of reliable equipment for measuring pavement deflections.

SECTION IV

NEW INSTRUMENTATION FOR DEFLECTION MEASUREMENT

Two mobile systems were developed during this study for the measurement of pavement deflections: the Light Emitting Diode (LED) system and the Linear Variable Differential Transformer (LVDT) system.

The LED system is essentially an optical system which does not make contact within the deflection basin of the pavement. On the other hand the sensing elements of the LVDT system make contact with the pavement surface. A summary of the research effort expended in the development of these systems and their basic developments is presented below.

4.1 The LED System

The LED measurement system consists of three major operational sections: they are,

- a. the beam
- b. the sensor modules, and
- c. the recording system

a. The Beam

The beam is made of aluminum. Its principal function is to support the sensors above the pavement surface. It is designed so that a 20-mile-per-hour gust of wind will deflect the end of the beam less than 0.004 in. Figure 10 shows schematic details of the beam. Figure 11 shows the LED beam setup. Seven LED sensor modules are shown between the channel of the beam.

The LVDT system was developed by Baladi (Reference 52) as part of his research effort, sponsored by the Federal Highway Administration.

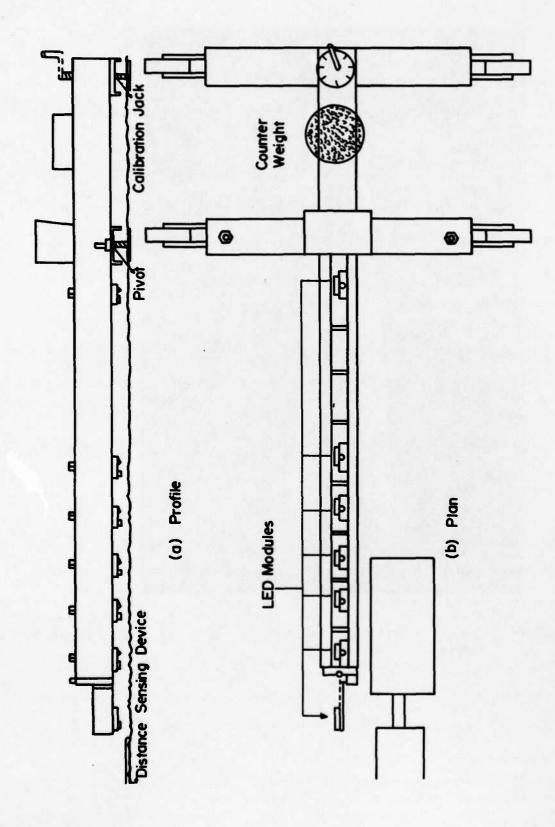


Figure 10. Light Emitting Diode (LED) Beam Schematic Details

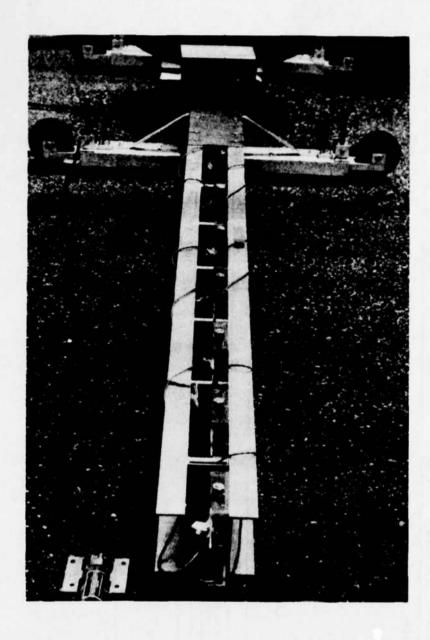


Figure 11. LED Beam Setup

b. Sensor Modules

Several ways of obtaining deflections using optical, non-contact techniques were considered prior to selection of the final sensor.

These methods can be categorized under two general classes; namely, point methods and global methods. The point methods considered were focal point displacement, beam triangulation, retro-reflector beam displacement methods, diffraction gages, interferometers, Moire gages and grating gages. The global techniques considered were holography and Moire contour techniques.

The technique selected for development was the beam triangulation method, using a light emitting diode as the coherent light source.

Figure 12 shows the basic geometrical optics of the heam triangulation method. Basically, pavement deflection is measured by the amount of displacement of the reflected light impinging on the collector lens.

Research has established that a line of light impinging on the surface of small chips of various materials, asphalt, concrete, and sand paper, produces a reflected signal that can be normalized to produce a linear position measurement independent of surface reflectivity. The sensors have a range of 0.2 in and an accuracy of 0.0005 in. Figure 13 shows a complete sensor module ready for mounting on the beam. Additional details are given in Appendix A.

c. The Recording System

Signals from the sensors are collected on a magnetic tape unit.

Figure 14 shows the recorder used during this investigation. Figure 15

⁴A light beam recorder was also used in some special tests.

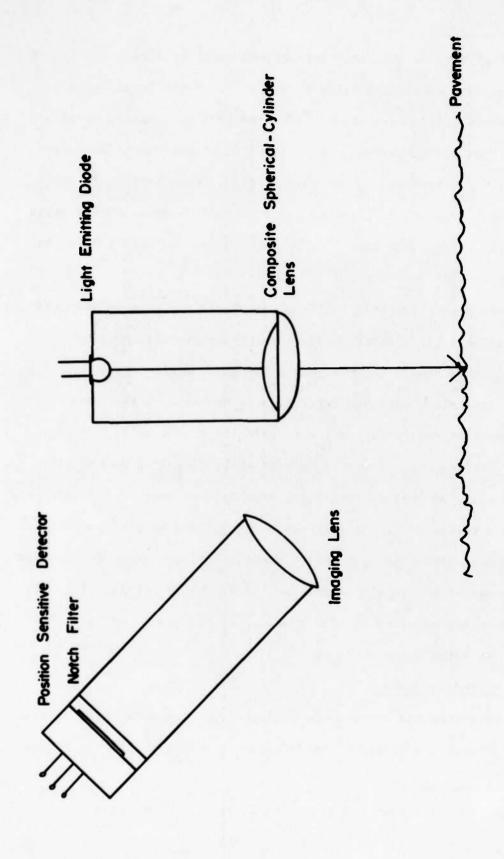


Figure 12. Light Emitting Diode Triangulation Arrangement

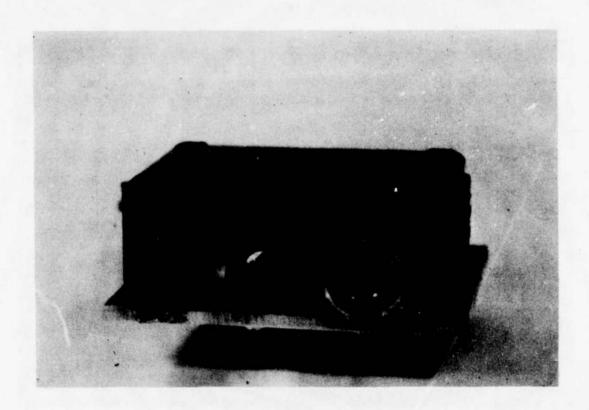


Figure 13. Typical LED Sensor Module

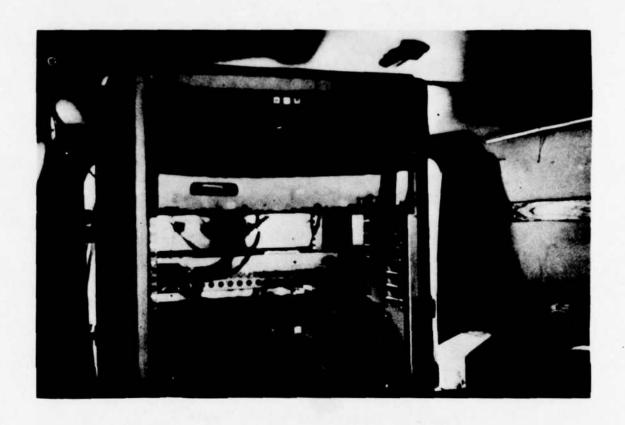


Figure 14. Magnetic Tape Recording Equipment

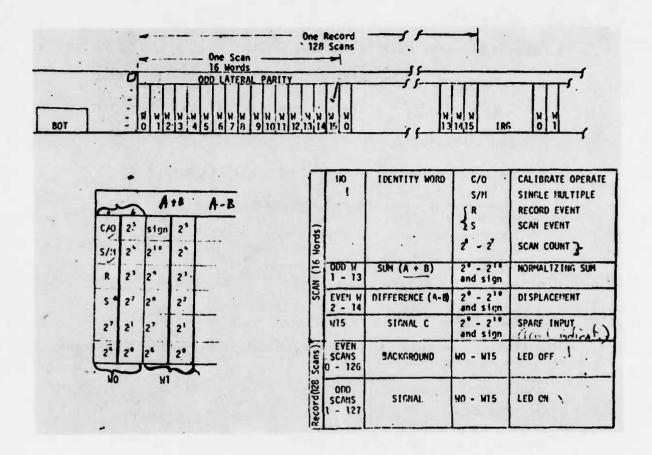


Figure 15. Arrangement of Data on Magnetic Tape (After Murphy [Reference 56])

demonstrates the basic arrangement of the data on the tape. Additional details on the tape decoding and data reduction computer programs are presented in Appendix C.

It should be noted that in the LED system:

- The output, after being recorded on the magnetic tape, must be fed into a digital computer for data reduction and analysis, and
- 2. No contact (excepting infrared light) is made between the sensing elements and the pavement surface. The LVDT system required some (minor) contact within the deflection basin of the pavement.

4.2 The LVDT System

The LVDT system consists of three principal operational sections. They are:

- a. the beam
- b. the sensor elements
- c. the recording system

a. The Beam

The beam is a modified wide flange aluminum I-beam. Its dimensions and principal features are shown schematically in Figure 16.

A picture of the beam, set up in position for measurements, is shown in Figure 17.

b. The Sensors

The sensors are LVDT's. These are fixed on the beam as shown in Figure 17, with the sensing elements in contact, through adjustable jacks, with the pavement surface. The relative movement between the

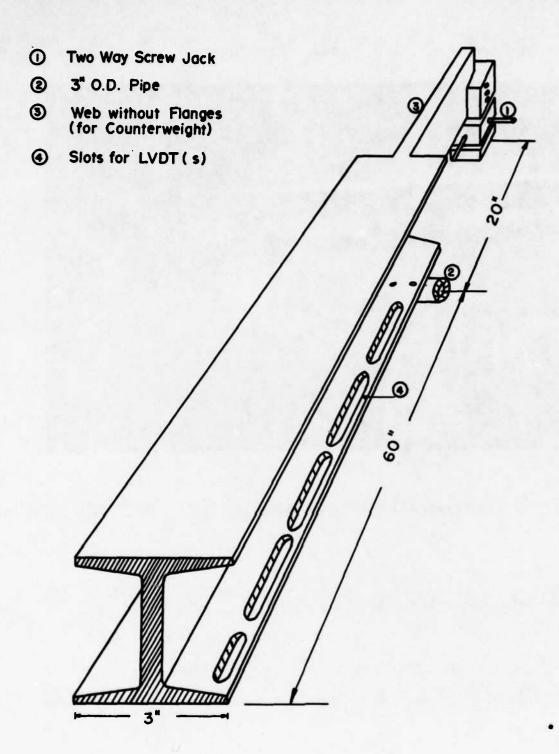


Figure 16. LVDT Beam Schematic Details

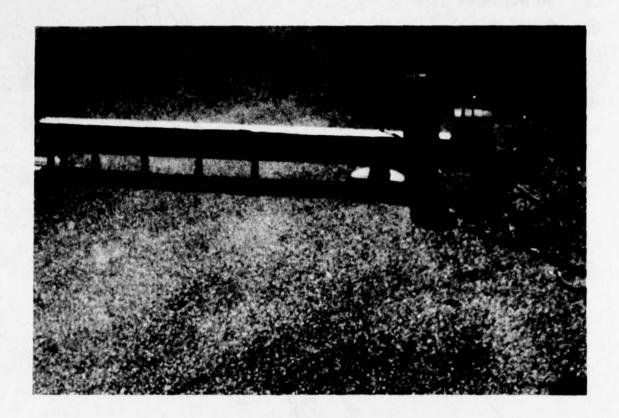


Figure 17. LVDT Beam Setup

core of the sensing element and the induction coil of the LVDT, fixed on the beam, produces a signal which is a measure of the displacement.

c. The Recording System

The LVDT recording system is a six channelled light beam recorder. Figure 18 shows the recorder used in this study. An analogue signal is recorded on light sensitive paper tape. The paper tape output can be converted to digital data by using a digitizing system (Reference 57) which automatically provides punch data cards.

It should be noted that, although the LVDT system requires contact between the pavement and the sensor, its simplicity, directness of data collection and ease of operation make it very adaptable for general use.

The LVDT and the LED systems were both used at various stages of this study.

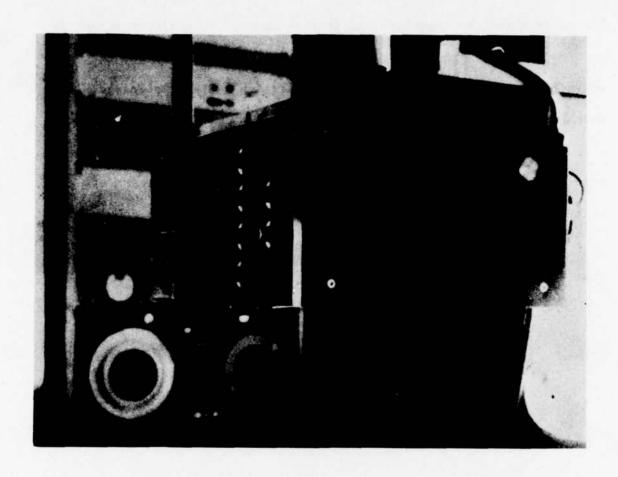


Figure 18. Project Light Beam Recorder

SECTION V

FIELD INVESTIGATIONS

The results of two series of field investigations were used in the validation of the evaluation and prediction schemes discussed previously. The first series of field investigations, Series A, was conducted during the course of this project. The second series, Series B, had been conducted in 1972 at Kirtland AFB and the basic data were reported by Boyer (Reference 11).

5.1 Series A Field Investigations

The principal objectives of this series of field investigations were:

- To verify that the new nondestructive testing equipment,
 i.e., LED beam (and LVDT beam), can provide measurements
 of pavement deflection consistent with destructive test
 measurements.
- 2. To use the NDT (nondestructive testing) equipment to obtain deflection measurements for several different vehicles at different sites, for the purposes of providing initial input into the evaluation scheme developed in conjunction with the NDT equipment.
- To show that the evaluation and prediction schemes are consistent with the factual information currently available.

The methodology evolved over the period of this study therefore reflects the need to satisfy the principal objectives.

The basic methodology with respect to equipment development, shown in Table 2, evolved over the period of this study. As can be seen from

TABLE 2. FIELD TESTING METHODOLOGY OF DEVELOPMENT

Stage	Method of Measuring Deflections	Features of Method	Distribution of Measurements
1.	Gages Installed in Pavement	(a) Destructive (b) Fixed	(a) Along one
2.	LVDT Beam	(a) Nondestructive (b) Contact (c) Stationary	(a) Along one line and/or Global
3.	LED Beam	(a) Nondestructive (b) Noncontact (c) Stationary	(a) Along one line and/or Global
ц.	LED Beam	(a) Nondestructive (b) Noncontact (c) Moving	(a) Global

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Table 2 the method of measurement of deflections progressed in four steps from destructive fixed instrumentation, namely gages installed in the pavement, to noncontact, nondestructive instrumentation. Details of the gage installations, LVDT beam and the LED measurement systems are presented in Appendices A and B.

Deflection measurements were obtained during each step using different vehicles at different sites. The developed instrumentation was used as available for measurement purposes as shown in Figure 19. Figure 19 also shows a synopsis of the observed characteristics of the test sites, instrumentation and loading vehicles used in the Series A field investigation. Details of the sites and testing program conducted at each site during each phase are presented in Appendix B.

It should be recognized that the sites selected reflect reasonably well the extremes of the range of the airfield pavement conditions.

Site 1 at Eglin AFB is a functionally failed flexible pavement and Site 2 is an in-service taxiway. The loading vehicles used also reflect a range of tire pressures using airfield pavements. The F-4 aircraft and the P-2 fire truck have tire pressures of 255 psi and 55 psi, respectively. The contrast in gross static wheel load also spans a reasonable range. The F-4 aircraft and the F-2 fire truck have gross single wheel 1 loads of approximately 25 kips and 8 kips, respectively. Figures 20 and 21 show the F-4 aircraft and the P-2 fire truck, respectively, and their pertinent characteristics. Figure 22 shows the F-4 load-cart loaded and unloaded. When loaded the central load wheel exerts a load on the pavement equivalent to a single main gear of the

⁵For the F-4, this refers to the main gear.

(a) EGLIN AFB: SITE 1

	used as parking area.					
Date	Loading	Instrumentation				
5-75	(a) P-2 Fire truck (b) F-4 Aircraft	(a) Gages installed in pavement				
11-75	(a) P-2 Fire truck (b) F-4 Aircraft	 (a) Gages installed (b) LVDT Beam (c) LED Beam 				
3-76	(a) F-4 Load Cart	(a) LVDT Beam				

Figure 19. Synopsis of Series A Field Investigation

(b) EGLIN AFB: SITE 2

	General Site Description: Few cracks: in use as a taxiway			
Date	Loading	Instrumentation		
5 -7 5	(a) P-2 Fire truck	(a) Gages installed		
11-75	(a) P-2 Fire truck	(a) Gages installed		
3-76	(a) F-4 Load Cart	(a) LVDT Beam		

Figure 19. (Continued)

(c) PEASE AFB

Gener	al Site Description:	Uncracked in-service runway: Rutted by several passes of KC-135A Aircraft and P-2 Fire truck
Date	Loading	Instrumentation
7 - 75	(a) P-2 Fire truck	

(d) EGLIN AFB: TAXIWAY

General Site Description: Cracked not-in service taxiway							
Date	Loading	Instrumentation					
6-76 ^b	F-4 Load Cart	(a) LED Beam ^C (b) LVDT Beam					

- a Stationary Beam Moving Load operation
- b 6000 feet of runway were also tested on this date.
- c Moving Beam Moving Load operation

Figure 19. (Concluded)

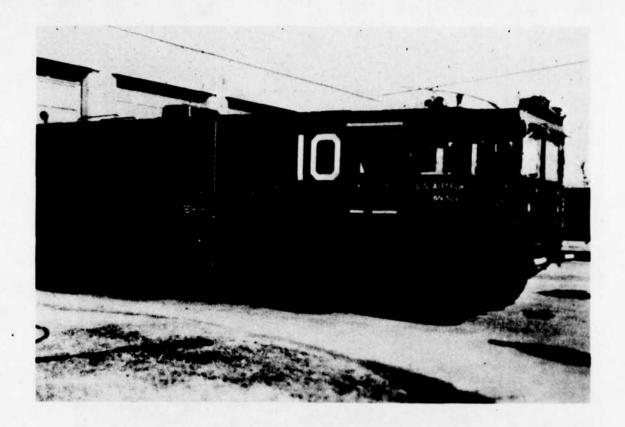


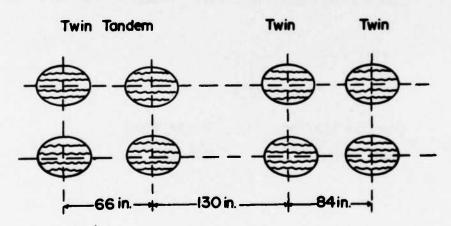


Gear Configuration
Test Weight

Single Wheel 25 Kips

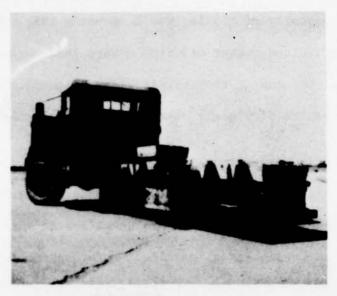
Figure 20. F-4 Aircraft and Pertinent Characteristics





STD. TEST WEIGHT 66,000 to

Figure 21. P-2 Fire Truck and Pertinent Characteristics



(a)Loaded



(b) Unloaded

Figure 22. F-4 Load Cart

F-4 aircraft. The majority of testing was done using this vehicle.

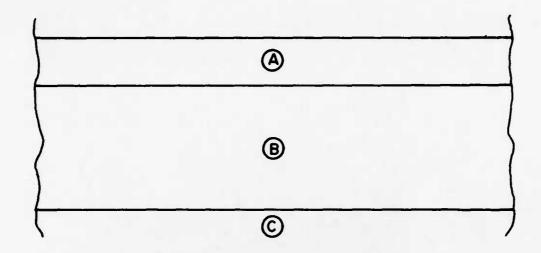
Because of the limited number of prime movers that were made available during the periods of testing, it became necessary to utilize some test results previously reported by Boyer (Reference 11).

5.2 Series B Field Investigations

The field investigations in Series B were conducted by Boyer at Kirtland AFB in New Mexico in 1972. In his investigation deflection measurements were made using gages installed in the pavement in a manner similar to the procedure used in the Series A field investigations. For details on his instrumentation and methodology the reader should refer to the original reference (Reference 11). The pavement characteristics at the three sites reported are shown in Figure 23. The prime movers and their characteristics are shown in Figures 24 through 26.

As previously stated the series B data provide additional information required to test the evaluation and prediction schemes. The gear configuration of the C-130, C-135 and C-131 provide, in addition to a variation in gross load and tire pressures, a loading function due to multiple and sequential wheel loads, and combinations thereof. The C-131 (single-twin) is an example of a multiple but not sequential loading function. The C-130 (single-tandem) is an example of a sequential but partially multiple, loading function. The C-135 (twin-tandem) is an example of sequential and a multiple loading

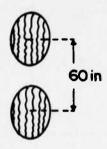
⁶The P-2 fire truck is an example of a purely sequential loading function.



Test Site No.	Taxiway No.	Asphaltic Concrete(in)		© Subgrade
-	6	3	9	Silt/Sand
2	2	6.5	8	Silt/Sand
3	8	10	6	Highly Compacted Silt/Sand

Figure 23. Site Characteristics Series B Field Investigations (Reference 11)



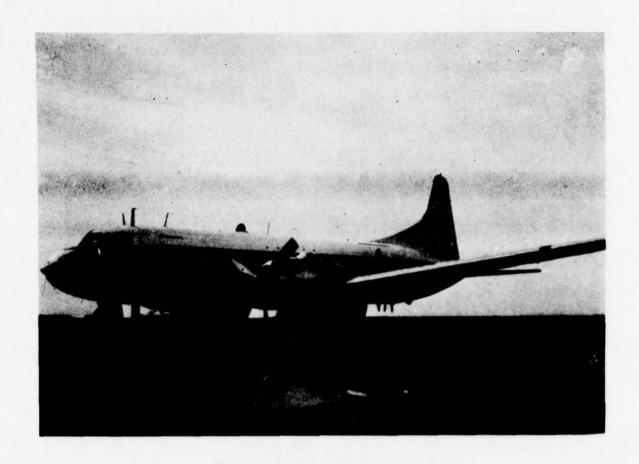


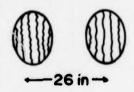
Gear Configuration STD. Test Weight

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Single Tandem 90 Kips (Gear load)

Figure 24. C-130 Aircraft and Pertinent Characteristics



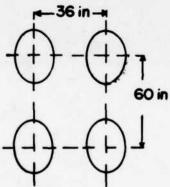


Gear Configuration STD. Test Weight

Single Twin 20 Kips (Gear load)

Figure 25. C-131 Aircraft and Pertinent Characteristics





Gear Configuration STD. Test Weight

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Twin-Tandem 210 Kips (Gear load)

Figure 26. C-135 Aircraft and Pertinent Characteristics

function.

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The data from the Series A and the Series B Field investigations were analyzed using the evaluation and prediction schemes previously presented.

SECTION VI

RESULTS

This research effort encompasses a broad spectrum of engineering activity ranging from the development of NDT equipment and a procedure for nondestructively evaluating flexible pavements, to the development of a methodology for predicting the deflection response of various pavements to different aircraft. This section presents the following:

- 1. typical results of equipment field verification tests,
- results of the evaluation of the two sites (Series A:
 Field Investigations) using the developed evaluation scheme, and,
- 3. results of the application of the prediction methodology to three different aircrafts at three different sites (Series B: Field Investigations).

6.1 Equipment

It is necessary during the development of new equipment for measurement of pavement deflections to show that the new equipment provides measurements which are consistent with those obtained using established measurement systems. It is assumed that the LVDTs installed in the pavement comprise the absolute measurement system.

6.1.1 The LVDT System

The verification check of the LVDT system consisted of a comparison between pavement deflections measured by an LVDT mounted on the beam and an LVDT installed in the pavement as shown in Appendix B of this report.

Figure 27 shows a comparison of the pavement deflection measured

by an LVDT installed in the pavement and an LVDT mounted on the beam, and produced by the same pass of a P-2 fire truck during the testing sequence conducted at Eglin AFB, at Site 1 on November 17, 1975.

The results show that the installed LVDT and the LVDT mounted on the beam give basically the same measurements.

6.1.2 The LED System

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The verification checking of the LED system consisted of two phases: a laboratory and a field testing phase.

The laboratory phase was conducted by Science Applications
Incorporated, and is summarized in Appendix A of this report. The
principal field checkout consisted of a comparison between the output from an LED on the LED beam and the output from an LVDT mounted on the
LVDT beam.

Figures 28 and 29 show the pavement deflections measured by the LED system and the LVDT system, respectively, and produced by one pass of a P-2 fire truck (Figure 21), during the testing sequence at Eglin AFB, Site 1, on November 19, 1975. Figures 30 and 31 show the actual digital output from the LED system and the analog output from the LVDT system as recorded by the light beam recorder. Since the deflections shown were measured for two separate passes of the P-2 fire truck, a direct comparison of measured deflections can not be made.

As stated previously a direct comparison of the measurements made by the two beams (LED and LVDT) could not be made since it is

Due to persistent problems of compatibility between the LED modules and the magnetic tape recording unit, the LED output was recorded on a light beam recorder.

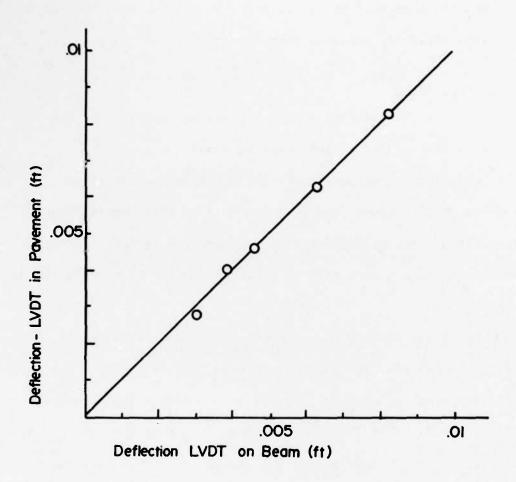


Figure 27. Comparison of LVDT on Beam and LVDT Installed in Pavement

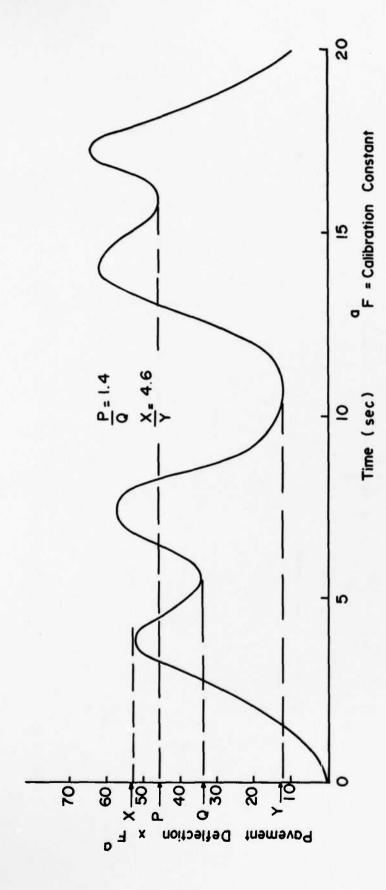
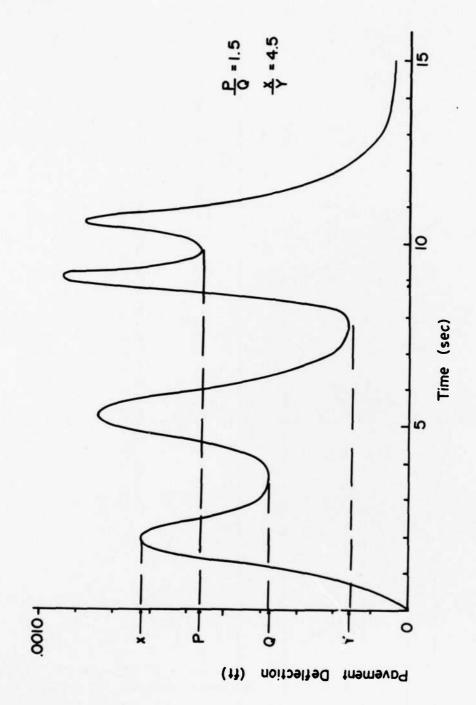


Figure 28. Pavement Deflections Measured by LED System: Eglin AFB - Site 1: P-2 Fire Truck Loading



Pavement Deflection of Gage Closest to Edge of Wheel Measured by LVDT System: Eglin AFB - Site 1: P-2 Fire Truck Loading Figure 29.

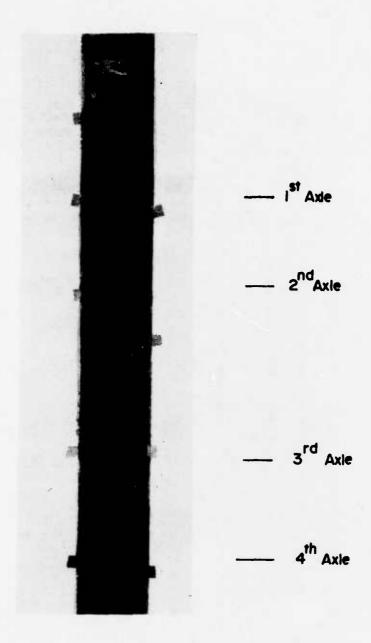


Figure 30. Actual Digital Output From LED System On Light Beam Recorder: P-2 Fire Truck Is Loading Vehicle

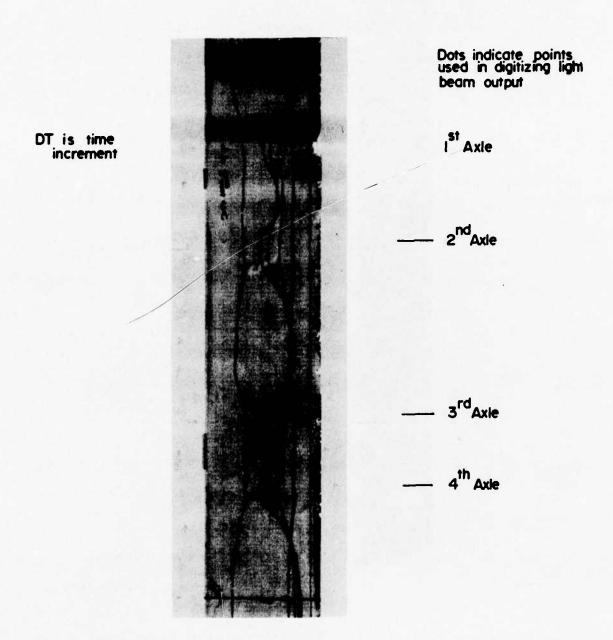


Figure 31. Analog Output From LVDT System: P-2 Fire Truck

impossible to get the two beams over the same point at the same time. Thus, a comparison was made of significant points on the measured deflection plots; e.g., peaks and troughs. The points P, Q, X and Y were selected, where P is the minimum deflection between the second two peaks of the deflection time plot; Q is the minimum deflection between the first two peaks; X is the first peak and Y is the minimum between the first two peaks and the last two peaks. The ratios P/Q and X/Y are thus dimensionless, and are independent of the system (dependent) calibration constants. That is,

 $\frac{X}{Y} = \frac{Actual \ Light \ Beam \ Recorder \ Output \ at \ point \ X \ times \ Constant}{Actual \ Light \ Beam \ Recorder \ Output \ at \ point \ Y \ times \ Constant}$

Pavement Deflection corresponding to X Pavement Deflection corresponding to Y

Figures 28 and 29 show that the corresponding ratios have essentially the same values. Both the P/Q and the X/Y ratios differ by only 0.1.

From the discernable deflection observable at point Y in Figure 28 and Figure 29, it can be seen that measurements of less than 0.0001 feet are obtainable by both the LED and LVDT systems.

In June 1976, both the LVDT and the LED systems were used in connection with a study to investigate the difference in the dynamic response of aircraft.

A mechanical system was designed to allow the LED beam to be rigidly mounted on the F-4 load cart. The system was fabricated at Tyndall Air Force Base by Air Force personnel and was then transported to Eglin AFB. At Eglin, 2000 feet of taxiway and 1 1/4 miles of runway

were tested using the LED system attached to the F-4 load cart as shown in Figure 32. The LED measured deflections due to the load wheel of the F-4 load cart as the load cart moved at creep speeds over the LED measurements were made continuously. The LVDT system pavement. was used to make deflection measurements at stations 100 feet apart as the load cart passed in front of the stationary beam. The continuous measurements made by the LFD were synchronized with the LVDT stations by the use of an electronic event marker of the LED system. synchronization allowed the time at which the load wheel passed the LVDT stations to be recorded on the magnetic tape to within 0.25 sec as follows. At the instant the load wheel passed the LVDT station marked on the taxiway, a button was depressed on the control panel of the LED system. This caused a digital record to be made on magnetic tape. Since the scan rate was four scans per second, an event marker could be recorded to within 0.25 seconds, which constituted one scan. Figure 33 shows a summary of the taxiway deflection data. The deflections on the dashed line in Figure 33 were measured at locations offset by about 20 feet from the previous measurement locations. Listing of typical measurements made by the LED system is given in Appendix D.

The results shown on Figures 28 and 29 indicate that the pavement deflection measurements with the LED system are consistent with those of the LVDT system, in form; and the LED can measure deflections with the desired accuracy in a field environment.

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The LED system data reduction software, a complete listing of all the taxiway data, and a copy of the data (on magnetic tape) has previously been supplied to the Civil Engineering Center, Tyndall AFB, Florida.

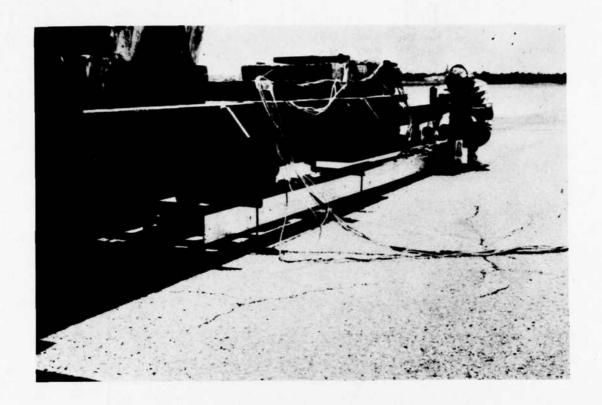
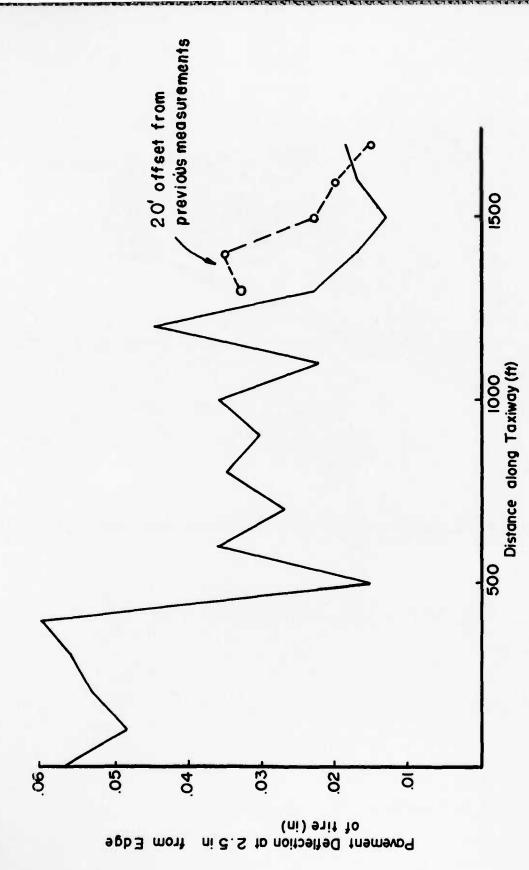


Figure 32. Setup During Moving Beam - Moving Load Operation: F-4 Load Cart



Summary of Taxiway Deflection Measurements-Eglin AFB 1976 Figure 33.

6.2 Evaluation

Evaluation, as presented here, consists of the following:

- 1. determining the parameters A_{peak} , β_{peak} , and r_{peak} of the maximum lateral deflection profile due to F-4 loadings,
- obtaining the k, c, and m parameters of the transfer function, and
- 3. relating the determined parameters to performance.

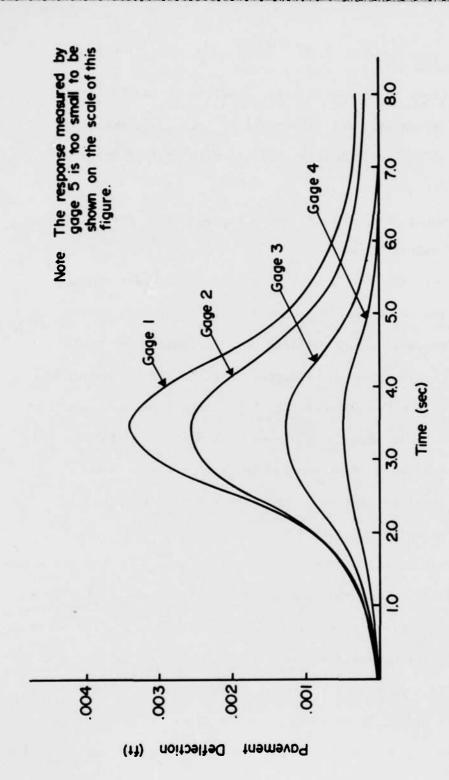
 The two sites in Series A field investigation are compared using the data obtained during the field testing program conducted at Eglin AFB in March 1976.

 As the F-4 aircraft is the vehicle of primary interest in this study only the results obtained using the F-4 load cart are presented in this section. A more complete listing of the data base and results is given in Appendix D.

Briefly, the evaluation procedure consisted of measuring pavement deflections using the LVDT beam as the F-4 load cart passed in a direction perpendicular to the stationary LVDT beam. From these deflections, the parameters $\Lambda_{\rm peak}$, $\beta_{\rm peak}$, and $r_{\rm peak}$ of the maximum lateral deflection profile, and the signature were determined as discussed in detail in paragraph 3.2.1. The k, c, and m parameters at each site were then obtained using the methodology of paragraph 3.2.2.

6.2.1 Pavement Deflections

Figure 34 shows a typical set of deflection functions due to the F-4 load cart as measured by the LVDT beam. Gage 1, which recorded



System Response Functions F-4 Loading: Eglin Site 1 (Typical) Figure 34.

the largest deflection is located closest to the wheel. Figure 35 shows the maximum deflection of each gage plotted against the distance of the gage from the edge of the tire print. The dashed line through the points G-1 through G-5 is the lateral profile of maximum deflections at Site 1 for one pass of the F-4 load cart. Figure 35 also shows the A_{peak} , β_{peak} , and r_{peak} parameters determined as discussed in paragraph 3.2.1 and the maximum deflections at each gage calculated by equation 2. Figure 36 shows a similar plot for one pass of the F-4 load cart at Site 2. Tables 3 and 4 give the stations 10 and values of the A_{peak} , β_{peak} , and r_{peak} value for Sites 1 and 2, respectively. Figures 37 and 38 show the variability in the measured maximum lateral deflection values for Sites 1 and 2, respectively. Figures 39 and 40 show a comparison of the local and site characteristics at the two sites as given by the A_{peak} , β_{peak} , and r_{peak} values. "Local" refers to a station or location at a "site". The lines in these figures are drawn for convenience and do not represent continuity of parameter values. Figure 41 shows the typical effects of two consecutive passes on the maximum lateral pavement deflection profiles.

These results show the following:

The gage (Gage 1, Figure 34) closest to the wheel records
the maximum deflection and retains the most permanent set
after the wheel load has passed. The gage (Gage 4, Figure
34) farthest from the wheel does the opposite; namely,

Sites are indicated in Figures B-1(a) and B-3(a).

¹⁰ Stations are given in Appendix B, Figures B-1(b) and B-3(b).

Lateral Distance From Edge of Tire Print (ft)

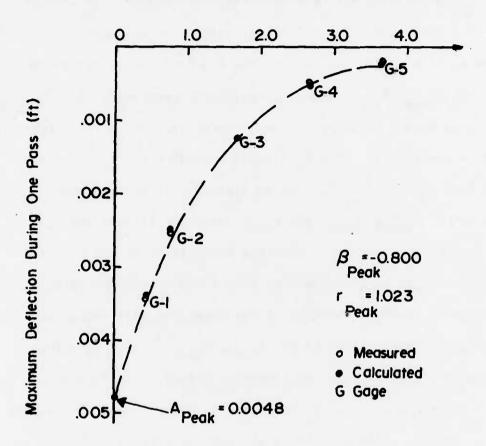


Figure 35. Eglin AFB: Site 1 Lateral Profile of Maximum Deflections for F-4 Loading (Typical)

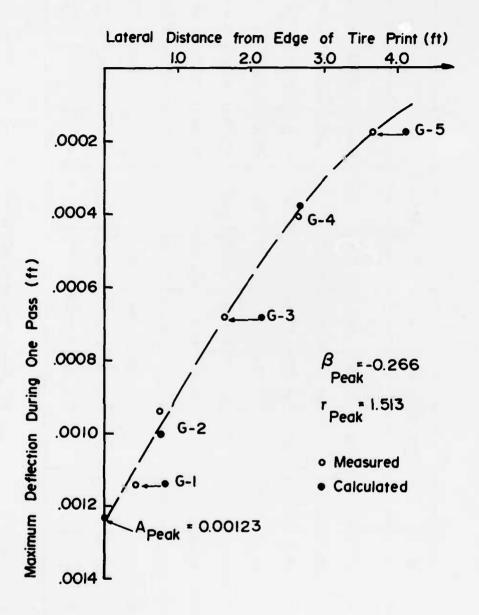


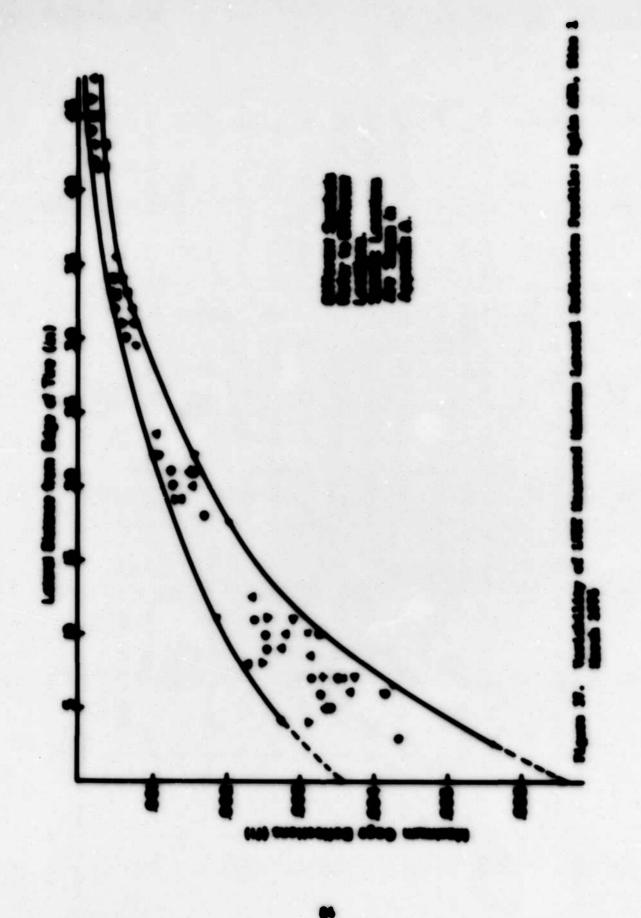
Figure 36. Eglin AFB: Site 2 Lateral Profile of Maximum Deflection for F-4 Loading (Typical)

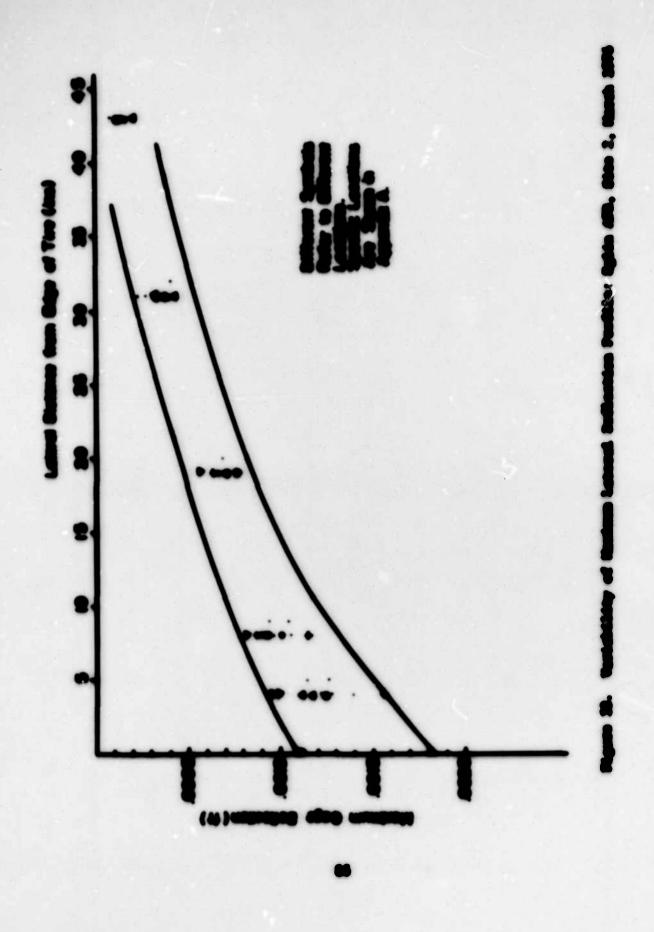
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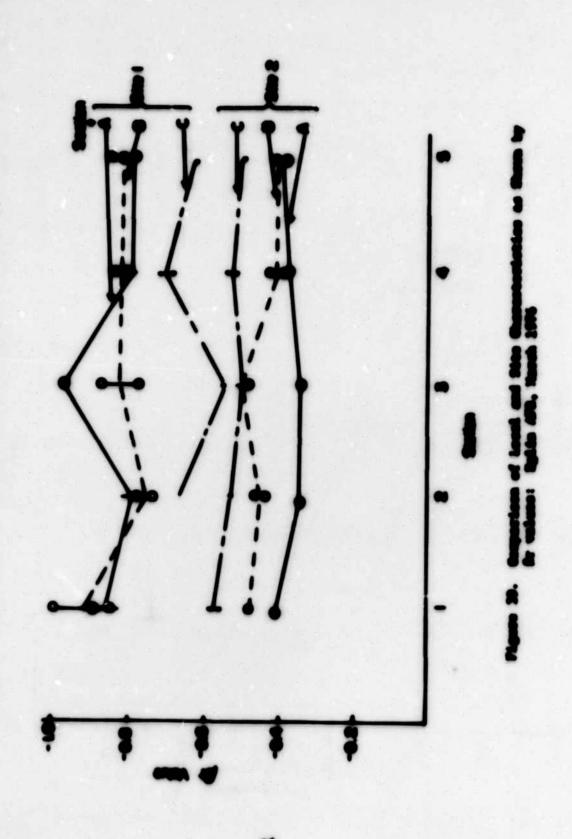
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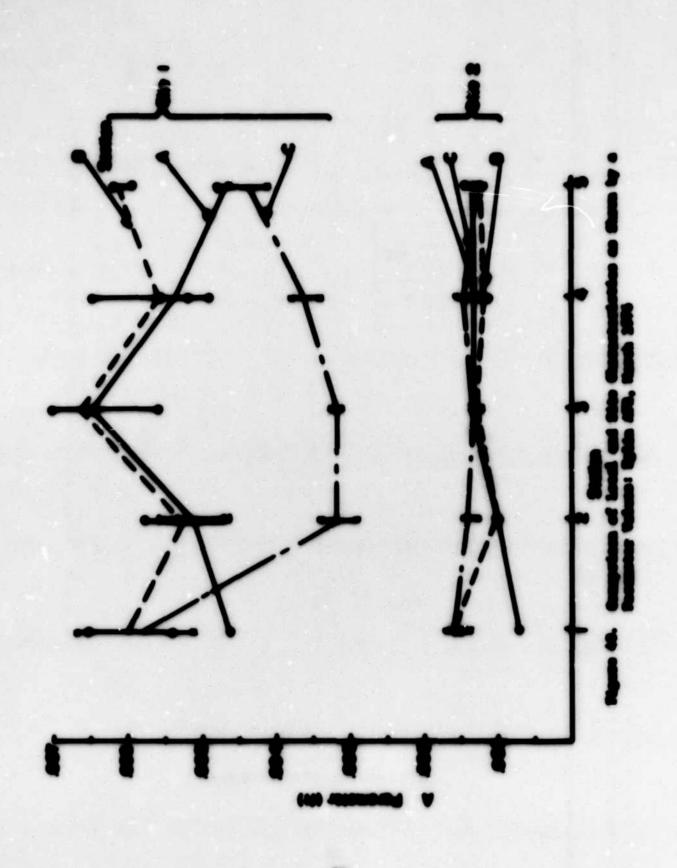
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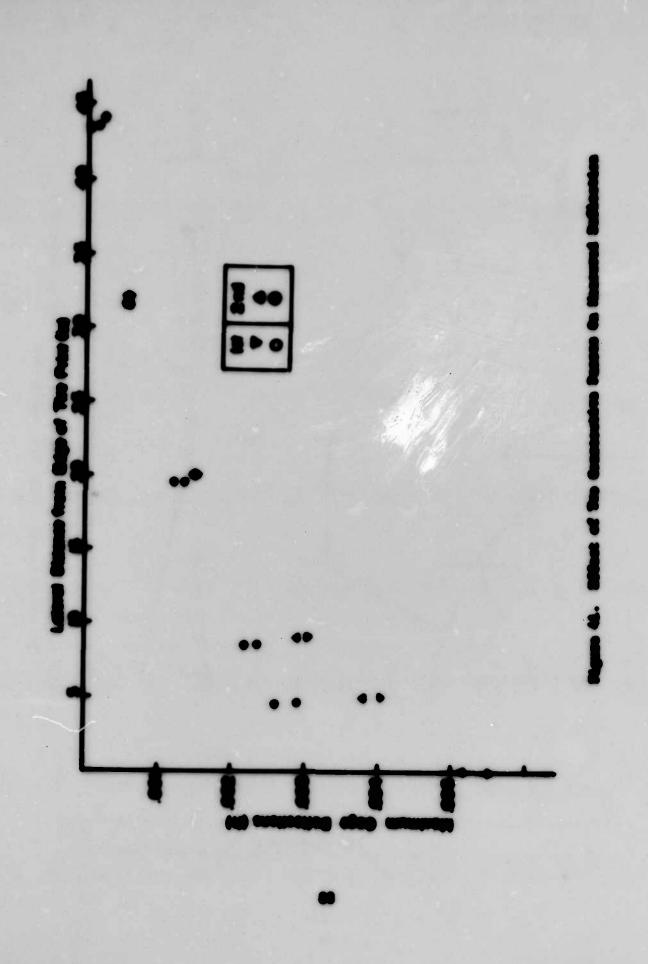
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the expect of personent set torresses so distance from the cheel decreases.

- 2. The pathodoses of deflection-time plate decreases as the distance from the wheel increases (Pigure 34).
- 3. The width of the measured meature letteral deflection bacts to deat 4 feet to 5 feet for both Sico 1 (Pigure 35) and Sico 2 (Pigure 35).
- 4. The assistan assessed deflection of fits 1 to shout three to four times the maximum deflection of fits 2.
- 5. There is a close fit between the calculated and measured maximum lateral deflection profiles.
- There is considerable variability in the asserted maximum lateral deflection profile at both Site 1 and Site 2, ever the areas tested (Figures 37 and 36).
- The absolute values of r_{peak} and f_{peak} are consistently larger for Site 2 than Site 1, and the A_{peak} values are smaller for Site 2 than Site 1.
- 6. There term repetition of lead decreases the measured deflection i.e. the second pass of the P-4 lead cart produced less deflections than the first pass (Figure 41).

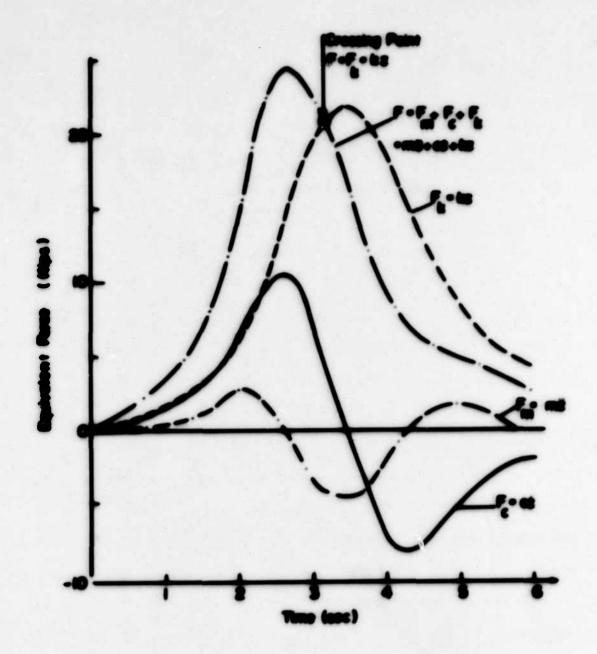
6.2.2 Incodes President

Transfer functions were calculated using the theory and methodology presented in paragraph 3.2.2. Briefly, the precedure involved finding the transfer function which, when enerolated implicitly with the eigenture yielded an empirelest input, which astisfied certain

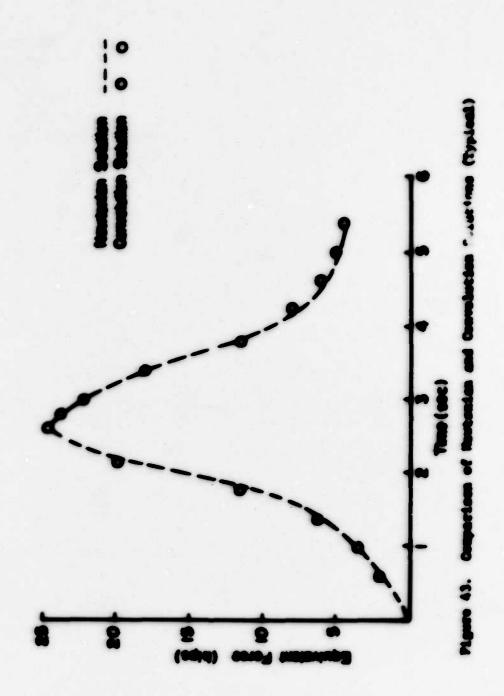
epocified conditions.

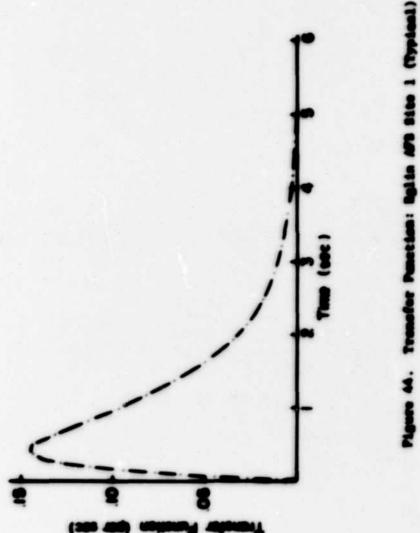
Plants 42 shows the calculated controlast force and the community of the force on the left hand olds of equation 1. Plante 41 shows a experienced the contrology force extended by implicit convolution (equation 15) and by direct substitution to equation 1. Pigures 44 and 45 they typical transfer functions for fittes 1 and 2, respectively. Tables 5 and 6 list the parameters of the transfer function for the two ettes tested. Ptetten labels (Al. A2 etc) correspond to etetten locations show to Appendix B. The L. c. and a appendicre were obtained from deflections assessed at each location. There two values ere given at the came etation, the first listed value corresponds to a forward page and the escent, to a backuped page of the P-4 land cort. Additional entries designate the repotition of the cycle. Pigures 44 and 47 there a comparison of the local and eith characteristics as given by the emissions existence (b) and demine (c) persectors. Plaures 48 (TDT to time dependent transfer) and 49 class a comparison of the aggettude of the first pasts and the time to the first past of the tounsfor functions having the parameters listed in Tables 5 and 6, respectively. Pigure 30 shows the tapet force, P(t), and the eignsture s(t) for one pass of the F-4 load cort. These results show the following:

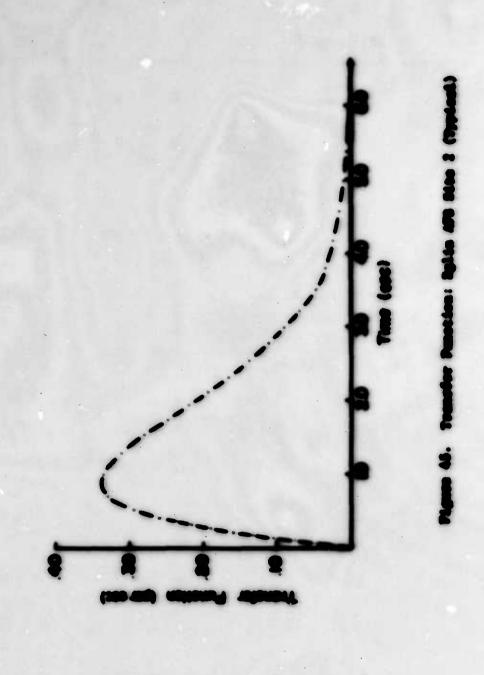
 the sumptions procedure used to find the transfer function to constituent with the developed theory, both in the determination of the force assumption (Pigure 42), and implicit convolution techniques to obtain the environment force (Pigure 43).

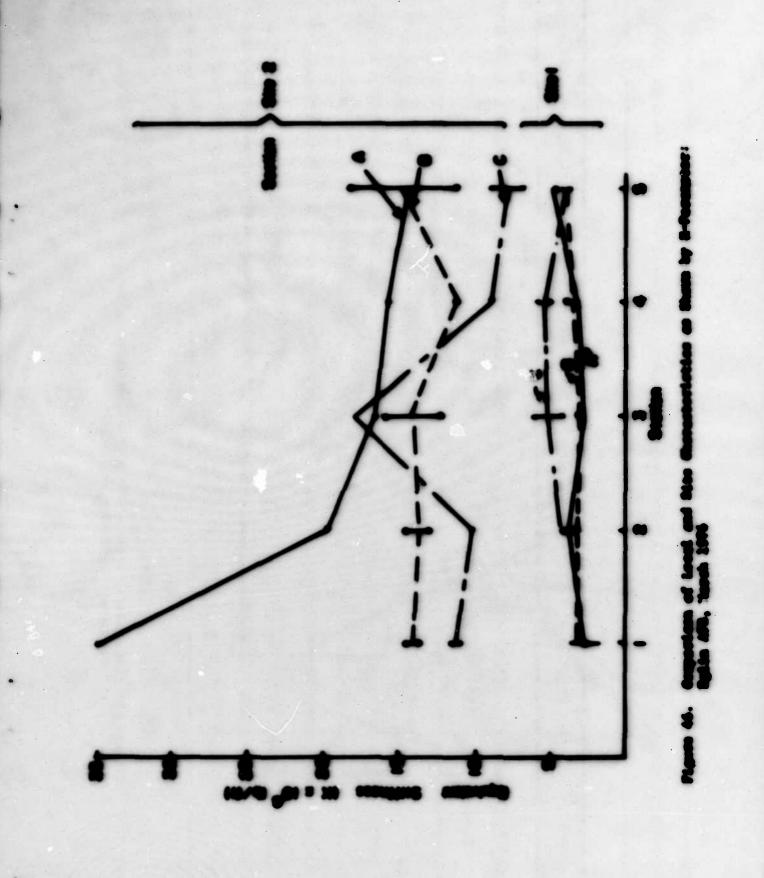


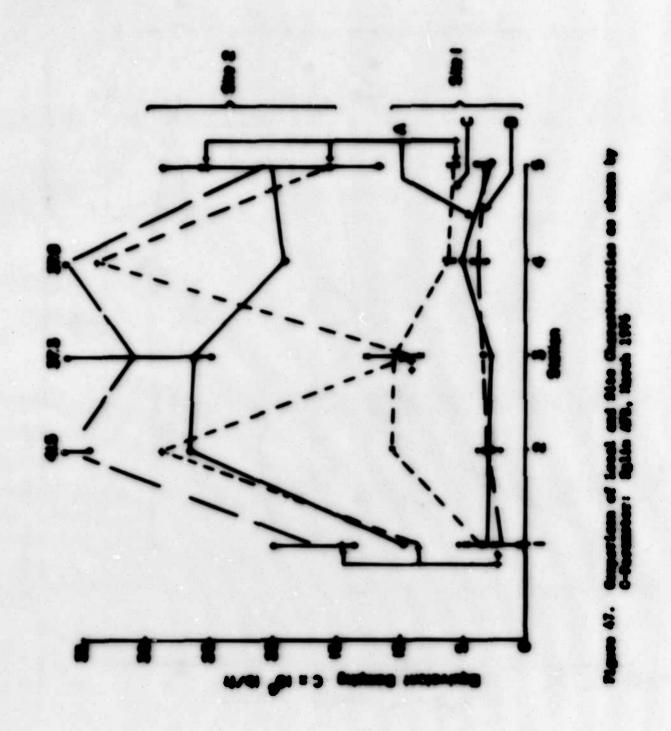
Pigure 42. Coloulated Squivalent Perso and Components (Typical)

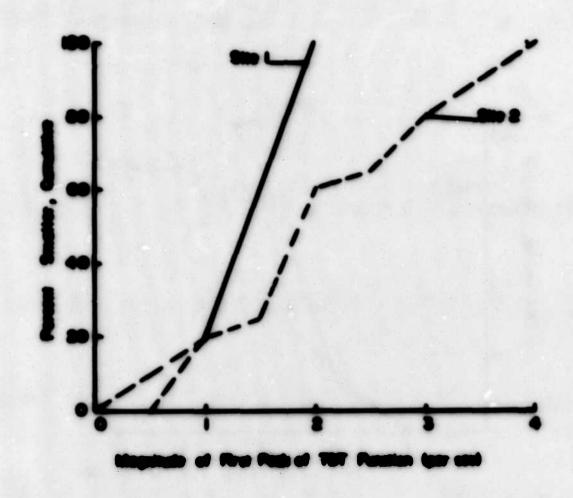




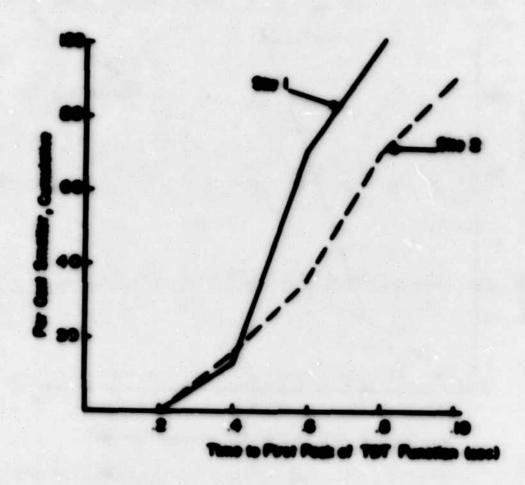








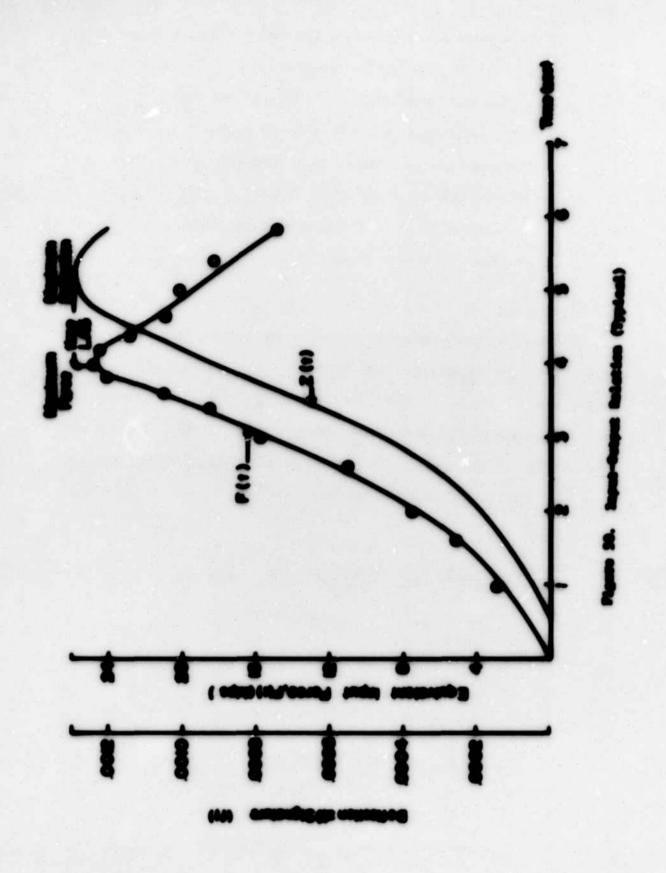
Pigents 44. Appendixon of The Pende of The Transfer Panettes: byter
AFF Speed 1976



Region 40. Companions of These to River Fact of

**	1.3			•	B.70	3. S.	1.999	22.70	32.TO	1.387
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	4	1		1			3.78			3.12
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	9.83	3		.9		3	M.70	•	1	1
***	ž	3	3.15	3,	3.98	2.78	3.6	•	3	•
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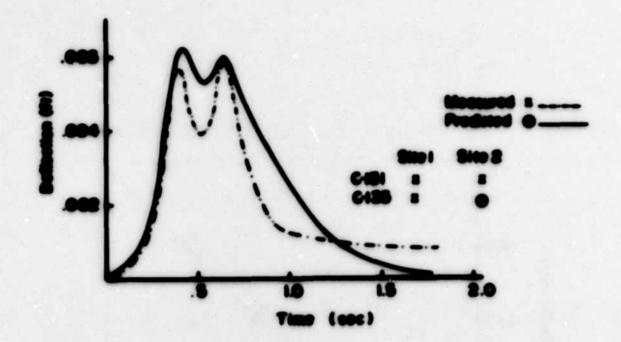
	3	8.8	•	27.8	_	•	•	13.9	15.8	8.8
11	:	1	•	4	-	•	•	1	1	4
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	20.00	2.9	n.s	3	8.8	8.8	•	3	27.52	8
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	9.0	•	8.8	8.8	•	8.8	8.8	2.4		
	87 8	•	8.3	X.8	•	8.8	3	79		
	1			2	1		8			

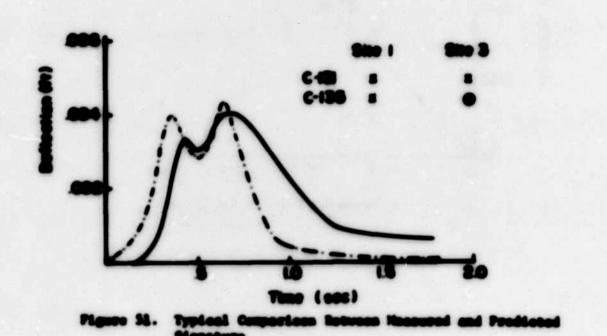


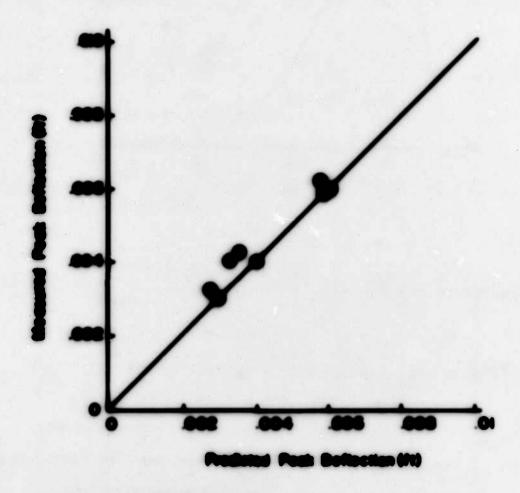
- 2. Both cites are ever critically desped (Figures 44 and 45).
- 3. Site 2 has essetstantly larger k, c, and n parameters than Site 1 (Tables 5 and 6 and Figures 46 and 47).
- 4. Miner differences ecour between the peaks of the transfer functions and the time to first peak for the two sites tested (Figures 46 and 48).
- 5. The maximum deflection lags behind the maximum input force by shout 1.0 second (Figure 50).

6.3 Proffetten

The methodology developed proviously use used to predict the pask deflection response of the C-130 and the C-135 at filter 2 and 3 (paragraph 5.2, Series 8 Field Investigation). The C-131 and filtr 1 uses the standard validate and elte, respectively. Pigure 51 chara a typical plot of the predicted and measured algorithm. Pigure 52 chara a commany plot of the measured and predicted peak dynamic deflections.







Plause SS. Comparison of Hassard and Predicted Peak Reflections for the Sites

SECTION VII

DISCUSSION

The ecope of this work includes equipment development, personnel evaluation, and prediction of personnel response. The significance of the results and the reliability and practical applicability of the equipment are discussed in this section.

7.1 Sentences

Several optical systems were considered during the initial stages of the development of this study. The first system used a least so a coherent light source. The initial concept was that the laser would be positioned off the parament and shoot at either the moving lead vehicles or reflectors on the parament surface. Changes in the position of the reflected light bean implaying on a final surface would indicate deflections.

It was next thought that, rather than having a locar fixed at a particular location off the paveoust, a contilever been with its pivot extelde the deflection basis would yield a more versatile and mabile spoten. The been would support locars which would shoot down on the paveoust surface. Changes in the light been reflected from the surface and impinging an collector loss on the been would indicate paveoust deflections. Several locars would have to be purchased and asserted as a contilever been. The expense and buildiness of the locars prompted a search for a more compact, inexpensive coherent light source. Light entiting diades on a contilever been were finally selected as the breadboard measurant system for measuring paveoust deflections.

The deflection appearing equipment (LD) hear) developed in this

the order of 0.001 inch in a field covironment. The companion LIDT bases has been ever rugged, reasonably simple to use, and very reliable. The breakboard LID system is commute more delicate. Special care had to be taken when in the moving male, to consure the enfety of the lenses of the LIDs. Considerable problems also erose with the compatibility of the LID module and the magnetic tape recording system. These may be a result of the fact that the magnetic tape recording system. These may be a result of the fact that the magnetic tape recording system was designed for use with another system, and thus the finer vertections in the electronics which may not have affected the operation of the original design, may have been amplified when the system was used in compaction with the LID modules. Although afforts were made to reduce those problems at Purdue University other again undiscovered incompatibilities between the LID conser modules and the magnetic tape recording system may still exist.

Figure 53 chare a custory of measurement systems available today. It can be seen that actual vehicular loads are applied only in the procedures using the Beshleson boom, and the California and LaGroix deflectameter. It should be assed however that for those only the rebound deflections are measured. The total time response for one complete pass of a vehicle is not measured by any current device assopting those reported in this study. Under normal operating conditions a vehicle passes over a povement vithout passe. This situation is very different from the conditions modelled by the beshleson beam or the California or LaGroix deflectameters. For

¹¹ Goveleped under contract with the Paderal Highway Administration.

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these the testing precedese starts at a senious deflection under static leading. The validate then moves any and the residual deflection is recorded. The complete lead spale is not recorded by this stapping and starting precedese over though an estual validate is used.

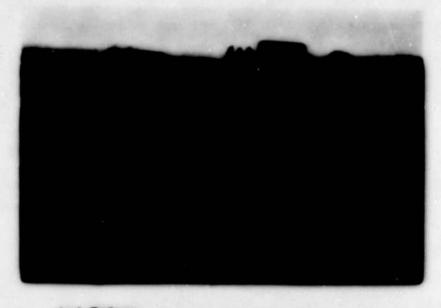
Vibratory techniques also main! a situation which is not representative of prototype vehicular leading on a percent-subgrade system. The precedure for conducting vibratory toots generally consists of prelaphing a personant-subgrade system by applying a static prolond to the personent surface through a plate of a specified slap. At a certain preload level the steady state vibratory load is than applied at a designated frequency. Pavement deflection or a cortain espect thoroof is then engewood. These are significant differences between this pressure and the prototype leading of a pavernet-subscade system. Pirot, a vehicle does not apply a static pooling to the pavementsubgrade system. As proviously mentioned, vehicular leading under normal operating conditions to applied to the system without pause. Second, a stoody state vibratory force applied vertically dissipates expressionately 67 persons of the total emergy on Releich warnes, 26 percent as chapt verse and 7 percent as compression values (References 50). The bulk (67 percent) of the energy to therefore disalgeted by generating surface disturbance at distances in escape of 20 feet from the vibestory lead as evidenced by ground notion at these distances. Prototype leading, however, generally dissingtes its energy at distances of loss than 5 feet. For the F-4 elegraft the surface effect of the prototype load only extended between 4 to 5 feet from the wheel.

Impact and quasi-static leading liberies do not madel protetype

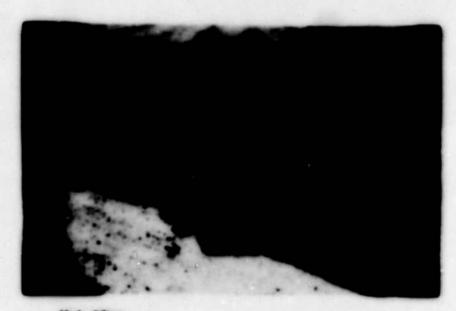
whichler leading. The time of impact leading to extremely small compound with the time a paramete continue extensity feels whichler leads. In the other hand for quest-static leading, the time of leading is very large compound with that of prototype whichler leading. In addition, for impact leading, a major portion of the input energy is used to paramete curface disturbances similar to stoody state vibratory leading. Prototype leading, however, processes little lateral curface disturbance out offerts of prototype curface leading are falt at depths of greater than 10 feet (heference 40). Insider important aspect of prototype leadings will be illustrated by an actual energie.

North the North field investigation a request was made by the Air Perce officials at Eglia AFE that a runney everum area to evaluated to determine whether it was adequate for use by P-4 aircroft. Figure 34(a) shows the area before portetype leading. Figure 34(b) shows the area before portetype leading. Figure 34(b) shows the section after the first passage of the P-4 lead east. The test vehicle pyshod through the paramet. It is dealtful that any antilable stoody state vibratory leading testing apparatus (that discipates a large masses of its energy as faleigh wome) would have descentioned the discovered inadequary. The significance of this experience may be commerciant as follows: An evaluation estates that was protectype leading in conjunction with condestructive mineuromate of paramet-endgrate response provides, est only encourants of its magence, but more importantly a test of mesonality.

The speed of operation or the number of tests per day shown in Figure 53 indicate results very from very slaw to 5000 per day. The literature (Reference 23), however, above the apper limit to be everly



(a) Culur



(0) After

Piguro SA. Russey Overrus Test - Baldo AFG: Harsh 1976

epitaletic. In the other hand, the operational epond of the boughteest USSF and LED epotens are indisputable.

In June 1974, both the LHRT and the LHD species were used at lights. Mr Perso base to help terrestigate the difference between located and substants profiles. The street even made on 2,000 feet of tentury using both species. The LHRT stationary bear-working local specialism was used to excesse the species sequence at 100-feet intervals. The LHR bear use fitted to the P-4 local cost and use used to conjunction with the LHRT excessants to provide as actionic of deflections between the 100-feet LHRT stations. The excesse time for easily, including equality surving the LHRT bear between locations and profittening the local cost, use 7 to 14 minutes. In addition to the 2000 feet of tentury, the LHR oppins second to the local cost was used to test 1 1/4 miles of recory. The survey matring each local cost was used to test 1 1/4 miles of recory. The

It is important to note also that in the electionary hom-worksy lead made the LD and LHRT beams both provide the complete defination-time response of a personant-entegrade due to prototype leading, that is, the entire dynamic defination basis is abtained. To enter instrumentation system correctly evallable has demanstrated this equilibility. Furthermore, the LD system does it without mixing maximated content with the parameter corrier.

7.2 Industra

The evaluation precedure consists of enseating parametric deflorations due to prototype leadings. From those deflocations the eigenture and purementers of the maximum letteral dynamic deflocation profile and the transfer function are obtained using the computer program detailed in

Appendix C. The persenters of the system lateral dynamic deflection profile and the transfer function are than related to performance.

7.2.1 Granate Head to Granater Process

The developed computer program determines the m, c, and k personstern of the equation that describes the behavior of the hypothesical mobil of valido-personnt-subgrade interestion. The equation of the mobil,

describes at least an approximation to physical reality. The larged parameters represent energy markets of the astual inestial (a), daying (a), and existence (b) characteristics of the real space. P(t) represents an input force at time t_{p} and s(t), $\delta(t)$, and $\tilde{s}(t)$ represent the eigenstates and its first and exceed derivatives.

Towards (Informed 61) has shown that when a weblete to in writing F(t) may have instantaneous values varying from 27 persons greater to to persons loss than the static wheal load. The wide range is caused by may fectors including both persons and vability characteristics. The maximum F(t) of 25 hips used to establish the 300-pound eritories (see pages—though—) in the computer program is thus chaply a commutest approximation to estable numbers values which may range from 12.5 hips to 37.5 hips. The 300-pound eritories thus correspond to a secondaring function and in effect has little physical eigenflowers, course in the correspond to the chaptilesses.

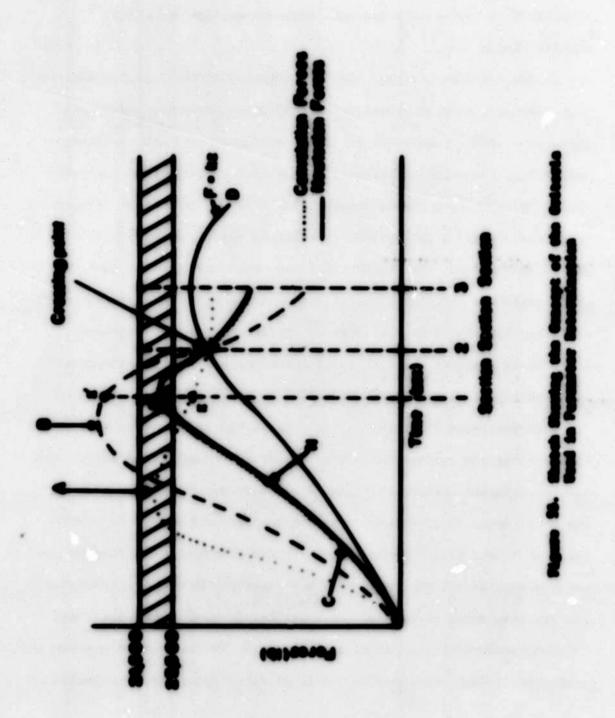
The larged parameters of the tomosfer function represent markets of the inertial (n), stiffness (h), and damping (c) characteristics of

the percent-enhance eyeten. Local ¹² characteristics may very eightfloomtly; however, when the everall interestion medianism to translated to curfose deflections, local variations less thair eightfloomes.

In the computer program, FREFRE parameters of the tempolor function upon ditained using an iteration-interpolation precedure detailed in paragraph 3.2.2. Conditions at times corresponding to the maximum deflection, the consolog point, and the point of inflection had to be extincted. Two criteria wave used. The first criterian was that the input load obtained by implicit convolution had to be between 34,360 and 25,360 pounds. The second and final criterian was that the c value extinities at the point of inflection. All forces were obtained by implicit convolution of the expectation. All forces were obtained by implicit convolution of the eigensture o(t) and the tempolor function using a corries of total values of b, c, and n.

With reference to Figure 35, in escent the first criteries expense that the exchange force lies within the creechstoked area. The count criteries excurse that the convolution and Newtonian colutions are very close. For example, consider the curves labelled A, B, B, C, and B in Figure 35. Suppose that the curve labelled B is the true but exhaus (Martenian) colution. Curve B estimates enougher criteries as (1) the peak value is cutoffe the specified (shaded) hand width, and (2) the computed c-value would be too large. The follows of the latter criteries is demonstrated when the force F(t₁) (shteland by convolution)

¹² Local, as used here refers to diameters of the else of a typical laboratory specimen.



of 2' (Figure 35) to used testeral of 2 to extended the a parameter at the point of inflection (section (1), Figure 35), using equation (17a). Curve A extinities the 300-pound extention but done not extinity the e-parameter exiterion, class the value of the force 2 at the point of inflection to less than 2. Substitution of 2 for F(t) in equation (17a) useld yield a value of a which would be less than the decired value. Curve C would extinity both exiterio. For this curve the convolution and fluctuation existings are executively the same. The fact that the two colutions are the same execute that a standard point of convolution to used for all calculations. It does not man that the k, c, and a parameters are chesiste values class as proviously cited, the parameters are decired values, and are influenced by the approximation [F(t)] to reality inhorant in the hypothesised model.

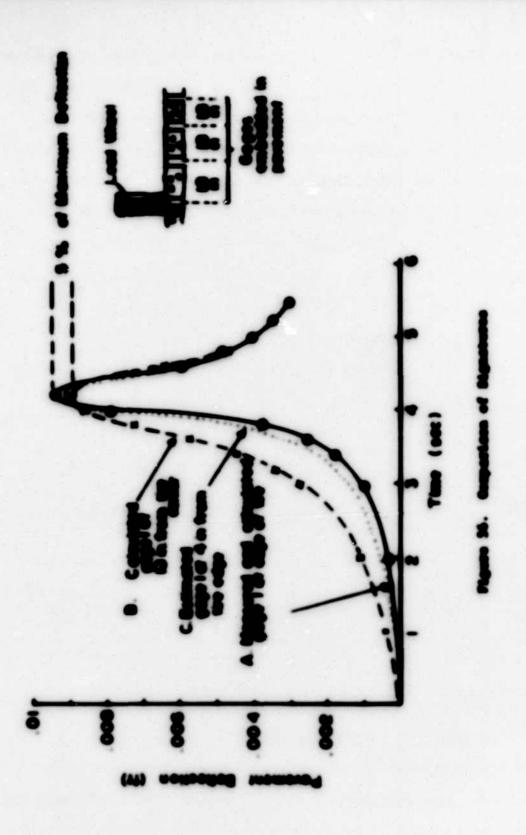
tion print, it was assessery to calculate the eigenture (the deflection at the edge of the time print) using the characteristics of the lateral dynamic deflection profile. It was assessed, and subsequently varified (Pigeres 35 and 35), that the deflections under the sheel and at 100 edge are assessfully agend. In calculating the eigenture the three parameters \$\langle_{\text{peak}}\$, \$\langle_{\text{peak}}\$, and \$\langle_{\text{peak}}\$ were used to provide a scaling of the lateral attenuation of the peak deflections with them. Pigere 35 chars a comparison of a typical measured and calculated eigenture. The eigenture (curve 4) was encoured as the \$P-4\$ possed over a gage exhaulted in the paramet. The other exhaulted gages measured points on the deflection havin which provided the inputs to the calculations. The elected gage was 16 instead from the center of the last wheel. The difference between the qualum calculated curve (8)

and answered deflocations to about 3 percent. Stace all field answerments analyzed in this work were undo with the LVRY beam at distances closer than 16 inches from the edge of the tire, the correspondence between the calculated and the answered eigentures in Pigure 36 indicates the upper bound of the differences between calculated and answered eigenture. Uning linear interpolation, the detted curve (C) in Pigure 36 corresponds were closely to typical calculated eigentures used in the data analyzed herein close the LVRY beam paralited measurements to be made as close as 3 inches from the edge of the tire print.

7.2.2 Parameters of the Nanisan Lateral Inflaction Profile

Personnel deflection has been used as an indicator of the support condition of a personnt-subgrade system for each time. Nothedo of system have been developed that are bound on (a) magnitude of deflection (Noference 62), (b) handing index (Noference 63), (c) redice of curvature (Noference 23), and (d) the "slape of deflection" (Noference 64). These methods in general have used static or quasi-static leadings. For the resonne discovered proviously in paragraph 7.1, those deflection measurements are not true indicators of the dynamic response of a payment-subgrade system subject to validation leads.

The dynamic lateral deflection profile defined by equation 2 has three parameters Apack, Apack, and Tpack which together account for both the against and the curvature of the lateral deflection basis. Since the deflection accordance were made during a maneter passage of the P-4 load cart in front of the LVDT beam, the three parameters are representative of the actual maximum dynamic lateral deflection basis due to prototype leads.



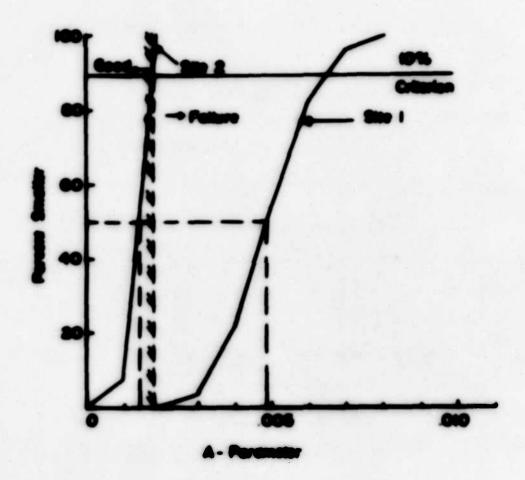
The receive presented in prongraph 6.2.1 chared come vertexion to the parameters A_{book} , B_{pook} , E_{pook} at each cite, and there was distinct differences between Sites 1 and 2. In order to better characterise the two sites and give due consideration to the parameter variability at the cites, plots of cumulative persent smaller versus each parameter were proposed from the results in pagagraph 6.2.1. Pigures 57 through 59 show those plots. The distinct differences between the two sites are displayed as a shift of the individual cite characterisation plots.

The U. S. Air Porce in recognition of the structural inadequacy of Site 1¹³ uses the area as a perhing area. Site 2 on the other hand is an in-service tensivey. If the lower 10 percent of the Site 2 persenters is arbitrarily selected as the line denoting failure, criterian based on the correlation of parameters with performance is established. This is about in Figures 57 through 59. The lower 10 percent as used here means that 10 percent of the values do not next the criterion. Thus, in Figure 59, the line denoting failure intersects the Site 2 cumulative percent amilier curve at the 10 percent smaller value whereas in Figures 57 and 58, the 90 percent smaller value of the cumulative percent smaller plot is intersected by the criterion line.

1.2.1 Parameters of the Transfer Punction

The results given in paragraph 4.2.2 should that there is some variation in the parameters k, c, and n of the transfer function ever

¹⁸ See Appendix 3, Figure 9-1(a)



Pigure 57. Comparison of A Parameters: Splin AFD March 1976

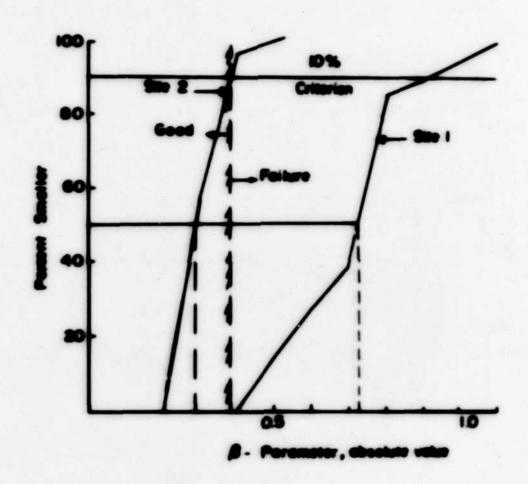


Figure 58. Comparison of the S-Parameters: Eglin AFS, March 1976

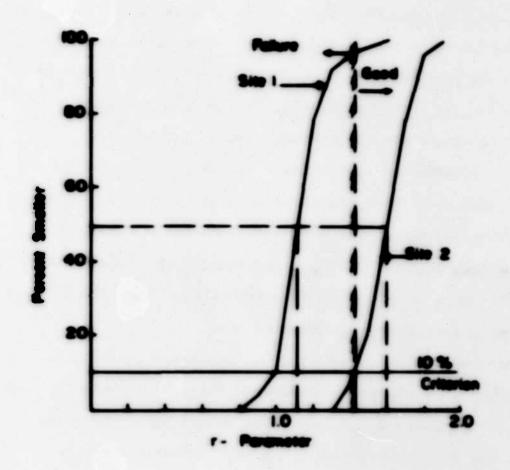


Figure 59. Comparison of r Parameters: Eglin AFB North 1976

can of the ten older. Plate of camilative percent canller versus can parameter have been proposed for each site. Planes 40 through 62 class these plate. It can be seen that 50s 1 cabilities were californity and cancidentally canller values of all three parameters than 50s 2. With reducemen to Planes 40 the fact that the entire curve of "manifetive percent canller" to chiffed tensors higher chiffboness for 50s 2 indicates that the stiffboness determined in the conjunction procedure are not local hard spate, but represent the variability in the stiffbone characteristics of the area testad-

If the arbitrary 16 persons extendion shown in Figures 57 through 59, to applied to each of the personstore k, c, and n of the transfer function, a extension based on performance can be had. The line denoting follows to shown as Figures 60 through 62. The performance based extension chann to Figures 60 through 62 can then be used to the case manner as was presented in the discussion of the parameters of the maximum lateral dynamic defication basin.

It is interesting to compare the k parameters with stiffness as determined in a 1975 evaluation (Indocumes 20) by the Air Porce, to which so P-4 algorith was used so the local vahiole and the bushelms has we used to measure released deflections. Pigers 63 does the leastions whose relative stiffnesses was determined. Pigers 64 does a bicongram on the relative stiffnesses relationed by their precision. Pigers 64 also show the case k parameters for Sites 1 and 2 as calculated by the precisions developed in this measure offert. Site 1 is cheen to have a smaller case stiffness than all the sections evaluated, while Sites 2 is in the range that occurs with about 9

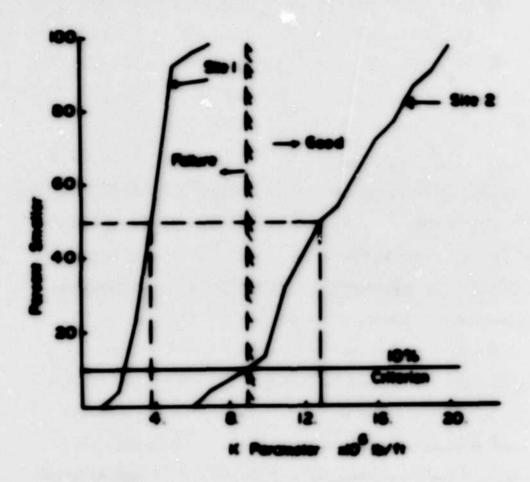
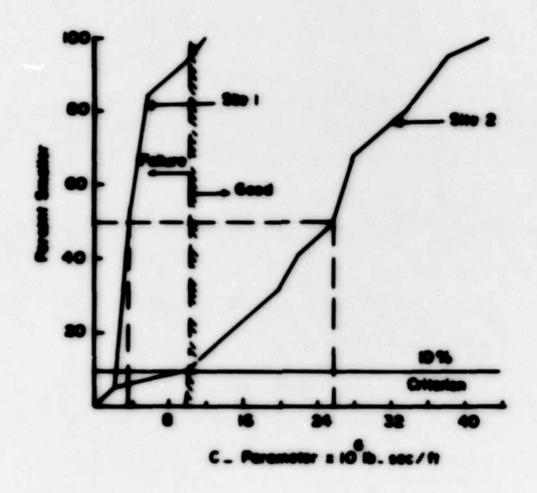
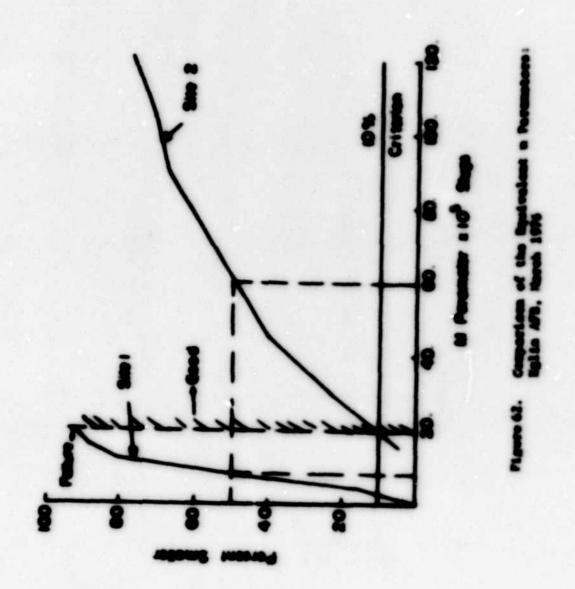
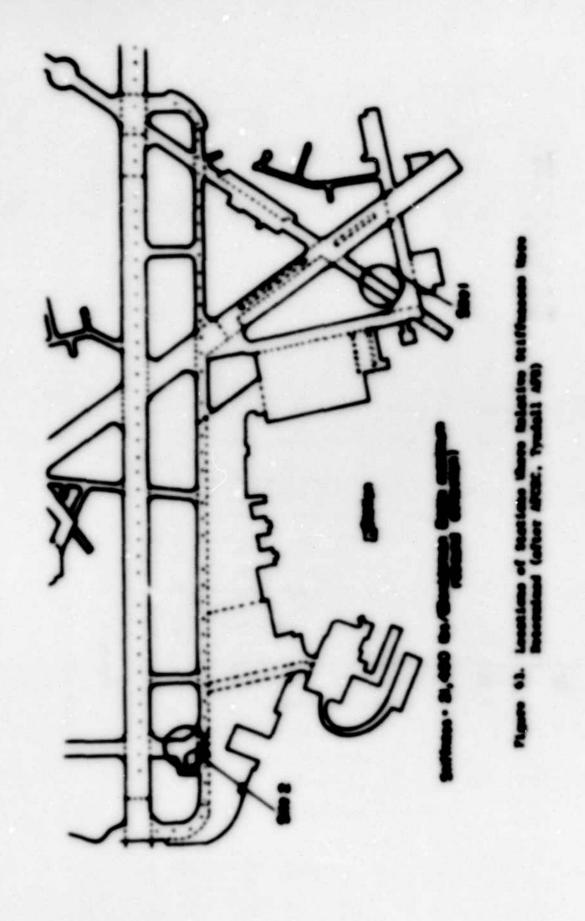


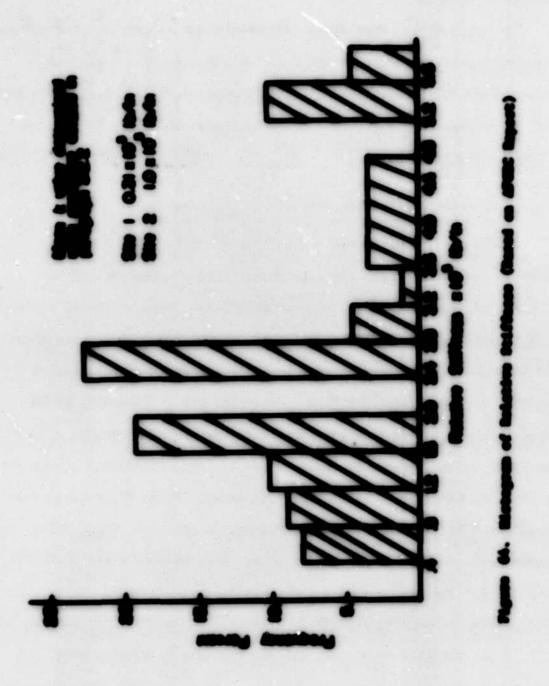
Figure 60. Comparison of Equivalent & Parameters: Eglis AFB



Piguro 61. Comparison of Equivalent c Parameters: Eglin AFS Nameh 1976





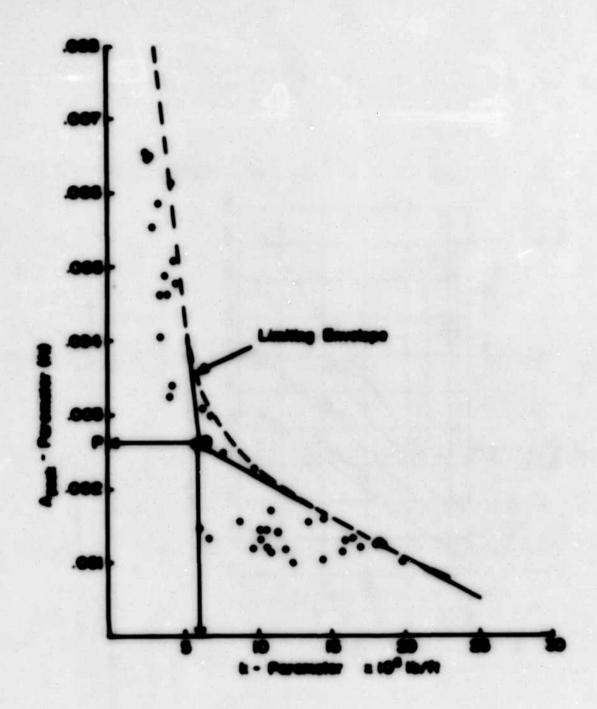


percent frequency.

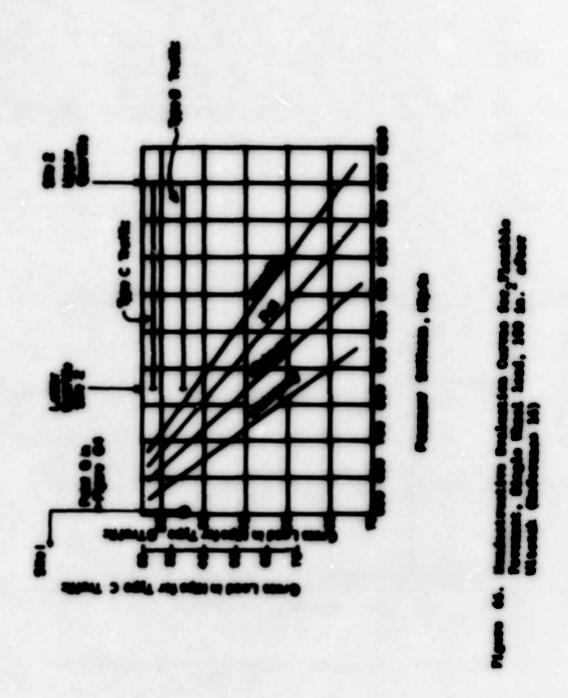
It appears than that field performance and comperison of stiffness both substantiate the lower limit of the k parameter. A reasonable lower bound of an acceptable k value of about 9.0 ± 10^6 lb/ft (750 kips/in) to suggested by the fits 2 curve in Figure 60. This is the lower 10 paramet value.

7.2.4 helationship between the b and A perameters

The A passesser defined in this study to besically the namium deflection under the lead wheel. The k parameter, on the other hand represents an equivalent stiffness. This stiffness reflects the complicated unknown interestion of all the materials in the mass of influence of the wheel load. The k parameter is thus a manuer of the stiffness resulting from this interestion. Pigure 45 shows a plot of the Anna and corresponding k personeters from the two eltes so given in the results in Tables 5 and 6. The deshed line represents a limiting envelope. There appears to be two linear sections to this envelope. If the straight line portions are entended, the A and k parameters corresponding to their point of intersection are 0.0006 ft (0.0312 ta) and 6 = 10° 1b/ft (500 hips/ta) respectively. It to intersecting to note that the Site 2 (active taxiver) (k. A) parameters fall below the line Pr (0.0312 in) and the Site 1 (perking ages) (k, A) parameters lie above the PO line. It is also interesting to note where the & coordinate of this point (0) lies, if plotted on Figure 66. The curves in Figure 66 were developed by the Corps of Ingineers using vibratory techniques and correlations with actual



Pigure 45. Relationship Netwoon A and & Peremeters



element lands. Since the P-4 element year to a single wheal of contest once 100 in. 2 located to about 25 hips (Reference 24), the h value of the point Q (Pigure 65) can be plotted directly on Pigure 66 for C type or 3 type traffic. Pigure 66 also shows h-parameters corresponding to the lawer and apper quantile values at 51to 2 (active tenings). Site 2 like above the capability likes and Site 1 likes below the analysis of Sites 1 likes below the analysis of Sites 1 and 2.

The limiting excelepe to Figure 65, therefore, appears to provide an additional evaluation tool. Since deflections alone are assessed, the limiting it values can be estimated from Figure 65, and compared with available correlations as shown, to estimate load rating.

7.2.5 Apparationte Debad for Sentention h. c. and a for P-5 Leading

Figure 34 shared the typical relationship between the layer force F(t) and the extent s(t). Figure 47 share the frequency characteristics of F(t) extenditual by the maximum excessed charl load (25 bigs) at those contians corresponding to contians 1, 2 and 3 of Figure 7. Staty data note was used to devolupe the frequency curves.

With redormers to Pigers 67, at the print of infloration (mention 1), the committeed force ratio has a man value of approximately 6.5.

This engages that the maximum applied force excurs at a time close to maximum personnel valuesty. Statlardy at maximum deflection (mention 3),

9.5 of the maximum force is the man value of mentional force ratio.

This indicates that at the time the maximum deflection is a maximum the wheel has already person the point under consideration. At the

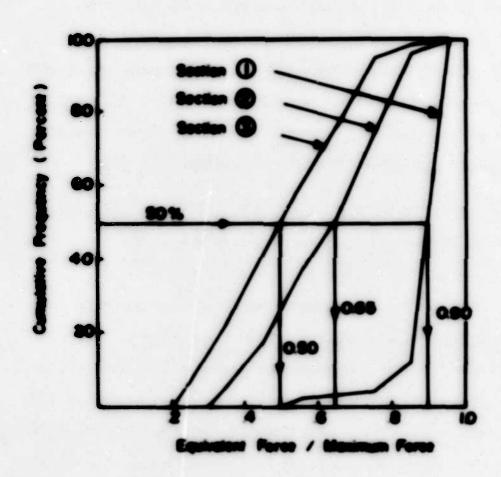


Figure 67. Progumey Characteristics of Porces at the Point of Infloction, Crossing Point and Naziman Deflection

especial point (meetion 2), approximately 0.65 of the maximum lead to the mass value of the force ratio.

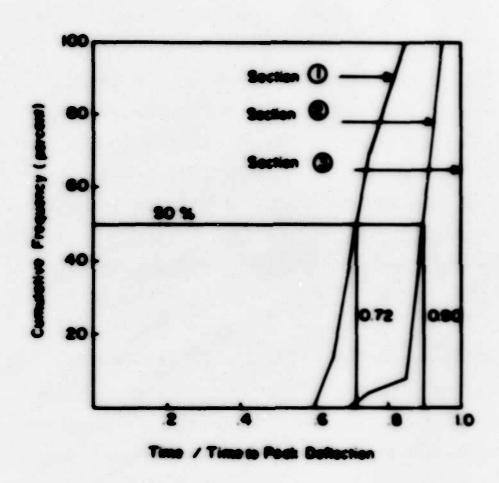
Figure 68 shows the frequency characteristics of the time mornalisad by the time to peak deflection corresponding to the point of inflection (coeties 1) and the crossing point (coeties 2) respectively. At the 50 percent level the normalised time ratio to seen to be 0.72 and 6.90 for the point of inflection (coeties 1) and the crossing point (coeties 2) respectively.

Figures 67 and 66 suggest a simplified approximate method for determining the transfer function parameters from the deflection-time biotecty of one pass of a vahicle. In order to determine a from the signature s(t), the deflection $s(t_2)^{14}$ is found at 0.9 T_p , where T_p is the time to the pass deflection (Figure 66). The force $F(t_2)$ applied at the time (t_2) is approximately 0.65 F_{max} (Figure 67) where F_{max} is 25 tipe for the 7-4 leading. As was moved in paragraph 3.2.2 at the exceeding point.

honor, using the appreniants values,

$$k = \frac{P(k_2)}{a(k_2)} = \frac{(0.65)(25000)}{a_{0.9} T_p}$$

¹⁴For the sake of clarity, the subscript of "t", such as in t,, will be retained to describe the corresponding section in Figures 67 and 68.



Pigure 68. Progusney Characteristics of Times at the Point of Inflection, Crossing Point and Maximum Deflection

Case k is known, the c parameter can be obtained by applying the case reasoning to the point of infloction (section 1, Figure 67); namely,

where $P(t_1)$ is 0.9 P_{max} (Pigure 67) and $z(t_1)$ and $z(t_1)$ are the deflection and numerical approximation of the velocity, respectively, at the time corresponding to 0.72 T_{n} (Pigure 68).

The peremeters a can then be estimated using the equation for conditions at the crossing point, where

and $\psi(t_2)$ and $\tilde{v}(t_2)$ are the numerical values for the velocity and escaleration at the time 0.9 T_a.

The developed apprenicate precedure for estimating k, c and a war emphasis with respect to 35 data sets from both Sites 1 and 2 at Eglis 475¹⁵. The normalized forces at the exceeding point and the point of inflection were taken to be 0.65 and 0.85¹⁶, respectively. The normalized times at the exceeding point and the point of inflection were taken to be 0.72 and 0.9, respectively. The k, c and a parameters obtained using the simplified procedure were then compared with their computer solutions. Correlation coefficients of 0.93, 0.93 and 0.90

¹⁵ See Appendix 8

^{160.9} was tried initially but 0.85 yielded c values which correlated more closely with the computer solutions.

the chained from the linear least equates correlation study for the h, a and a parameters, respectively. The advantage of the admitted presenters to that it allows reasonably reliable estimates to be made of parameters without the most of a digital computer.

7.2.6 Sectionaling Production

The undelse of subgrade reaction (\vec{k}) and CR of the subgrade can be actimated from the A_{nech} parameter of the peak deflection profile.

Votes the exemption of a Malvin model, Plater and William (Reference 26) developed the following relation between the subgrade deflection V(x), pressure on the subgrade (p), undeless of subgrade reaction (k) and a shear stiffness parameter (CE) on the distance from the edge of the located stee (n):

This expression is similar in form to equation (2),

which was found to describe the maximum lateral dynamic deflection profile. If it is assumed that the two relations correspond at their maximum values (for x=0), it follows that

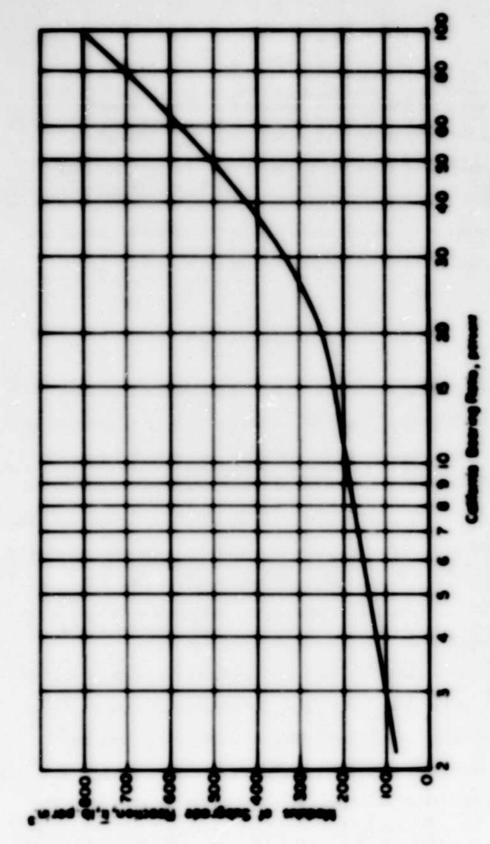
Honce, this suggests that the modulus of subgrade reaction can be obtained as.

Using a 1:1 simplifying assumption (cf. Today (Reference 55)) to calculate the extraordies of the wheel lead, $T_{\rm cons}$, through the parameter continue of thickness b, the context presents, p, equals $T_{\rm cons}/(5.44+9)^2\tau$; where 5.44 is the matter of the equivalent elements contact once due to the P-4 alrestoft with time presents 250 pet and a $T_{\rm cons}$ of 25,000 pounds. Substituting in equation (23a) gives

CDR values can then be estimated using established correlations
(Reference 65) between \$\hat{k}\$ and CDR. One such correlation given by the
Portland Commit Association (Reference 65) is shown in Figure 69.

Using equation (23b) and the maximum, modion and minimum values of A_{post} calculated at site 2, and a D of 17 inches, E values of 323, 462 and 646 pounds per cubic inch were obtained. The same calculation for site 1 (with D = 9 in) gave E values of 257, 316 and 515. The correlation chart shows in Figure 60 gave the range of CSR value for site 2 from 26 to 73, and for site 1 from 18 to 52.

Modeli of subgrade reaction were not determined under asphaltic concrete powerent sections at Eglin Air Perce Base by the Air Perce Civil Engineering Conter during the 1974 Survey (Reference 21): subgrade CERs were obtained for those sections and those only at a few locations (about 10 in number). In the area of site 2, a CER



value of 75 was reported. As noted above, this corresponds quite closely to the maximum calculated value of 75. The range of calgrade CMb throughout the personnel continue varied from 10 to 85 persons. Imported 6 values, under the Portland Counts concrete continue, ranged from 200 to 520 pai.

It is interesting to note that the lower value of the range of CMs for site I was about 10 while the low value for all the results was 10. Even though so CM values were evaluable in the vicinity of site I, a comparison of its corresponding CM and E values with those at site 2 indicate it to be an inferior powerst. This had been recognized by the Air Perce as it is used only so a porking area; whereas, site 2 is an in-corvine tenistry.

In summary, the recommended preceders for estimating the \bar{k} and CMR values from prototype F-4 leadings to as follows:

- 1. Estimate A_{peak} from an arithmetic plot of distance (n_j) from edge of the tire print versus peak deflections at each gage (n_j) . A_{peak} is the intercept at x = 0.
- 2. Use equation (23b) to estimate £, the modulus of subgrade reaction, and, if desired,
- 3. Find CBR value from cotablished correlation charts such as Figure 69.

A verted example to show to Appendix E.

7.3 Prodiction

The prediction methodology consisted of (1) determining the relationship between the inpute of two vehicles at a standard elte,

and (2) using this relationship to product the response of one of the validation at another olds, if only the response of that olds to the other validate to ensured.

7.3.1 Validity of the Greent of the "Tentral over Prosting"

The concept of the equivalency function has been used in many sense of engineering. In foundation engineering it has many name, some of which are "shape factor", "dayth factor", and "inclination factor". In highway and airfield work it is known as "the equivalent single wheel load" or the "equivalent single cale load" or "layer equivalency factor". The basic unforlying except of "equivalency" is that two different entities can be related by a multiplier. In this work, the unitiplier is a function of time - the equivalency function - and not a number.

The question arises as to whether the equivalency function defined at one elte is applicable at another site where another set of conditions provedle. The results, Figure 32, should that the equivalency function is indeed applicable under sets of conditions different from those under which it was first determined.

The equivalency function so defined in equation 21 is essentially a linear ratio of inputs of two vahicles at the same site. In reality the relationship between inputs is probably not linear; however, three factors seen to combine to make the effects of smallmoority negligible.

First, consider the strain induced in the system due to prototype leading. Measured peak deflections are of the order of 0.05 inch,

and the state of t

thoras the depth of influence of the theal lead to of the order of 12 fact or 144 tackes (Reference 60). Small excelse, in general of thout 0.000), are thus induced in the system as a result of prototype leading. The linear approximation to valid because strains are small.

The linear approximation to extual explinear exterial behavior at small exterior has been used to exchange for years. Here (Reference 66) has shown that if the functional relationship (T_1) between extress (6) and extens $(c_2, c_y, c_y, \dots, \gamma_{nn})$, given by the equation

is expended in MacLauria series, the equation

$$a_n = (r_1)_n + \left[\frac{\delta r_1}{\delta r_2}\right]_n c_n + \left[\frac{\delta r_1}{\delta r_2}\right]_n c_n + \dots + \left[\frac{\delta r_1}{\delta r_{nov}}\right]_n v_{nov}$$

results, if the higher order towns in the Harlauria series are eliminated. The zero subscripts denote the value of the function F_1 relationship and its derivatives at zero strain. From this relation it can be seen that if strains are small a linear relationship exists between σ_g and c_g , c_g , c_g , etc. Applying this same resonaing to the equivalency functions, the linear approximation between equivalent inputs works, because of the strains induced in pavenest-subgrade spectrum due to prototype leads are, in general, small at all sites. The equivalency function at one site is thus the same equivalency function at mother site where strains are small and of the same order of appairable.

The second factor, which contributes to the validity of the

linear equivalency function, to inherent in the flexible powerest design procedures, that have evalved, and supports the "small strain" essent proviously cital.

Percents are designed, basically, to limit subgrade deflections. This can be inferred from the fast that conventionally, thick sections are constructed ever weak subgrades, while on the other hand, thisner sections are built ever etrong subgrades. The not effect to that deflections and thus etrains are forced to stay within the same order of magnitude of small etrains by the design precedures. Since percent sections are generally designed using conventional precedures, the "small etrains" concept proviously discussed should be valid for most percents. Thus, the equivalency function between two such sites would be valid if atrains at the two sites are of the same order of magnitude and small.

The third and final factor concerns the time the maximum lead to actually applied at a point on the paramete. The time period that the wheel lead is ever a point is very small. Authors (References 40 and 47) have used Dirac Delta (impulse) functions to represent valicular leads. Since the time is small, factors such as settlement, which normally are amplified in static cases, are of minimal importance in the dynamic mode.

7.3.2 Effects of Goot Configuration

The C-135, C-131 and C-130 electeft have different goor configurations as shown in Figures 24 through 26. The prediction methodology permitted a vehicle having a single-twin goor configuration

(0-131) to be used as a standard valuation to product the response of a tota-tandar pair (0-136) and a single tandar pair (0-136). This implies that any valuable is the inventory can be used as a standard valuation. The combination of evaluation and production can thus be performed extentamentally by the collected valuation. Thus, at each other conducted, the response of all valuation in the inventory can be producted as a part of the evaluation.

7.3.3 Balantim Mileste of h. c and a Passessers

Pigure 42 stands a typical plot of the calculated equivalent total force and of its force compensate with respect to the parameters k, a and a. The maximum restating force due to the k parameter is seen to be about total that of the restating force die to the a parameter. If design procedures would be directed toward greater subilization of the k parameter it appears that a more legical backs for design could be had.

mercan vill

STREETS OF SESTELECTION OF PASCES ON SUMMET PERFORMANCE

It is generally accepted by personnal concerned with sirfield personnes that the response of the personnet-subgrade system is a function of the number of wheels on an aircraft year, its gross load, tire processes, width of tire print, ambient conditions and the lateral distribution of passes (References 1, 2, 3 and 4).

A method for quantifying the cumulative effect of the lateral distribution of passes, type of sireraft and arbitat conditions in the context of paramet evaluation is presented. This method expands the esseept of cumulative total paramet deflection presented by Righter and Nerr (Reference 12). They found that a paramet could only accounts a cortain number of feet of cumulative total deflection prior to the esset of distress. Their data base consisted of AASED hard feet data, data collected from Pease and Castle Air Perce Rasse and from prototype testing on specially designed test sections at Kirtland Air Perce Rass. Analysis of the data should that a threshold cumulative total peak parament deflection, at which severe distress secure is asphaltic sirfield paramete, was about 2000 feet. For asphaltic highway paraments the threshold, was about 1200 feet.

8.1 Page to Coverage (o/s) Entire Recoi on Tire-Contest Area Width

The pass to coverage (p/c) ratio, in cancept, relates the number of passes made by a vahicle across a pavement-section to the number of passes over a selected point on the pavement. This ratio has been

related conventionally to the number of whosle on an adversary landing good, the width of the time print and the leasest distribution of the adversary whosl paths.

The precedures (References 1 and 57) have been used to define the p/c ratio. Both of these procedures use as principal parameters the width of the tire print and the lateral distribution characteristics of the passes over a particular paramete section. The TAA procedure (Reference 1) defines p/c using the relationship

WANT

c - coverages

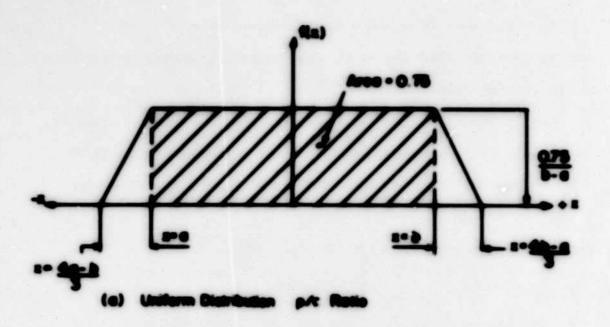
N - Masher of whoele of goor of aircraft

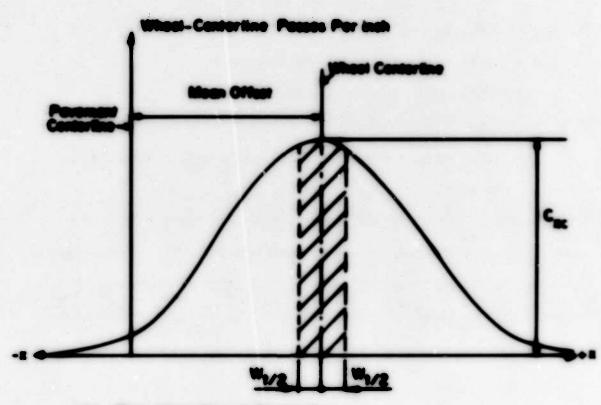
T - Traffic width in fact (generally, 7.5)

W - Width of tire contact area of one tire in inches, and

D - Cycles of eperation (one cycle consists of one landing and one tabout!)

The constant, 0.75, is based on the assumption that the lateral distribution of aircraft can be approximated by a uniform distribution as shown in Figure 70(a). In concept, one cycle of operation is excentially two passes. Hence using D = p/2, equation (24) becomes,





(b) Theoretical Narmal Distribution

Pigure 70. Leteral Distribution of Aircraft Wood in Besign of Airfield Payment (after Moong [Reference 57])

hand on measurements of the lateral distribution of aircraft on vertices runways and taxiways, Holong (Reference 57), proposed the use of a normal distribution instead of the uniform distribution, to describe the lateral distribution of aircraft. He gave the following approaches for the p/c ratio:

where V_{ξ} is the width of the tire contact area, ${}^{**}C_{3\xi}$ represents the number of wheel-conterline passes per lack of width, per aircraft pass, which accumulate at the point where maximum accumulation occurs. Coverage c is defined as the maximum number of wheel passes at the point of maximum accumulation. Figure 70(b) illustrates the p/c ratio concept as applied to a single wheel. In concept, with reference to Figure 70(b), only the effects of tire prints or partial tire prints applied to the surface within the width V_{ξ} are considered in this method.

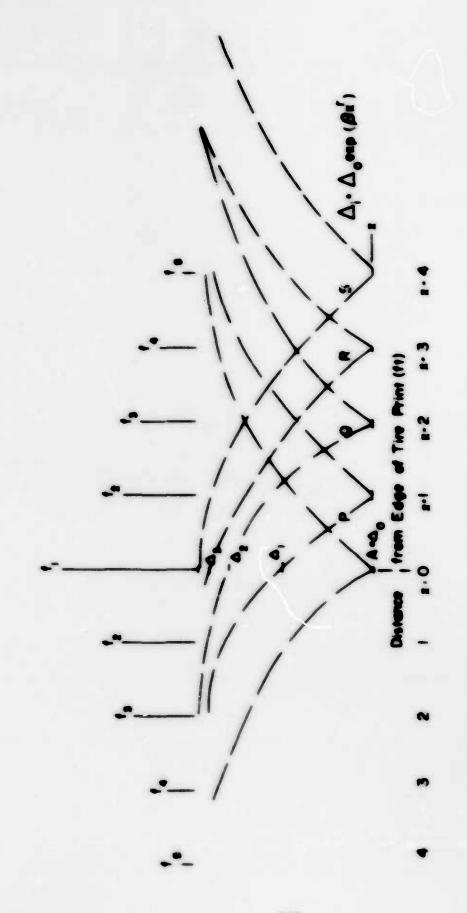
The current FAA procedure and RoSong's proposed method both use the tire print (contact area width) as a key parameter in the definition of the p/c ratio. The method developed here uses the measured dynamic lateral deflection prefile, due to the prototype land, to define pass to coverage; and relates this ratio to performance extending the peak cumulative deflection concept introduced by Highter

8.2 Deflection-Resed p/c Retie

The fundamental concept underlying the deflection-based p/c ratio is founded on the following reasoning. The lovel of significance of a loaded wheel during one pass over a pavement section is indicated by the measured lateral dynamic deflection basin. As the lateral distance from the load wheel increases, the work done by the load decreases. Hence, immediately under the wheel the influence is greatest; whereas, at a point where the measured deflection is zero, the influence is negligible.

under the visel to zero at some distance from the visel; for a p/c ratio to be meaningful, it should account not only for the induced energy level corresponding to the maximum deflection but for all the levels in the range. The principal reasoning have is that the pavement-subgrade system "remembers" not only the maximum deflection but all load induced deflections. Hence, in general, large load induced deflections at a point result in greater work done and remembrances of it, than do smaller load induced deflections.

With reference to Figure 71, each of the curves labelled P, Q, R, S represents a series of peak lateral deflections. If one pass of a load wheel is applied at a point corresponding to x = 0, the maximum deflection (at x = 0) would be $A = \Delta_0$. If the same load had been applied at x = 1, a maximum deflection of Δ_1 would have been measured at x = 0. If the load had been applied at x = 2, then Δ_0



Pigure 71. Overview of the Concept Used in the Deflection based p/c Ratio Calculations

would have been measured at x = 0; similarly for a load at x = 3, etc.

One coverage then, in concept, is the cumulative peak deflection at the point in question that corresponds to maximum deflection A under the wheel. For example, if one pass of a load wheel is made at x=0, then one coverage results at that point. On the other hand, one pass made at x=3 corresponds to the fractional coverage Δ_3/A at x=0.

Suppose that f_0 through f_0 are the discrete frequencies at x=0, 1, 2, 3, ..., n, of the distribution of passes of aircraft laterally about x=0. Then the cumulative peak deflection at x=0 due to this frequency distribution becomes

where N is the number of passes and Δ_1 are the peak deflections corresponding to the points in question. The coverages N due to the N passes is then,

Therefore,

$$\frac{\frac{N_{p}}{N_{c}} \cdot \frac{p}{c} \cdot \frac{\Delta_{o}}{\frac{n}{\sum_{i=0}^{n} \Delta_{i} \cdot \ell_{i}}}$$

Equation (24) is seen to have the same form so the expression for the equivalent permebility of a layered system (Enforcese 67)

$$k_y = \frac{d}{\int_{-\infty}^{\infty} dx/k_x}$$
 (25)

where k_y is the equivalent vertical permeability, $d_{\tilde{k}}$ and $k_{\tilde{k}}$ are the thickness and coefficient of permeability respectively of layer \tilde{k} , and d is the sum of all the thicknesses.

6.3 Effects of Leteral Metribution

The affects of the frequency and of the lateral distributions of aircraft on the cumulative peak deflection and the deflection-based p/c ratio were studied.

Equation (2) in the form,

who used to calculate the lateral dynamic deflection profile, where Δ_j and x_j are the deflections and distances from the edge of the tire print at measurement points j in a direction perpendicular to the direction of motion of the wheel. Δ_g , δ , and r are the parameters describing the maximum dynamic lateral deflection basis as discussed in paragraph 3.2.1. The symmetrical values, shown in Table 7, will be used to illustrate the procedure. These values were obtained from data collected at Site 1, Egilic AFB, in North 1976.

Table 7

4	- 0.00479
8	0.001
•	- 1.021
x, (ft)	A _j (ft)
0	0.00479
1	0.00215
2	0.00094
3	0.00041
4	0.00018

Four discrete distributions are shown in Figure 72 for an assumed single wheel load. Figure 73 shows the effect of the four assumed lateral distributions (keyed by number to Figure 72) on the cumulative deflection. Figure 74 shows the effect of the scatter of the frequency distribution, as given by its standard deviation based p/c ratio. These results show that,

- 1. the assumption of a uniform distribution of passes (distribution ① on Figure 72) yields the least cumulative peak deflection per pass and,
- as would be expected, the distribution with the smallest scatter yields the greatest cumulative peak deflections.

8.4 Discussion

The results showed the dependence of cumulative pank deflection

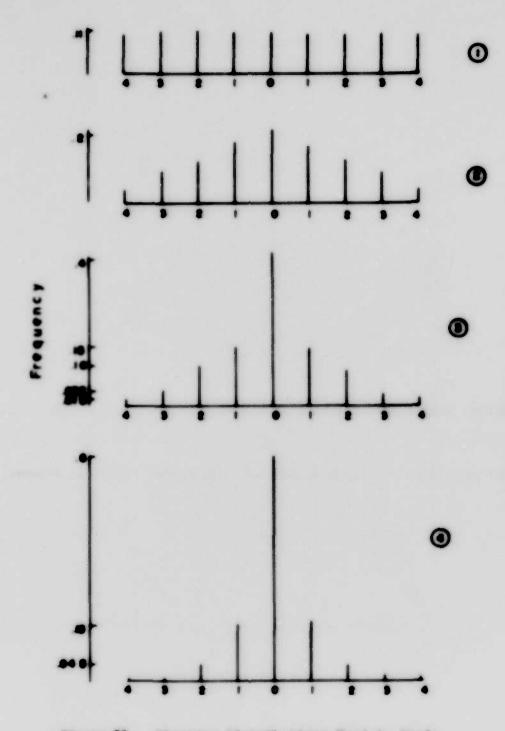
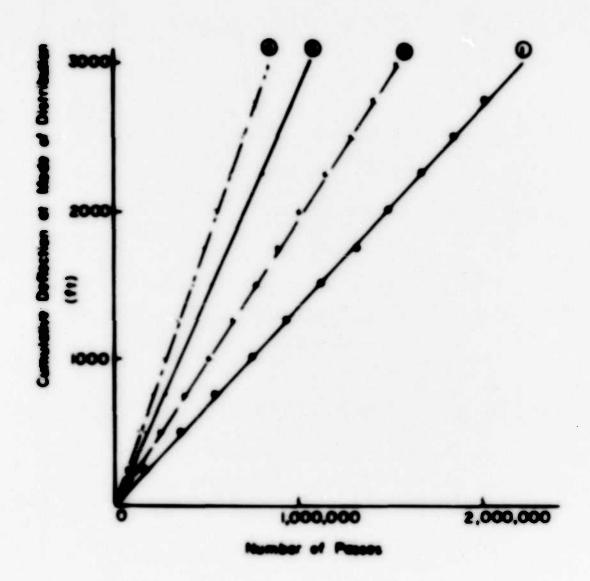


Figure 72. Discrete Distributions Used in Study



Pigure 73. Cumulative Deflection vs. Passes for Four Discrete Distributions of Passes

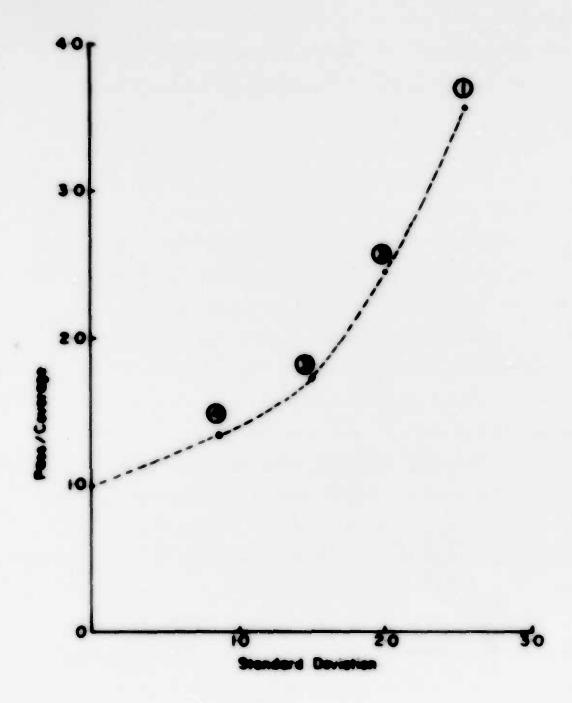


Figure 74. Effects of Standard Deviation on Deflection-Based p/c Ratio

on the lateral distribution of passes and the peak deflection profile.

The p/c ratio thus provides a mans of quantifying the variability of the input (alreraft loads) characteristics and pavement-subgrade system response (output).

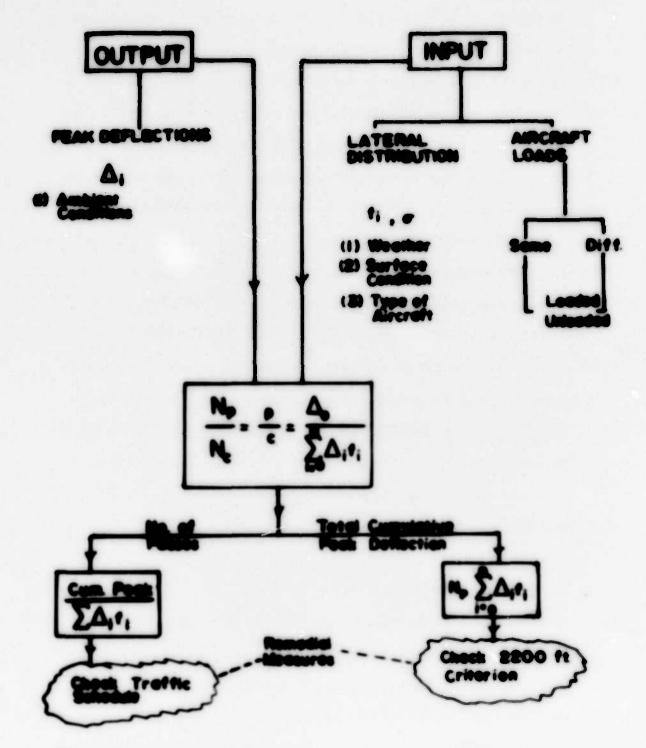
With reference to Pigure 75, input variables consist of aircraft leads and lateral distribution characteristics of leads. These can include vind effects, surface condition, type of aircraft, and quantity of fuel aboard. The main output variable is the peak payment deflection profile. Inherent in this unacurement are temperature offects and other arbient conditions. Pigure 75 shows hav the p/c ratio can be utilized to estimate the amount of energy imported to a povement-subgrade system due to variable inputs and also hav to estimate the number of passes required to induce a certain energy imput as given by the total cumulative peak deflections.

For example, assuming that the lateral distribution of passes has a standard deviation Of 2.0 (distribution ② in Figure 73), the number of passes required to reach the threshold total cumulative pask deflection value of 2200 feet is 1,160,000 passes or 500,000 cycles of operation.

On the other hand, if the assumed lateral distribution of passes has a standard deviation of 2.66, then 500,000 cycles of operation would yield a cumulative peak deflection value of about 1500 feet, which is well below the threshold value for that distribution. For this case 850,000 cycles of operation could be telerated before severe

and the contraction of the contr

The threshold value of 2200 feet was given by Highter (Reference 12) vithout due regard to the lateral positioning of traffic. Lacking additional information at this time the same value was used in the present study.



Pigure 75. Overview of Evaluation Scheme Using Deflection Prequency Relationship

distress might be expected.

As another example, suppose that due to everlay or recurfacing of pavement, parameters and peak deflections change to those given in Table 8 after 290,000 cycles of operation, then 700,000 cycles of operations (rather than 290,000) would now be required to attain the threshold value of 2200 feet. This assumes that the section "remembers" the leading prior to resurfacing. If resurfacing had not taken place,only 290,000 cycles of operation would have been required to show effects of distress.

The above examples have attempted to show that actual measured distribution data such as that provided by MoSang (Reference 57), and actual measured deflections (which account for ambient seasonal effects) can be combined to estimate the cumulative damage effects due to prototype loads. Once a standard site and standard vehicle have been selected, the damage effects of various magnitudes of loads

Table 8

4.	• 0.0012
2	0.2256
t	- 1.7770
*, (ft)	Aj (ft)
0	0.00120
1	0.00096
2	0.00055
3	0.00024
4	0.00045

¹⁴ See Footnote 13.

and poor configurations and their distributions can be estimated. By unnitoring the effects of specific personne sections, the direction which remodial uncourse should take could be established [see Highter and Herr (Reference 12)].

SECTION IX

SUPPLARY

The findings of the present research effort can be summarised under three basic categories, namely:

- 1. neadestructive pavement testing equipment development,
- nondestructive pavenest evaluation and behavior prediction, and
- entension of cumulative peak deflection-damage-energy related condepts.

Two equipment systems for use in the mendestructive evaluation of flexible pavements have been developed. They are the LED system and the LVDT system. The feasibility of rapidly measuring povement deflections caused by moving leads using the developed equipment has been investigated. Field testing of the developed systems using prototype leads was conducted.

A mondestructive pavement evaluation methodology using transfer function theory has been developed. Parameters of the transfer function which reflect site characteristics have been isolated and related to performance of two sites. Parameters of the lateral peak deflection profiles have been investigated. Simplified procedures for estimating the parameters of the transfer function and parameters (CDR, etc.) used in design have been suggested.

A methodology has been presented for predicting the deflection if the response at a standard site to a standard load vehicle and another vehicle, and the response of the other sites to the standard vehicle are known. The same transfer function parameters that are used in the evaluation phase are used in the prediction school.

A method of accounting for the lateral distribution of aircraft on a personant, and variability of input and output to the personant-outgrade system has been presented. The interrelationship among the components of MT personant evaluation-response prediction-cumulative damage prediction based on deflections caused by prototype leading has been discussed. The deflection based pass to coverage ratio was shown to be a vital link in the scheme.

SECTION X

CONCLUS TORS

This research effort has attempted to cover a spectrum of activities that comprise numbertructive testing and evaluation of flexible airfield pavements. The following conclusions have been reached as a result of this study.

- Rapid nondestructive measurements of pavement deflections due to moving prototype loads can be obtained in the field using the LED system "breadboarded" in this research effort and the companion LVDT system.
- 2. The parameters of both the transfer function and the peak lateral deflection profile reflect characteristics of the pavement-subgrade systems and can be used as the basis of a nondestructive pavement evaluation games. Some common parameters such as CBR and undulus of subgrade reaction can be obtained from those defined in this study.
- 3. The developed methodology, using the same parameters of the transfer function that are used to evaluate a parament system, can be used to predict the deflection response function of a site to different magnitudes and configurations of loadings.
- 4. The concept of cumulative peak deflection (Reference 12) as it reletes to energy induced in a pavement-subgrade-system due to load repetition, was extended to account for the lateral distribution of eigereft. The frequency characteristics and the number of passes of eigereft are shown to

effect, significantly, the amount of distress a pavement might evidence in time.

SECTION XI

RECOMMENDATIONS

This work has afforded the opportunity to consider in some depth a significant portion of the current state of the art of numdestructive testing theory and methods used on airfield pavements. Based on the following of this work the following recommendations are forwarded:

- The use of prototype loadings in evaluation should be encouraged.
- The development of a scheme, based on the use of nondestructively measured pavement deflections due to prototype loads, similar to that briefly discussed in paragraph 8.4 of this work should be pursued.
- The prototype of noncontact measuring equipment shown to be feasible in this investigation should be developed. This development should take advantage of the current state of the art in optical design.
- recently been made (Reference 57). A similar information gathering study should be initiated to collect data on numbers and types of aircraft using specified pavements at both commercial and military installations. Pavement deflections should be obtained regularly at the sections being monitored. This data base is necessary if the concepts and the preliminary conclusions reached at this time are to be validated and generalized.
- 5. The speedy development of a prototype device to move over the

pavement sections and simultaneously measure its own dynamic deflection basin without significantly interrupting traffic would greatly facilitate the information gathering phase.

This would simultaneously provide a common denominator or standard vehicle.

6. Research should be directed towards determining the influences of the components of a pavement-subgrade system on the k, c, and n parameters. The success of this endeavor would be a stem towards optimizing design procedures and the use of paving materials.

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APPENDIX A

RESEARCH AND DEVELOPMENT

0.

HEASUREMENT SYSTEMS

The broadboard model of the LED system was subcontracted by Perdue University to Science Applications Incorporated (SAI), New Mexico, during the period May, 1974 to December, 1975. The contents of Section a of this Appendix is based principally on SAI's Final Report (Reference 56).

During the period January 1975 to June 1976, the breadboard model electronics was upgraded at Purdue University and work was done to convert the static measurement system in order to make it more compatible with a moving beam-moving load concept. Section b of this Appendix presents a summary of this work.

Section c of this Appendix presents some details of the Linear Variable Differential Transformer System (LVDT) developed at Purdue University by G. Baladi (Reference 52) for a study sponsored by the PRMA.

a. The LED System

Several optical systems were considered during the development of the LED system. These include:

- A retroflector module whose motion is detected by a laser beam.
- 2. A limited version of the global Moire method.
- 3. Focal point shift detection.
- 4. Triangulation of a light spot on a surface.

As the triangulation method was finally selected for development, only the summary details concerning this method are presented herein.

The breadboard model employing the triangulation method is shown in Figure A-1. This arrangement has an image shift δ_2 (Figure A-2) perpendicular to the line of view which is related to the pavement deformation by the equation,

$$\delta_2 = \frac{\sin (y + \beta)}{\sin y} \cdot \frac{\delta}{\tan (90 - y)}$$

where m is the magnification of the deformation. Other symbols are designated in Figure A-2. The pavement "roughness" function is represented by $f(\mathbf{x})$.

The straight line of the input center ray is given by

$$y = (x-b) \tan (90 - y)$$

10

$$y = x \tan (90 - y) + a$$

The intersection of the input ray with f(x) is at x_0 , giving

$$f(x_0) = x_0 \tan (90 - y) + a$$

and the intersection with the displaced f(x) is at x_1 , giving

$$6 + f(x_1) - x_1 \tan (90 - y) + a$$

Using a Taylor expansion near x to get $f(x_1)$ and dropping higher orders gives

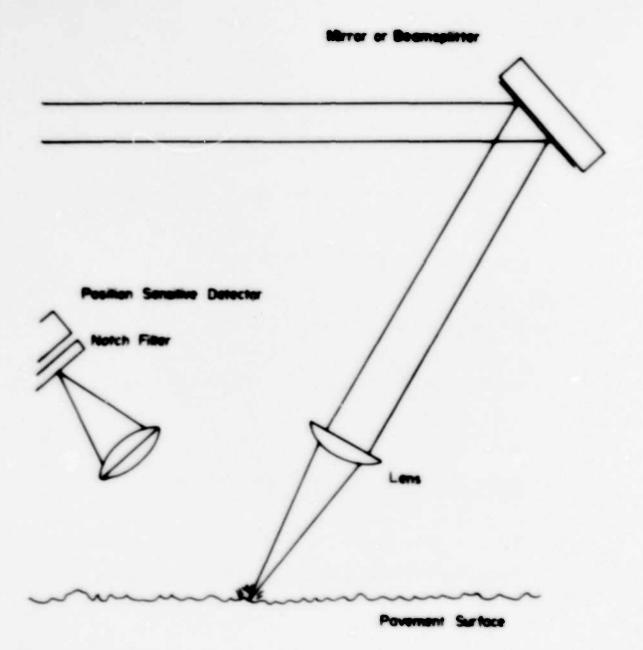


Figure A-1. Triangulation Arrangement

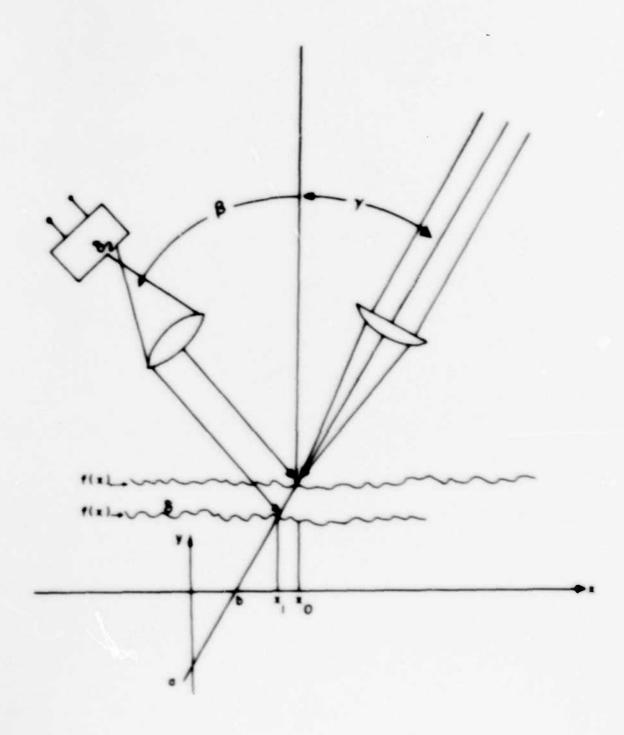


Figure A-2. Triangulation Parameters

$$f(x_1) = f(x_0) + \frac{f'(x_0)}{1} (x_1 - x_0) + \frac{f''(x_0)}{2} (x_1 - x_0)^2$$

Substitute this into the second intersection equation and subtracting the two gives

$$(x_1 - x_0)$$
 ten (9) - y) - $\delta = f'(x_0)(x_1 - x_0) = \frac{f''(x_0)}{2}(x_1 - x_0)^2$

which is solved to obtain,

$$(x_1 - x_0) = \frac{2\delta}{[f'(x_0) - \tan(90 - y)] + [f'(x_0) - \tan(90 - x)]^2 - 2f''(x_0)\delta}$$

The image point is shofted by the component δ_2 normal to the axis of the imaging lens where,

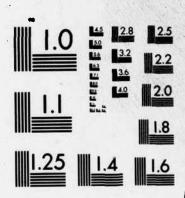
$$\epsilon_2 = \pi \frac{\sin (\gamma + \beta)}{\sin \gamma} \cdot (x_1 - x_0)$$

The image motion can then be obtained as a function of the arrangement parameter (v), the surface $f(\mathbf{x}_0)$ and the surface displacement, (δ_1) by the equation

$$\delta_2 = \mathbf{x} \, \delta_1 \, \left[\frac{\sin \, \mathbf{y} + \mathbf{g}}{\sin \, \mathbf{y}} \right] = \left[\frac{\cos \mathbf{y}}{\sin \mathbf{y}} - \mathbf{f}'(\mathbf{x}_e) \right] - \left[\frac{\cos \mathbf{y}}{\sin \mathbf{y}} - \mathbf{f}'(\mathbf{x}_e) \right]^2 - 2\mathbf{f}''(\mathbf{x}_e) \delta$$

However, the pavement is rough on the scale of the focused spot on the runway. To eliminate the dependency on surface roughness, there are two alternatives. First, make the focused spot a focused line long enough that the effects of roughness are averaged to zero and second, reduce the angle between the pavement normal and input ray, y, to zero. The image motion equation when y = 0 reduces to

NONCONTACT NONDESTRUCTIVE DETERMINATION OF PAVEMENT DEFLECTION UNDER MOVING LOADS(U) PURDUE RESEARCH FOUNDATION LAFAYETTE IN M E HARR ET AL AUG 77 FAA-RD-77-127 DOT-FA73WAI-361 F/G 13/2 AD-A847 161 3/4 UNCLASSIFIED NL III



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The sensitivity of the image motion is maximized when β goes to 90° but that is not physically possible. Also the collection of light for imaging depends on the solid angle of the imaging lens with respect to the pavement spot. The larger β forces the solid angle to smaller values.

A triangulation arrangement using a laser input was breadboarded and tested. Figure A-3 shows the laser setup in the laboratory at SAI. Some typical displacement signal curves are given in Figure A-4. These data are for four different samples and $\gamma = 23^{\circ}$, $\beta = 45^{\circ}$. Several other arrangements and the theory given above demonstrated that these calibration curves become smooth and repeatable when $\gamma = 0^{\circ}$, $\beta = 45^{\circ}$ using a cylinder lens to focus a line on the surface. Also shown in Figure A-5 is a curve which was obtained by using the sum of the two position signals as a normalization factor. An extension of the linear range is obtained with the normalization. Using the normalization method also yields a further more important advantage, that is, that the reflectivity of the surface is removed from affecting the signal. Figure A-6 shows normalized data for concrete and gray sandpaper which have the maximum and minimum reflectance of the sample set used. Their reflectance difference was greater than a factor of 2. The differences in the two signals appear in the regions where normalization has extended the linear range. The major contributor to this error was the inaccurate measurement of the normalization constant. If higher precision electronics had been used in this breadboard, most of the

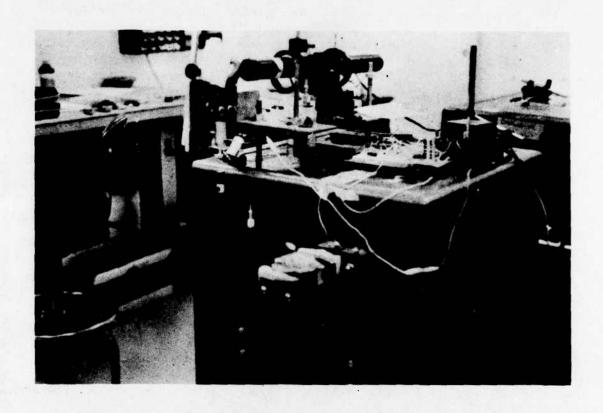


Figure A-3. Laser Setup in Laboratory

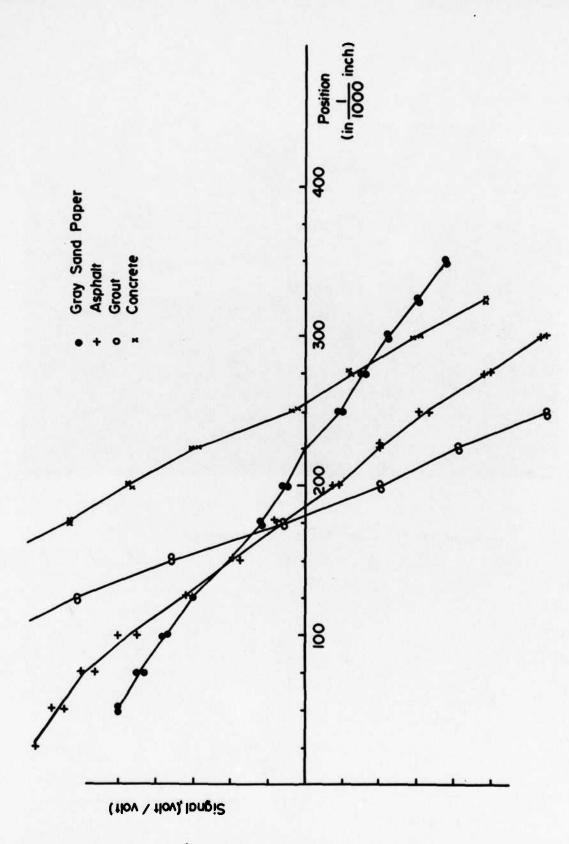


Figure A-4. Typical Triangulation Displacement Curves

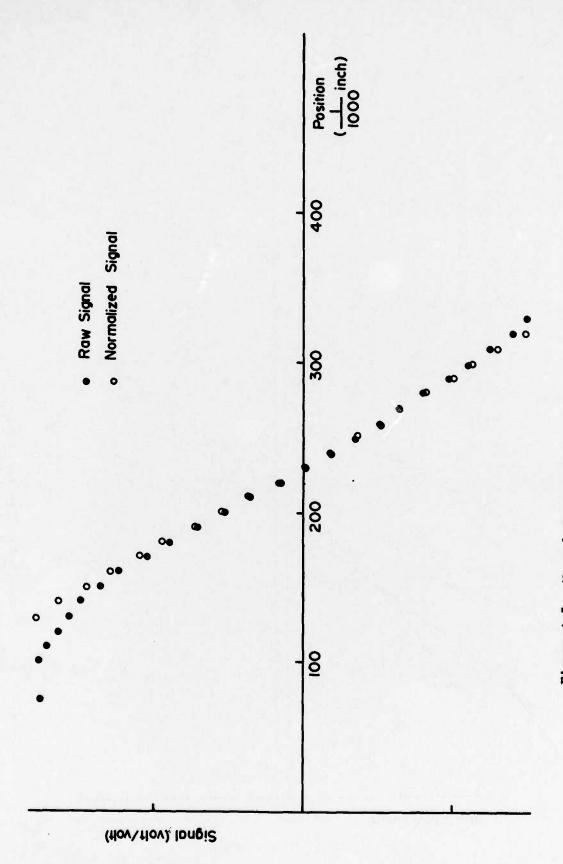
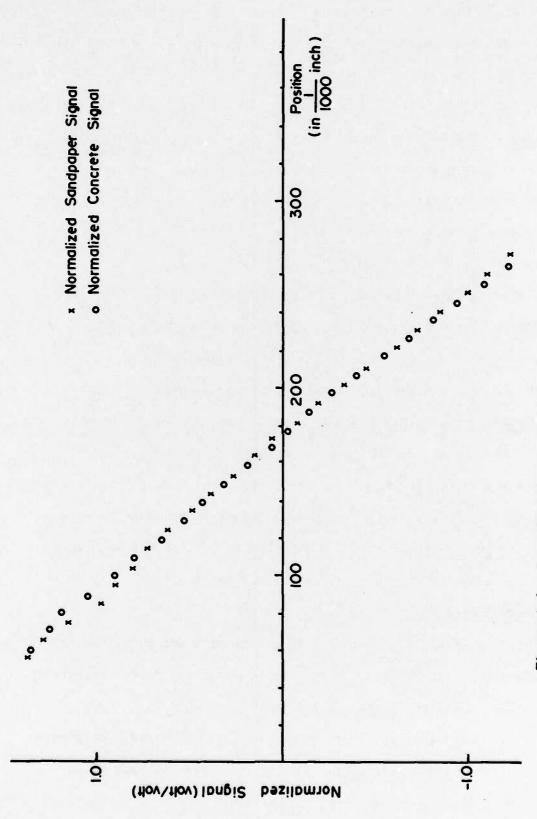


Figure A-5. Normal Incidence Displacement Curve

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Figure A-6. Normalized Deflection Curves

signal differences would disappear.

The choice was made to use LED sources in a triangulation arrangement to deflections. This arrangement is shown in Figure A-7 and is the same triangulation arrangement as was tested before except that the light emitting diode replaces the input laser beam and the cylinder lens has been modified by adding a convex-plano spherical lens to form a composite spherical-cylinder lens. This lens arrangement was developed by mapping the three dimensional distribution of energy using an aperture and photodiode. The peak of the intensity was found for each lens arrangement until an optimum arrangement was determined. The displacement signal, shown in Figure A-8 demonstrates effects of the change in sharpness of the focal zone. The curve shows a change in slope in either side of the central focus region that is due to the larger area for the "line" focussed on the surface.

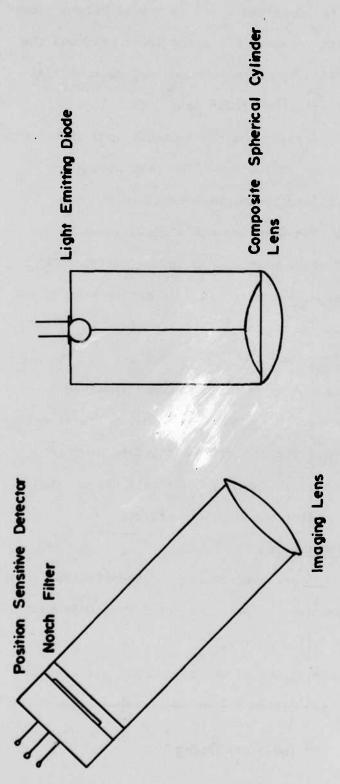
The basic optical principle as described herein remained unchanged throughout the course of system development. The electronics and the mechanical attachments were however modified to be more consistent with the concepts of the moving load-moving beam which evolved subsequent to the conclusions of SAI's effort.

b. Additional Work on LED System

Repackaging of the breadboard electronics and modification of the software supplied by SAI was done by personnel at Purdue University.

This work consisted of the following:

 Converting the basic wiring of the breadboard electronics to printed circuits. Figure A-9 shows the repackaged



. Pavement

Figure A-7. Light Emitting Diode Triangulation Arrangement

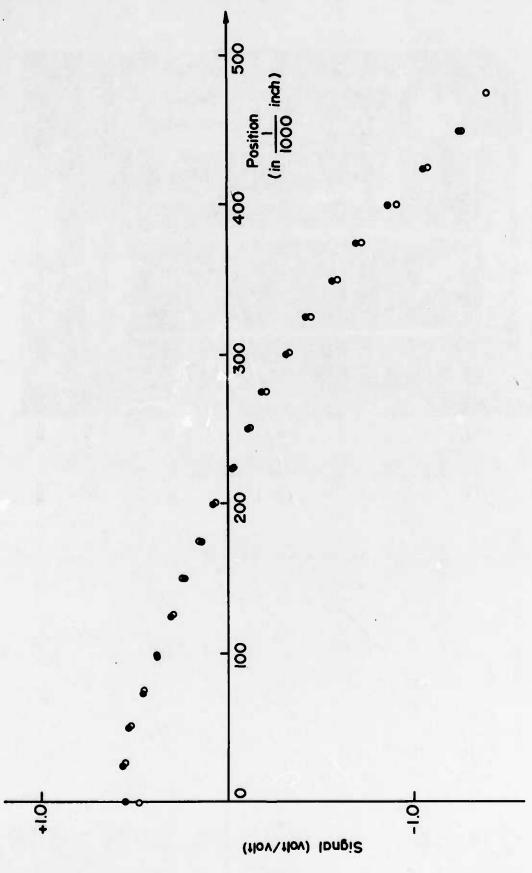


Figure A-8. Light Emitting Diode Displacement Curve

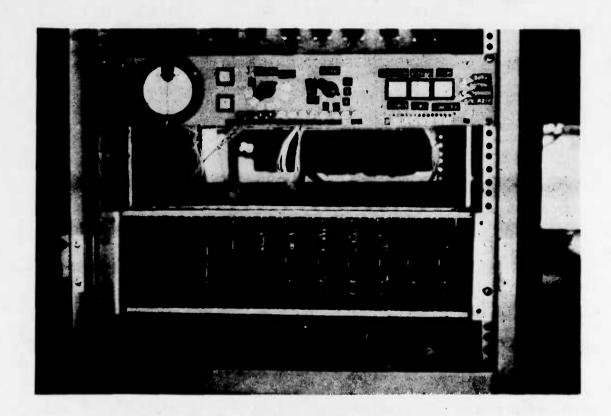


Figure A-9. Repackaged Electronics

electronics.

- Testing the basic moving beam-moving load concept.
 Figure A-10 shows the basic test setup.
- 3. Changing the software to be compatible with the new concept of a moving beam-moving load. Figure A-11 shows a flow chart of the modified software.

c. The LVDT System

The LVDT system was designed and fabricated at Purdue University as part of a study sponsored by the Federal Highway Administration (FHWA). Because an overview of the system has already been presented previously in this repot only the details of the system components are present here.

The use of the system to measure response functions has been discussed previously in this report. For additional details see Baladi (Reference 52). A listing of the basic components of the LVDT system is as follows:

- ·Light Weight Aluminum I-beam, 80 inches long, 12 inches wide
- ·Linear Variable Differential Transformers (LVDTs)
- · 2 Way Screw Jack
- ·Pipe (pivot)
- ·Small Screw Jacks under LVDTs
- ·6-Channel Oscillograph Light Beam Recorder
- ·Kodak-2022-light Sensitive Paper
- ·Sears Electric Power Generator
- ·AC-DC Power Converter
- ·Micrometer-Caliper



Figure A-10. Test Setup Moving Beam - Moving Load Operation: Purdue University

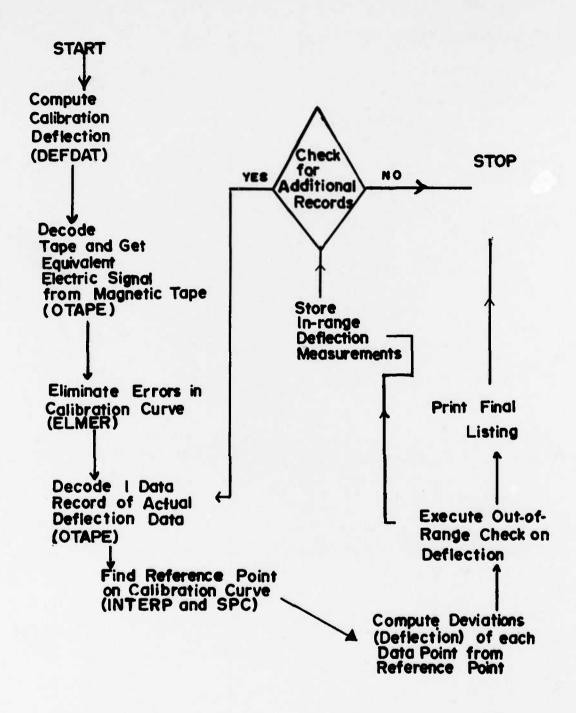


Figure A-11. Summary Flow Chart of LED System Software

- ·Hydraulic Scales
- ·Heavy Duty Silver Tape
- ·Thermometer
- ·Miscellaneous

APPENDIX B

SERIES A: FIELD INVESTIGATIONS

The Series A field investigation program consisted of conducting field tests at Eglin Air Force Base, Florida and Pease Air Force Base in New Hampshire during the period of this study.

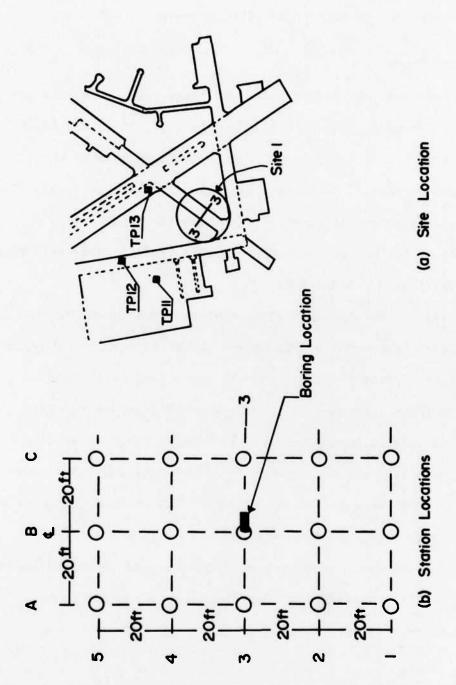
a. Eglin Air Force Base

Two sites were selected at Eglin Air Force Base. The location of the sites is shown in Figures B-1(a) and B-3(a). At each site, LVDT gages were embedded in the pavement. At Site 1, five gages were installed and at site 2, four gages were placed in the pavement. The locations of the gages are shown in Figures B-1(b) and B-3(b).

Details of the boring locations and site profile characteristics are shown in Figures B-2 and B-4.

A summary of the gage installation procedure is shown in the photosequence in Figure B-5. Figure B-6 shows the details of typical installed gage. After installation, the gages were calibrated as discussed in Boyer (Reference 11), and pavement response functions

Grids were laid out as shown in Figures B-1(b) and B-3(b) and response functions were determined using the LVDT beam. The gages were first calibrated using the equipment shown in Figure B-7. Then response functions were obtained at each of the grid locations shown in Figures B+1(b) and B-3(b). Figure B-8 shows a typical pass being made by the P-2 fire truck.



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Figure B-1. Eglin Air Force Base: Site 1

Layer							
4	Boring Number	B I	B 2	В 3	B 4	B 5	
8	Layer-A-Asphalt Concrete 2.25 +	2.25	2.0	2.0	1.35	2.0	
(Layer-B- Sand Asphalt 7.0	7.0	7.5	7.0	7.0	7.0	
ی	Layer-C-Medium Sand 168	891	891	891	168	168	

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+ Layer Thickness in Inches

(a) Site Profile Characteristics



(b) Details of Boring Location

Figure B-2. Eglin AFB - Site 1: Details of Boring Location and Site Profile Characteristics

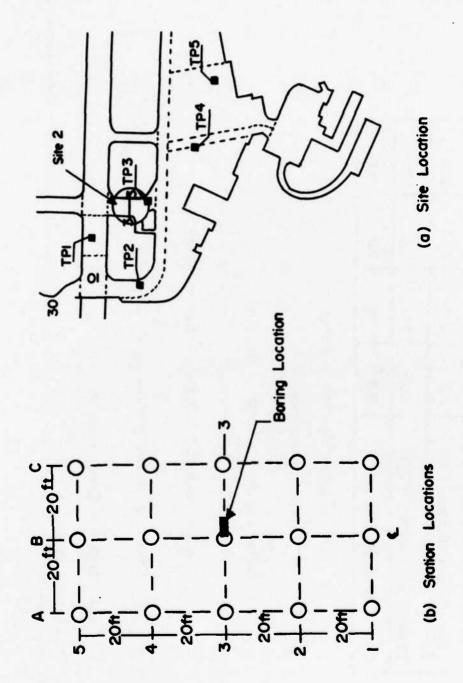
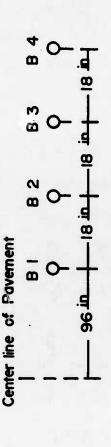


Figure B-3. Eglin Air Force Base: Site 2

Layer	Boring Number	- 8	8 2	B 3	8 4
4 6	Layer A- Asphalt Concrete	1.25	1.125	1.25	1.25
U	Layer C-Sand Asphalt	4	13.5	1.5	12.0
٥	Layer D - Medium Sand	132	132	132	132
ш	Layer E-Silty Sand	48	48	48	48

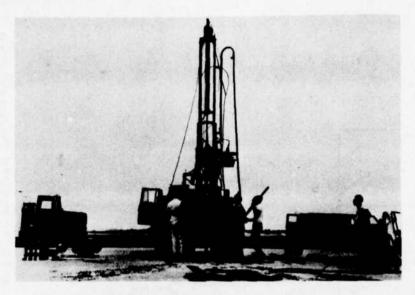
1 Resurfacing 2 Old Surface + Layer Thickness in inches

Site Profile Characteristics 9

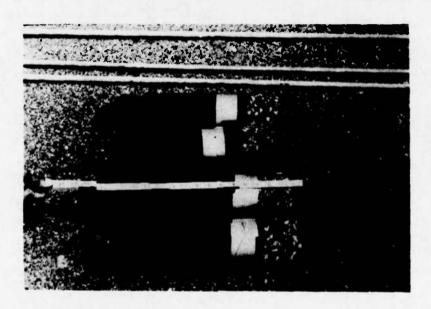


Details of Boring Location <u>e</u>

Figure B-4. Eglin - Site 2: Details of Boring Location and Site Profile Characteristics

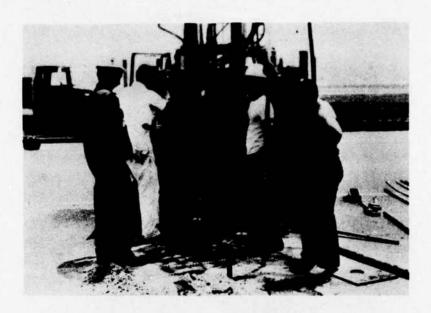


(a) Coring Pavement

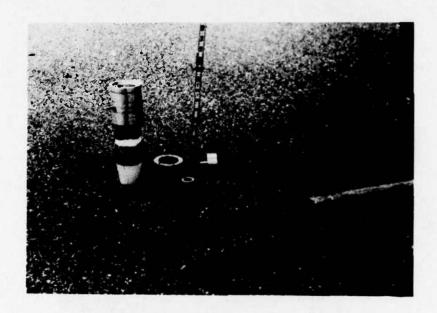


(b) Extract Cores

Figure B-5. Gage Installation Procedure



(c) Drill Hole to about 14 feet

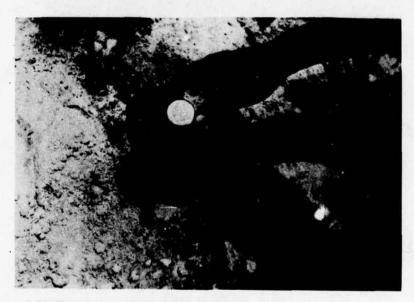


(d) Paraphernalia Ready for Installation in Hole

Figure B-5. (Continued)



(e) Lower Rod, Anchor and Casing into Hole

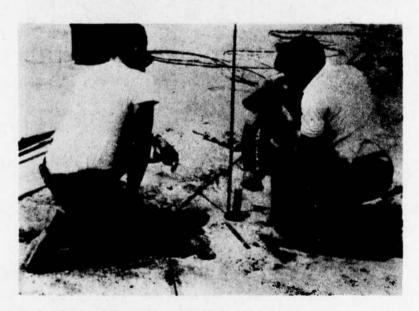


(f) Screw Impact Head to Inner Rod

Figure B-5. (Continued)



(g) Backfill with Sand and Compact



(h) Drive out Anchor Prongs

Figure B-5. (Continued)



(i) Position Upper Paraphernalia



(j) Backfill Around Upper Paraphernalia

Figure B-5. (Continued)



(k) Groove Pavement



(1) Epaxy Upper Paraphernalia to Asphaltic Concrete

Figure B-5. (Continued)



(m) Epoxy Cables for LVDT into Groove



(n) Typical Installation - Partially Epoxied Cables

Figure B-5. (Continued)



(o) Epoxy Cable in Groove to Edge of Pavement



(p) Gages Ready for Element (Sensor) Installation

Figure B-5. (Concluded)

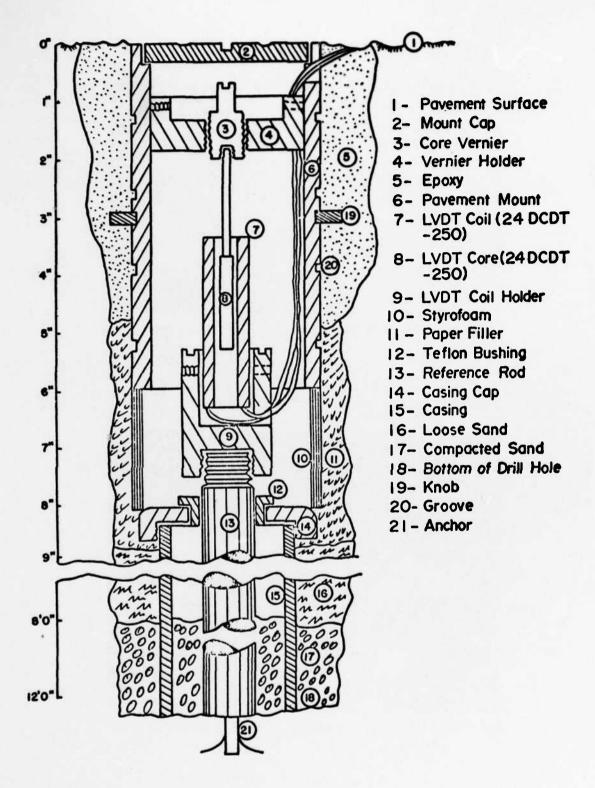


Figure B-6. Gage Installation (after Boyer [Reference 12])

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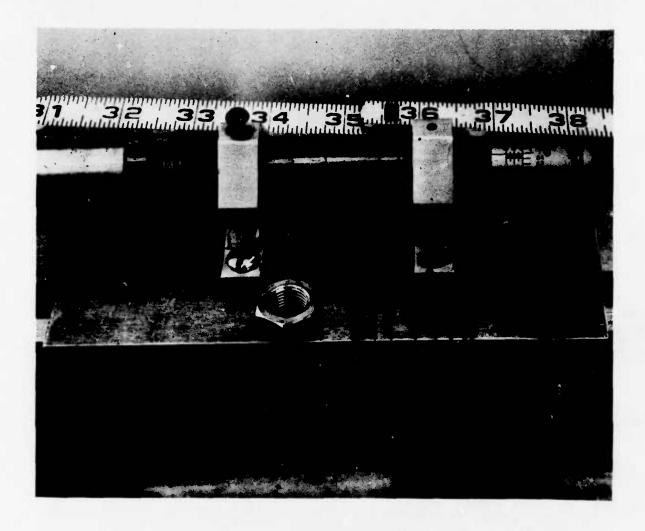


Figure B-7. Calibration Equipment for LVDT Gages



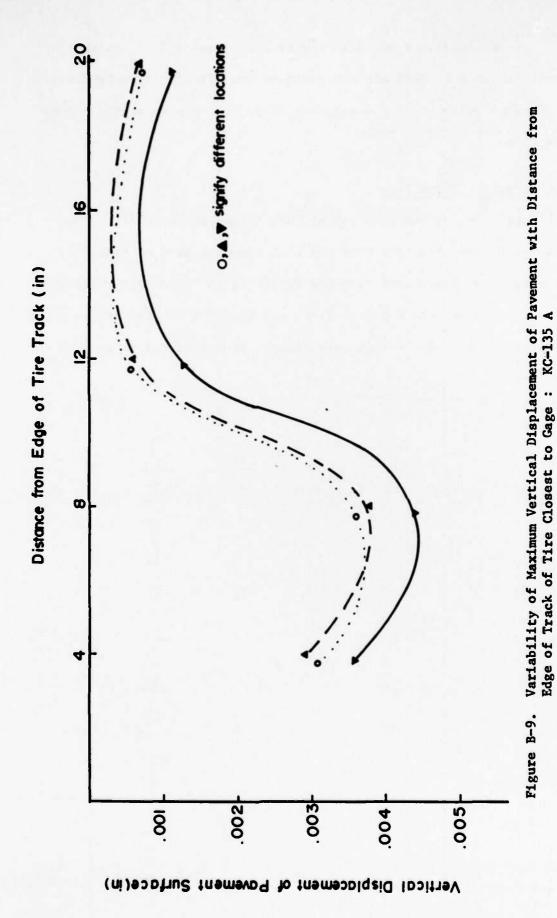
Figure B-8. Photo of P-2 Fire Truck Making a Pass During LVDT Beam Testing

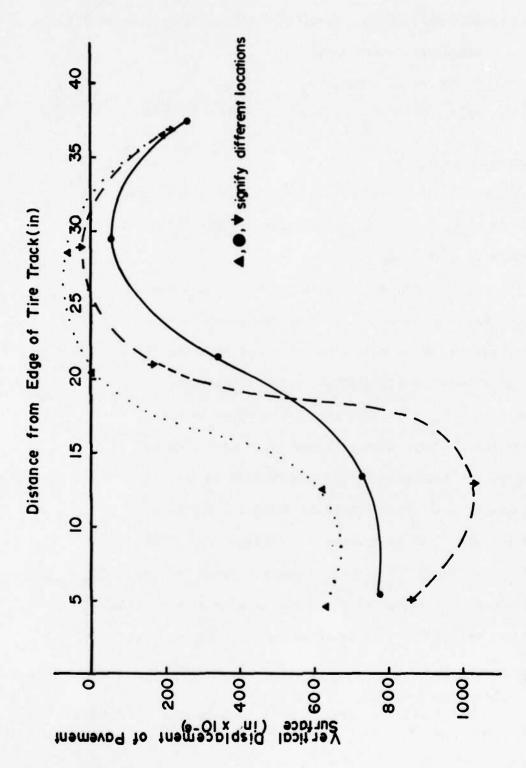
At the locations shown in Figure B-1(b) and B-3(b), attempts were also made to estimate the pavement transfer function by dropping a 60 pound weight onto the pavement close to the gage at the tip of the beam.

b. Pease Air Force Base

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One site, on the main runway, was tested at Pease AFB in New Hampshire. At this site only the LVDT beam was used to obtain measurements of pavement response functions in the manner previously described, using the P-2 fire truck and the KC-135A aircraft as prime movers. Typical deflection measurements are presented in Figures B-9 and B-10.





SOURCE CONTINUES PROPERTY SOURCES SOURCES SOURCES CONTINUES CONTINUES SOURCES SOURCES

Variability of Maximum Vertical Displacement of Pavement with Distance from Edge of Track of Tire Closest to Gage: P2 Fire Truck Figure B-10.

APPENDIX C

COMPUTER PROGRAMS

This Appendix contains the details of the computer program for,

- 1. Signature determination
- 2. Parameter determination
- 3. Prediction

a. Signature Determination

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The signature as defined in the text is calculated using the equations (3) through (10) and the subroutines EXACFIT and SIGNTR. The procedure is as follows:

With reference to Figures C-1 and C-2, a value of $r_{peak}(FXP)^{17}$ is assumed (0.2) then the left hand side (FLHS) and the right hand side (FRHS) of equation 5a are calculated. Their difference TVAL(1) is then calculated. $r_{peak}(FXP)$ is then incremented by 0.0125 (DFXP) and the left hand side and right hand side of equation 5a, and their difference TVAL(2) is again calculated. The ratio (FDIF) of TVAL(2) and TVAL(1) is calculated. If this ratio is positive, $r_{peak}(FXP)$ is incremented by 0.0125 (DFXP) and the ratio FDIF is again found. This procedure is repeated until the ratio is first negative. This is shown as S in Figure C-2a;

¹⁷ Symbols in capital letters correspond to symbols used in the sub-routine EXACFIT which performs the calculations.

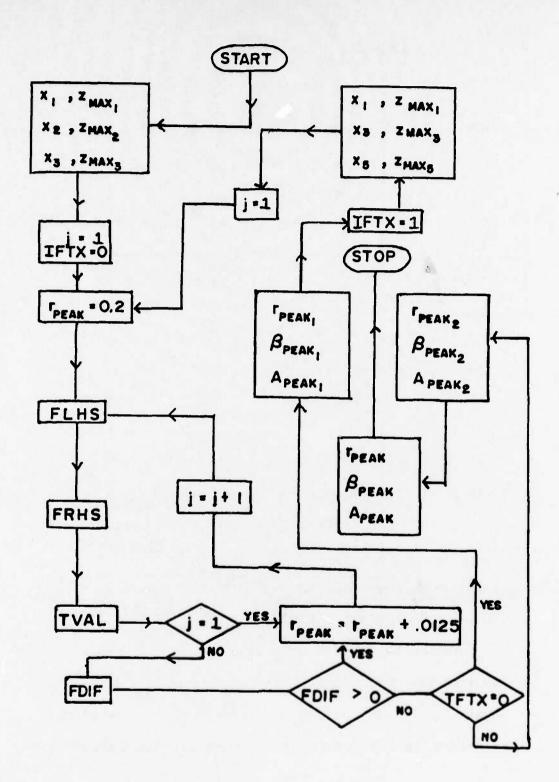
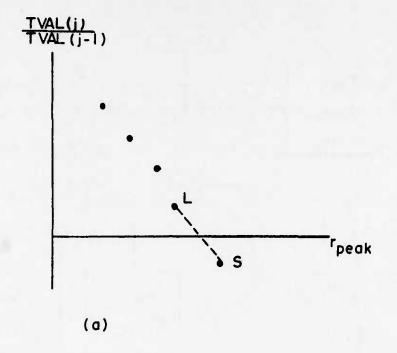


Figure C-1. Summary Flow Chart of Subroutine EXACFIT



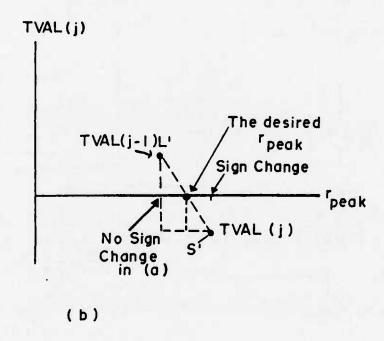


Figure C-2. Details of Interpolation in Subgrade EXACFIT

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the point immediately before, at which the ratio is positive is labelled L. In Figure C-2b, the point before the ratio becomes negative is L' and S' is the point at which the ratio is negative. The point where the difference between the left hand side and the right hand side of equation 5a is zero is found by linearly interpolating between L' and S'. With reference to Figure C-2b

 β'_{peak} is then calculated from equation 5 where

$$\beta'_{\text{peak}} = \frac{\ln z_{\text{max}_1} - \ln z_{\text{max}_3}}{r'_{\text{peak}} - r'_{\text{peak}}}$$
(5a)

and, A' is calculated from equation 3a

$$A_{\text{peak}}^{\prime} = z_{\text{max}_{1}}^{\prime} / [\exp (\beta_{\text{peak}}^{\prime} x_{1}^{\text{peak}})]$$
 (3a)

This procedure is then repeated using (x_1, z_1) , (x_3, z_3) and (x_5, z_5) instead of (x_1, z_1) , (x_2, z_2) and (x_3, z_3) , and values of $r_{peak}^{"}$, $\beta_{peak}^{"}$ and A_{peak} are calculated.

The desired r_{peak} , β_{peak} and A_{peak} are found as the average of the values using the equations,

$$r_{peak} = (r'_{peak} + r''_{peak})/2$$

 $\beta_{peak} = (\beta'_{peak} + \beta''_{peak})/2$

The second line was a second line of the second lin

$$A_{\text{peak}} = (A_{\text{peak}}^{\dagger} + A_{\text{peak}}^{\bullet \bullet})/2$$

The A_{peak} , r_{peak} , and β_{peak} values thus calculated are the parameters describing the maximum lateral deflection profile.

The signature is then calculated using the $r_{p\bm{e}ak},$ and $\beta_{p\bm{e}ak}$ parameter and the basic equation, equation 2a but in the form

$$A(t) = z(t)/\exp (\beta_{peak} x_1^{peak})$$
 (10)

where z(t) refers to the deflection at gate 1 (closest to the wheel), and t is the time. Subroutine SIGNTR performs these computations by direct substitution of z(t), x_i , r_{peak} and β_{peak} in equation (10).

b. Parameter Determination

The k, c and m parameters as defined by equation (1) in the text are calculated using the methodology, concepts and equations presented in Section III in the paragraph with subheading "Parameter Determination". The subroutines EXACT1, NWCONV, SIGNTR, and EXACFIT are called from the main program FNDPRM in order to perform the necessary computations.

With reference to Figure C-3 and Figure C-4, initially, a value of the spring constant k is assumed where

$$k = EQKA = AA1 \times PMAX/DMAX$$
 (29)

where $AA1^{18} = 0.05$

PMAX = 25000, and

DMAX = Maximum value of the signature.

¹⁸ For convenience, the symbols in capitol letters in section (4) are the same symbols used in the computer program FNDPRM.

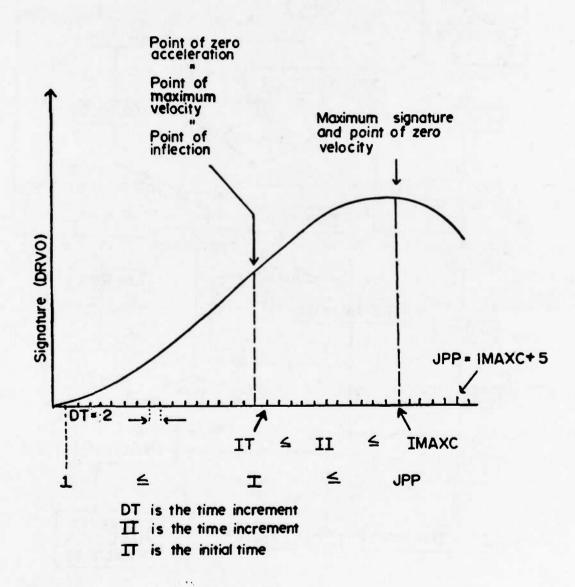


Figure C-3. Definition Sketch for Signature

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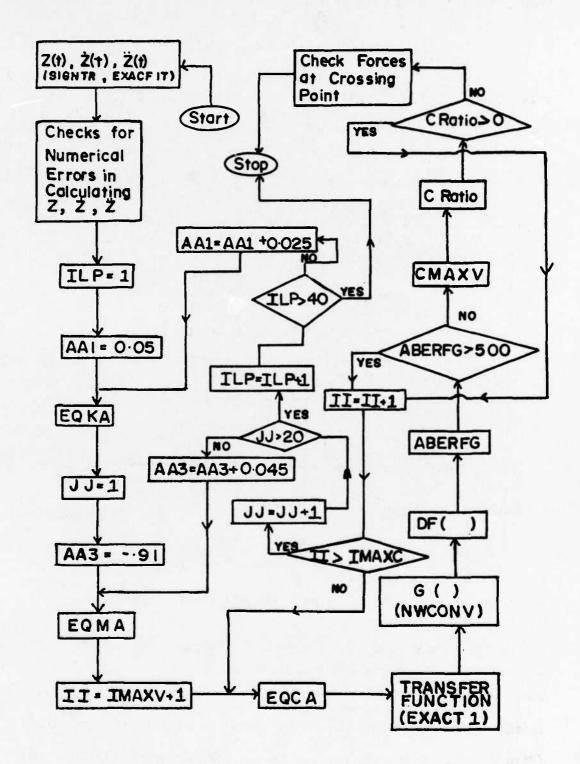


Figure C-4. Summary Flow Chart of Program FNDPRM

Then a value of m is calculated using the equation

$$m = EQMA = AA3 \times PMAX/DRV2 (IMAXD)$$
 (30)

where DRV2 (IMAXD) is the acceleration at maximum deflection and AA3 - .91. Equation 30 is derived as follows: Let

$$k = AA1 \times PMAX/DMAX$$
 (29)

at maximum deflection, i.e., section (3), Figure 7. Using a constant AA2, the equation

is obtained. Therefore

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$$AA2 \times PMAX - AA1 \times PMAX = EQMAXDRV2(IMAXD)$$

$$(AA2 - AA1)PMAX = EQMA \times DRV2(IMAXD)$$
(31)

Letting, AA3 = AA2 - AA1 yields

$$m = \frac{AA3 \times PMAX}{DRV (IMAXD)} = EQMA$$

Next a value of c is calculated using equation (32) where

$$c = EQCA = EQMA \times DRV2(II)/DRV1(II)$$
 (32)

where DRV1(II) is the velocity at time increment II (Figure C-3). The initial value of II corresponds to the point of zero acceleration + 1 (Figure C-3). Using the initial value of k, m and c calculated using equation (29), (30) and (32) respectively, the transfer function is

calculated using equation (11a) or (11b) whichever is applicable. Subroutine EXACT1 performs this computation.

Next the value of $\frac{F(t)}{m}$ (equation 33) is calculated using the discrete implicit form of the convolution integral equation (34)

$$O(t) = \int_{0}^{t} I(t-\tau) \cdot G(\tau) d\tau$$
 (33)

where

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$$I(t) = \frac{F(t)}{m}$$

Its implicit form is

$$I(i) = O(i)/\Delta T - \sum_{m=1}^{k=i-1} G(\tilde{m}) \cdot I (k + 2 - \tilde{m})$$
 (34)

where the initial value of i is 2, and
$$I(1) = \frac{O(1)}{I(1) \cdot \Delta T}$$
 (34a)

 $G(\tau)$ is the transfer function and ΔT is the time increment. Sub-routine NWCONV performs these calculations.

Next F(t), the equivalent input function, is calculated using the equation

$$F(t) = m \times I(t) = DF(I)$$
 (34a)

The maximum value CPMAX of F(t), is then found and compared with 25,000 lbs. If

$$ABERFG = |CPMAX - 25000| > 500$$
 (35)

then II is incremented by 1, and the procedure subsequent to that step is repeated using the DRV2(II) and DRV1(II) values corresponding to

the new II. ILP, JJ and II are counters which have maximum values of 40, 20 and IMAXC (Figure C-3) respectively. These counters ensure that the tested range of values for AAl is from .05 to 1.05, for AA3 it is from -.91 to -.01, and for II it is from the first point after the point of inflection to the maximum deflection (Figure C-3). If

$$ABERFG = |CPMAX-25000| < 500$$
 (35b)

then c is calculated using the equation for section (1), Figure 9, equation 36

$$CMAXV = (ZVALUF - EQKA* ZVALVO)/ZVALU1$$
 (36)

where CMAXV = c

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ZVALUF = force at section (1) (Figure 7)

EQKA = k

ZVALVO = the deflection at section (1) (Figure 7)

ZVALU1 = the velocity at section (1) (Figure 7)

The ratio given by equation (37) is next calculated

$$CDIFFR = (CMAXV - EQCA)/CMAXV$$
 (37)

where EQCA is the c value computed in equation (32). For the first value of CDIFFR, the procedure increments II by 1 and repeats all the steps from that point. For the second and subsequent values of CDIFFR, the ratio of the current and the preceding value is calculated using the equation

$$CRATIO = QC(ICC)/QC(ICC-1)$$
 (38)

where QC(ICC) refers to the current ratio of CDIFFR, AND QC(ICC - 1) refers to the preceding ratio of CDIFFR. If

CRATIO > 0

then the difference between the c values computed from two independent equations, equation (32) at the crossing point, and equation (36) at the point of inflection, has not passed through zero, so II is incremented and the subsequent steps are repeated. If

CRATIO < 0

then the difference between the c values calculated by equations (32) and (36) has passed through zero and the c values calculated using these two equations are approximately the same.

It has been found that the two controls, namely,

ABERFG < 500, and,

CRATIO < 0

ensures that the conditions stated in paragraph 3.2.2, are satisfied.

Finally, the exact solution for the force is calculated by substitution EQCA, EQKA, EQMA, DRVO, DRV1, and DRV2 in the equation,

 $CMPXX(I) = EQKA \times DRVO(I) \times EQMC \times DRV1(I) + EQMA \times DRV2(I)$ (39)

where CPMXX(I) is the force at each time increment I in the record.

The force at the crossing point calculated by the convolution and substitution technique are computed. A listing of the program FNDPRM and a typical output is given at the end of this Appendix.

c. Prediction

The subroutines EXACFT, SIGNTR, NWCONV, EXACT 1, ERECT2, and MODPRM are used by the main program PREDCT to predict the signature of a wehicle (Vehicle A) at a site (site I) over which the vehicle has not traversed. It is required to know the signatures of vehicle A at another site (standard site), and a standard vehicle (vehicle B) at both the standard site and site I. Since details of the methodology for determining the parameters of the transfer function, the loading function, and the equivalency function has been detailed previously only a summary flow chart of program PREDCT is given here. Figure C-5 shows the summary flow chart of program PREDCT. A listing of the program and a typical output is given in the following pages of this Appendix.

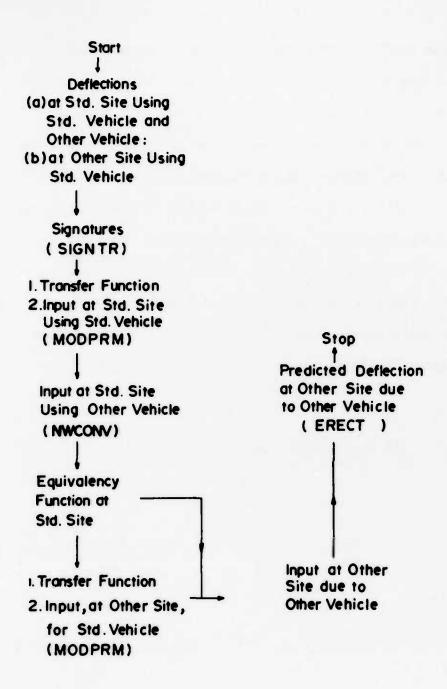


Figure C-5. Summary Flow Chart of Program PREDCT

```
C-
              PROGRAM FNDPRM(INPUT, OUTPUT, TAPE5=INPUT, TAPE6=OUTPUT, PLOT)
                         THIS PROGRAM COMPUTES PARAMETERS OF THE
                         REDUCED TRANSFER FUNCTION
FROM DEFLECTIONS MEASURED AT 5 POSITIONS
PERPENDICULAR TO THE DIRECTION OF TRAVEL
     Č
                         OF THE LOAD VEHICLE
                         SUBROUTINES REQUIRED
                                              (1)
                                                    EXACT1
     000000
                                              (2) NUCUNY
(3) EXACFIT
                                              (4) SIGNTR
                                      NR--NUMBER OF RECORDS

JM--NUMBER OF GAGES

AIN--DISTANCE OF GAGE FROM OUTER EDGE OF TIRE PR
NP-NUMBER OF DATA POINTS
                         INPUT-
     000000
                                       PT--TIME INCREMENT IN SECONDS WMAX--MAXIMUM WHEEL LOAD USED
                                       INITX--DIGITIZED DATA
     CCC
                                       CALG--CALIBRATION DATA
 2.
             DIMENSION FRESP(11,100,5), INITX(11,100,5), CALG(11,5), NP(11), PT(11)
            1, WMAX(11), DRV0(100), GMAX(11,5)
             DIMENSION VIPY(200);VISIG(200)
DIMENSION AI(5);FI(5)
DIMENSION AIN(5;11)
DIMENSION QC(2000)
 3.
 4.
 6.
7.
              DIMENSION APK(100), BT(100), TXPON(100)
 8.
             DIMENSION FR(10), DFY(10), FRR(10,6)
             DIMENSION DF(200)
DIMENSION T(200)
 9.
10.
             DIMENSION 0(200), GG(200)
11.
             DIMENSION V(200),G(200)
DIMENSION IVEL(11,5)
12.
13.
14.
             DIMENSION DRV1(200), DRV2(200), CPMXX(50)
15.
             READ(5,804)NR,JM
             DO 810 K=1,NR
READ(5,29)(AIN(J,K),J=1,JM)
16.
17.
18.
             READ(5,801)NP(K),PT(K),WMAX(K)
19.
              JP=NP(K)
             DO 800 I=1,JP
READ(5,802)(INITX(K,I,J),J=1,5)
20.
21.
22.
23.
        800 CONTINUE
             READ(5,803)(CALG(K,J),J=1,5)
        810 CONTINUE
24.
        801 FORMAT(15,2F10.4)
25.
        802 FORMAT(5(14,2X))
26.
        803 FORMAT(5F12.10)
804 FORMAT(2110)
29 FORMAT(6F10.5)
27.
28.
29.
     C************RECORDS RUN
     Сжжжжжжжжжжжж
                                      LOC B-4
                                                    LD. CT.
                                                                                3.0 .2
                                                                                              46
                        SITE 1
                                                                  3.10.76
           EGLIN
     Сжжжжжжжжжжж
     Сжжжжжжжжжжжж
                         TOP OF MAIN LOOP THAT OPERATES ON ONE RECORD AT A TIME
     C
30.
             DO 811 K=1,NR
                        CALCULATES DEFLECTION RESPONSE FUNCTION FOR JM GAGES AND THE PEAK FOR EACH GAGE FOR THE K TH RECORD
     C
     C
             TOT 806 . I=1 . . IM
31.
```

```
HILLY (TYPE)
             RMAX=0.0
              JP=NP(K)
             DU 807 I=2,JP
             FRESP(K, I-1, J)=FLOAT(INITX(K, I, J)-INITX(K, 1, J))*CALG(K, J)/12.
             VIPY(I-1)=FRESP(K,I-1,2)
IF(FRESP(K,I-1,J).LT.RMAX)GD TD 807
39.
40.
             RMAX=FRESP(K, I-1, J)
        807 CONTINUE
             GMAX(K, J)=RMAX
FI(J)=GMAX(K, J)
41.
        806 CONTINUE
                              LISTS REDUCED DATA
44.
45.
             WRITE(6,833)
        833 FORMAT(1H1,40X,*NEW RECORD*//)
             WRITE(6,814)
        814 FORMAT(//1X, **NUMBER OF DATA POINTS RECORD NUMBE

1EEL LOAD TIME INCREMENT NUMBER OF GAGES**/>
WRITE(6,815)NP(K),K,WMAX(K),PT(K),JM
                                                                                           MAX. WH
                                                                 RECORD NUMBER
48.
        815 FORMAT(10X, I4, 19X, I4, 18X, F6. 0, 19X, F4.2, 18X, I2/)
49.
50.
             WRITE(6,816)
        816 FORMAT (//40X, *DEFLECTIONS IN FT*/)
51.
        WRITE(6,817)
817 FORMAT(13X, **GAGE 1
52.
53.
                                                  GAGE 2
                                                                       GAGE 3
                                                                                            GAGE 4
                          GAGE 5*/>
             DD 818 I=1,JP
54.
             WRITE(6,819)(FRESP(K,I,J),J=1,JM)
55.
56.
        818 CONTINUE
57.
        819 FORMAT(10X,5(F7.5,10X))
58.
             WRITE(6,820)
59.
        820 FORMAT(//40X, *PEAK DEFLECTIONS*/)
             WRITE(6,819)(GMAX(K,J),J=1,JM)
                        FIT MEASURED PEAK LATERAL DEFLECTION PROFILE TO EQUATION USING SUBROUTINE EXACTIT
     C
     C
             CALL EXACFIT(FI, TFXP, TBETAF, FAF, AI)
61.
62.
             APK(K)=FAF
             BT(K)=TBETAF
63.
64.
             TXPUN(K)=TFXP
65.
             VBETA=TBETAF
66.
67.
             VRVAL=TEXP
             VIPX=AIN(2,K)
68.
             MV=JP
                        CALCULATE SIGNATURE OF PROTOTYPE USING
     C
     Č
                        SUBROUTINE SIGNTR
     C
69.
             CALL SIGNTR (VBETA, VRVAL, VIPX, VIPY, VISIG, NV)
                       CALCULATE FIRST AND SECOND DERIVATIVES
     C
             DO 74 I=1, JP
70.
71.
72.
73.
74.
75.
76.
77.
             DRVO(I)=VISIG(I)
             TRM1=(VISIG(I)+VISIG(I+1))/2.
             IF(I.EQ.1)TRM2=VISIG(I)/2.
             TRM2=(VISIG(I)+VISIG(I-1))/2.
             DRV1(I)=(TRM1-TRM2)/PT(K)
         74 CONTINUE
             DD 75 I=1, JP
TRN1=(DRV1(I+1)+DRV1(I))/2.
IF(I.EQ.1)TRN2=DRV1(I)/2.
TRN2=(DRV1(I)+DRV1(I-1))/2.
DRV2(I)=(TRN1-TRN2)/PT(K)
80.
         75 CONTINUE
                       IDENTIFY FIRST PEAK BY FINDING POINT WHERE FIRST DERIVATIVE CHANGES SIGN
```

```
83.
              BOTM=0.00000000001
              DG 726 I=1,JP
HUMP1=DRV0(I+1>-DRV0(I)
    84.
85.
              IF(HUMP1.GE. 0. 0)GO TO 726
    86.
    87.
              JPP=I+6
    88.
              GO TO 727
    89.
          726 CONTINUE
    90.
          727 CONTINUE
                      REPLACE ZEROES BY AVERAGING BEFORE AND AFTER POINTS
    91.
              IMAXV=0
    92.
              DY1MX=0.0
    93.
              ***************
    97.
              IF(ABS(DRV2(I)).LT.BOTM)DRV2(I)=(DRV2(I+1)+DRV2(I-1))/2.
    98.
              IF(DRV1(I).LT.DV1MX)GD TO 315
    99.
              DV1MX=DRV1(I)
   100.
              IMAXV=I
          315 CONTINUE
   101.
                      FIND MAXIMUM VALUE OF SIGNATURE
   102.
              DMAX=DRV0(1)
              DO 53 I=2,JPP
IF(DRVO(I).LT.DMAX)GO TO 53
   103.
   104.
   105.
              DMAX=DRV0(I)
   106.
              IMAXD=I
           53 CONTINUE
                      REPLACE
                               ERRONEOUS ACCELERATIONS BY AVERAGING
                      BEFORE AND AFTER POINTS
   108.
              IMAXV1=IMAXV-1
   109.
              DO 57 I=2, IMAXV1
   110.
              IF(DRV2(I).LE.0.0)DRV2(I)=(DRV2(I-1)+DRV2(I+1))/2.
              IF(DRV2(I).LE, 0.0)DRV2(I)=(DRV2(I-2)+DRV2(I+2))/2.
   111.
           57 CONTINUE
   112.
   113.
              IMAXVA=IMAXV+1
   114.
              DO 109 I=IMAXVA, JPP
   115.
              IF(DRV2(I).GE. 0.0)DRV2(I)=(DRV2(I-1)+DRV2(I+1))/2.
              IF(DRV2(I).GE. 0. 0)DRV2(I)=(DRV2(I-2)+DRV2(I+2))/2.
   116.
   117.
          109 CUNTINUE
   118.
              DO 107 I=1, JPP
107 CONTINUE
   121.
                      FIND TIME INTERVAL WHERE VELOCITY CHANGES SIGN
                      AND FRACTION OF VELOCITYTO ZERO BY LINEAR INTERPOLATION
        C
   122.
              DO 789 IA=1,JPP
   123.
              DERUTS=DRV1(IA+1)/DRV1(IA)
              IF(DEROTS.GT.0.0)GO TO 789
DZERO=DRV1(IA)-DRV1(IA+1)
   124.
   125.
              DDTZ=DRV1(IA)./DZERO
   126.
   127.
              NAFNL=IA
   128.
              GO TO 788
   129.
          789 CONTINUE
          798 CONTINUE
   130.
                      FIND INTERVAL AND TIME WHERE ACCELERATION IS ZERO AND THE CORRESPONDING DISPLACEMENT
                      AND VELOCITY
   131.
              DO 889 IA=1,JPP
              ZEROTS=DRV2(IA+1)/DRV2(IA)
   132.
```

```
IF(ZERDIS.GT.0.0)GU TO 889
TZERO=DRY2(IA)-DRY2(IA+1)
FDTZ=DRY2(IA)/TZERO
134.
135.
136.
137.
              IAFNL=IA
              DRORNG=DRVO(IA+1)-DRVO(IA)
138.
              ZVALU0=DRY0(IA)+FDTZ*DRORNG
139.
              DR1RNG=DRV1(IA+1)-DRV1(IA)
              ZVALU1=DRV1(IA)+FDTZ*DR1RNG
140.
141.
              GD TD 888
         889 CONTINUE
142.
         888 CONTINUE
143.
      Сжжжжжжжж
144.
145.
              WRITE(6,99)
          99 FORMAT(/37X, *FOR SIGNATURE*/)
              WRITE(6,76)
146.
147.
          76 FORMAT(//7X,*
                                     INCREMENT
                                                    SIGNATURE
                                                                           VELUCITY
                                                                                            ACCEL
            1ERATION*/)
148.
              WRITE(6,77)(I,DRV0(I),DRV1(I),DRV2(I),I=1,JPP)
          77 FORMAT(7X, 114, 3E15.5)
149.
150.
              WRITE(6,1091)
151.
       IMAXF=0
153.
154.
155.
              PMAX=WMAX(K)
              AA1=0.05
              ICC=0
                        TOP OF LOOP INCREMENTING AA1
156.
157.
              DO 104 ILP=1,40
EQKA=AA1*PMAX/DMAX
158.
              AA3=-0.91
      C
                        TOP OF LOOP INCREMENTING AA3
      C-
159.
              DO 103 JJ=1,20
              IT=IMAXV
160.
161.
              EQMA=AA3*PMAX/DRY2(IMAXD)
162.
              CPMAX=0.0
163.
164.
165.
              AA2=AA1+AA3
IF(AA2.LT.0.0)GD TO 1021
IMAXC=IMAXD
166.
              IT=IT+1
                        TOP OF LOOP VARYING TIME BETWEEN ZERO ACCELERATION AND ZERO VELOCITY AND CALCULATING EQCAFOR EACH INCREMENT
      CCC
             DO 102 II=IT, IMAXC
EQCA=EQMA*DRV2(II)/DRV1(II)
167.
168.
169.
              EQCA =- EQCA
170.
             EQVK=EQKA
171.
             EQMASS=EQMA
172.
173.
             EQVC=EQCA
             MM=JPP
174.
             DT=PT(K)
                        CALCULATING REDUCED TRANSFER FUNCTION USING
                        SUBROUTINE EXACT1
175.
              CALL EXACTI (MM, DT, EQVK, EQMASS, T, GG, EQVC)
176.
177.
             DO 73 I=1, JPP
O(I)=DRV0(I)
178.
179.
             V(I)=GG(I)
          73 CONTINUE
180.
             N=JPP
      C-
                        CALCULATE I(T) USING SUBROUTINE NUCONY
BY IMPLICIT CONVOLUTION
      C
      C
```

```
181.
              CALL NUCONV(N, V, G, D, DT)
                         CALCULATING EQUIVALENT FORCEAS PRODUCT OF
                         I(T) AND EQMASS
182.
               DO 508 MP=1,JPP
183.
               DF (MP)=G(MP) MEQMASS
184.
185.
         508 CONTINUE
              DO 509 MA=1, JPP
DF(MA)=(DF(MA)+DF(MA+1))/2.
186.
187.
         509 CONTINUE
                         AVERAGING AND FINDING MAXIMUM EQUIVALENT FORCE
              CPMAX=0.
188.
189.
              DO 501 M=1,JPP
190.
               DF(M)=(DF(M)+DF(M+1))/2.
191.
          IF(DF(M).GT.CPMAX)GO TO 20
GO TO 502
20 CPMAX=DF(M)
193.
194.
195.
               ICF=M
         502 CONTINUE
501 CONTINUE
196.
                         EXECUTING FIRST CHECK TO ENSURE THAT THE ABSOLUTE DIFFERENCE BETHEEN THE MAXIMUM COMPUTED
                         FORCE AND 25 KIPS IS LESS THAN 500 LBS
197.
198.
199.
              ERFCE=CPMAX-PMAX
               ABERFG=ABS(ERFCE)
               IF(ABERFG.GT.500.0)GB TO 102
                         EXECUTING SECOND CHECK TO ENSURE THAT THE DIFFERENCE BETHEEN THE C VALUES COMPUTED AT THE CROSSING POINT AND THE POINT OF INFLECTION GOES THROUGH ZERO
200.
               ICC=ICC+1
              DFRNG=DF(IAFNL+1)-DF(IAFNL)
ZVALUF=DF(IAFNL)+FDTZ#DFRNG
CMAXY=(ZYALUF-EGKA#ZYALU0)/ZVALU1
CDIFFR=(CMAXY-EQCA)/CMAXY
201.
202.
203.
204.
205.
               ABCD=CDIFFR
206.
               QC(ICC)=CDIFFR
207.
              IF (ICC.EQ.1)GO TO 102
CRATIO=QC(ICC)/QC(ICC-1)
209.
               IF(CRATID.GT.0.0)GD TO 102
      CC
                         CALCULATING FORCE RATIOS COMPONENTS OF FORCE
                         AND TOTAL FORCE AT THE CROSSING. POINT
210.
              FCHK=EQKA*DRYO(II)
211.
              DIFCHK=FCHK-DF(II)
212.
               DFCHKR=DIFCHK/FCHK
213.
              FPFRAC=DF(II)/PMAX
214.
               ICDPT=II
               TRMM=EQMA*DRV2(II)
              TRMC=EQCA*DRV1(II)
TRMK=EQKA*DRV0(II)
216.
217.
              CPMXX(II)=TRMM+TRMC+TRMK
218.
219.
              WRITE(6,847)
         847 FORMAT(//10X, #GOOD RUN#//)
220.
      GO TO 946
221.
       102 CONTINUE
222.
223.
              AA3=AA3+0.045
224.
      Симимимимимими
225.
        103 CONTINUE
226.
              AA1=AA1+0.025
      Сжижжийж
        104 CONTINUE
1000 FORMAT(/1X,#000#)
227.
228.
         845 FURMAT(6F15.4,10X,215,F10.3)
229.
              WRITE(6,1002)
230.
```

```
231.
232.
233,
234.
       1002 FORMAT(/10X, *DUTER LOOP COMPLETED*/)
        GO TO 94
846 CONTINUE
             WRITE(6,1043)
235.
       1043 FORMAT<///////////0/10X;#INCREMENT#;2X;#M-COMP.#;9X;#C-COMP.#;9X;#K-C
            1DMP. *,8X; *EXACT*;8X; *CDNVDLUTIDN*;8X; *TDT FUNC*;8X; *SIGNATURE*/>
                      CALCULATING COMPONENTS OF FORCE DUE TO INERTIAL
                      DAMPING AND STIFFNESS
236.
             JPN=JPP-3
237.
            DD 106 I=1;JPN
TMT=EQMA*DRV2(I)
TCT=EQCA*DRV1(I)
238.
240.
             TKT=EQKA*DRYO(I)
241.
             CPMXX(I)=TCT+TKT+TMT
242.
             WRITE(6,1001)I, TMT, TCT, TKT, CPMXX(I), DF(I), GG(I), DRYO(I)
        106 CONTINUE
243.
                      CALCULATING FORCE RATIOS AT SECTIONS (1), (2), (3)
            AFRNG=DF(NAFNL+1)-DF(NAFNL)
DVALUF=DF(NAFNL)+DDTZ*AFRNG
244.
245.
246.
            ZVALUR=ZVALUF/PMAX
      Сжжжжжж
            DVALZD=DVALUF/PMAX
247.
248.
            FDRTIU=ABERFG/PMAX
                      CALCULATING TIME RATIOS AT SECTIONS(1), (2), (3)
     C.
249.
250.
251.
             DENOM=FLOAT (IMAXD)
            DENOM1=1.0
TCRSPT=FLOAT(II)/DENOM
252.
             TZRDA=FLOAT(IAFNL)/DENOM
      Сжжжжжжж
253.
             WRITE(6, 1041)
       1041 FORMAT(///////10x, *EQUIV. MASS*, 10x, *EQUIV. STIFFNESS*, 9X, *EQUI
254.
           14. DAMPING*>
255.
256.
            WRITE(6,2041)
       2041 FORMAT(12X, *SLUGS*, 16X, *LBS/FT*, 20X, *LBS-SEC/FT*/)
257.
258.
259.
             WRITE(6,1042)EQMA, EQKA, EQCA
       1042 FORMAT(10X,F10.2,15X,F10.2,12X,F10.1)
             WRITE(6,1044)
       1044 FORMAT(//20X; *
                                  FORCE RATIOS(FORCE DVER 25000)*/>
260.
261.
             WRITE(6, 1045)
       1045 FORMAT(25X, *SECTION(1) SECTION(2) SECTION(3)*/)
262.
             WRITE(6,1046)ZVALUR, FPFRAC, DVALZD
263.
264.
265.
       1046 FORMAT(23X,3F10,3)
             WRITE(6,1050)
266.
       1050 FORMAT(///20X;*
                                     TIME RATIOS(TIME OVER TIME TO MAX. DEF.) >>>
267.
            DENOM1=1.
268.
            WRITE(6,1045)
WRITE(6,1046)TZROA,TCRSPT,DENOM1
269.
270.
             WRITE(6,1060)
271.
       1060 FORMAT(///20X, *C-PARAMETER DIFFERENCE RATIO*/>
272.
273.
274.
            WRITE(6, 1061)CDIFFR
       1061 FURMAT(30X, F7.3)
             WRITE(6,1070)
275.
       1070 FORMAT(///20X; *FORCE DIFFERENCE RATIO AT THE CROSSING POINT*/>
276.
             WRITE(6,1061)DFCHKR
277.
             WRITE(6, 1080)
278.
279.
       1080 FORMAT(///20X, *MAXIMUM FORCE DIFFERENCE*/)
            WRITE(6:1061)ABERFG
       1001 FORMAT(10X, 15, 7F15.5)
280.
281.
        811 CONTINUE
282.
         94 CONTINUE
283.
             STOP
284.
             END
```

```
SUBROUTINE EXACFIT(FI, TFXP, TBETAF, FAF, AI)
                            THIS SUBROUTINE CALCULATES THE PARAMETERS OF THE MAXIMUM LATERAL DEFLECTION PROFILE USING
                             THE DEVELOPED INTERPOLATION-ITERATION PROCEDURE
                             INPUT-
                                         FI--MEASURED DEFLECTIONS AT LOCATIONS LATERAL
                                         TO THE TIRE PRINT
AI-LATERAL DISTANCES FROM EDGE OF TIRE PRINT
                                               CORRESPONDING TO FI
                                          TFXP--R PARAMETER
                            DUTPUT
                                         TBETAF-BETA PARAMETER FAF-A PARAMETER
                  DIMENSION FI(6), TBETA(3), TVAL(999), AI(5), FA(5), FFXP(5) TRACE SUBSCRIPTS
TRACE ARITHMETIC
      2.
          C$
          C$
      з.
                  M=1
      4. 5. 6. 7.
                  IFTX=0
                  I=2
                  IG=I+1
                  S+I=MI
      8.
                  ITRID=0
          Сжижижижижижижижижижижижи
          Сжижижижижижижижижижижижи
      9.
            710 CONTINUE
     10.
                  FXP=0.2001
                  DFXP=0.0125
     11.
     12.
                  0 = 1
            771 CONTINUE
     13.
     14.
                  J=J+1
     15.
                  FXP=FXP+DFXP
     16.
17.
                  IFCHK=0
            772 CONTINUE
776 CONTINUE
     18.
     19.
20.
                  CONS1=ALOG(FI(I))-ALOG(FI(IG))
                  CONS2=ALOG(FI(I))-ALOG(FI(IM))
     21.
                  CONF=CONS1/CONS2
     22.
                  FLHS=CONF*((AI(I)**FXP)-(AI(IM)**FXP))
FRHS=(AI(I)**FXP)-(AI(IG)**FXP)
     24.
                  TVAL(J)=FLHS-FRHS
                 IF(J.EQ.1)GO TO 771
FDIF=TVAL(J)/TVAL(J-1)
ABVAL=ABS(TVAL(J))
IF(J.EQ.999)GO TO 726
     25.
     26.
     27.
                  IF(FDIF.GT.0.0)GD TD 771
TFDIF=ABS(TVAL(J-1))+ABS(TVAL(J))
     29.
     30.
                  FFXP(M)=FXP-(TVAL(J)/TFDIF)*DFXP
            773 CONTINUE
     33.
                  FBETA1=ALOG(FI(I))-ALOG(FI(IG))
     34.
35.
                  FBETA2=(AI(I)**FFXP(M))-(AI(IG)**FFXP(M))
                  TBETA(M)=FBETA1/FBETA2
                 FA1=TBETA(M)*(AI(1)**FFXP(M))
FA(M)=FI(1)/EXP(FA1)
     36.
37.
                  GO TO 777
IF(ITRID.EQ.1)GO TO 727
     38.
     39.
AUTION - STATEMENT CANNOT BE EXECUTED - NO STATEMENT NUMBER 40. IF(IFTX.EQ.1)GO TO 777
                                                                                         *************
     41.
                 M=M+1
     42.
                  IG=IG+1
     43.
                  IM=IM+2
            IFTX=1
GD TD 710
777 FAF=FA(M)
     44.
     45.
     46.
     47.
                  TBETAF=TBETA(M)
                  TFXP=FFXP(M)
     48.
                 GO TO 728
          Сжижжижжжжжжжжжжжжжжжж
          Сжжжжжжжжж
     50.
            726 CONTINUE
```

```
ITRID=1
51.
52.
53.
54.
55.
56.
57.
58.
60.
            M=M+1
            IG=IG+1
            IM=IM+2
            IFTX=1
            IF(IM.LE.5)GO TO 730
            WRITE(6,731)
       731 FORMAT(/20X, *DATA DOES NOT FIT--CHECK DATA*/)
       GO TO 720
730 GO TO 710
61.
       727 CONTINUE
    FAF=FA(2)
63.
64.
65.
66.
67.
            TBETAF=TBETA(2)
            TFXP=FFXP(2)
       728 CONTINUE
            WRITE(6,775)
       775 FORMAT(/20X; * A PARAMETER
                                                  BETA
                                                                   EXPONENT R*/)
       WRITE(6,778)FAF, TBETAF, TFXP
778 FORMAT(20X,3E15.5/)
68.
69.
70.
71.
72.
73.
74.
            WRITE(6,750)
       750 FORMAT ( /20X , * CALCULATED Y " MEASURED Y
                                                                    LATERAL DIST. */>
            DO 701 M=1,5
            YY1=TBETAF*(AI(M)**TFXP)
            YY2=EXP(YY1)
75.
76.
77.
            YY=FAF*YY2
            WRITE(6,20)YY,FI(M),AI(M)
       701 CONTINUE
78.
       720 CONTINUE
        20 FORMAT(20X, 3E15.5)
80.
            RETURN
81.
            END
            SUBROUTINE SIGNTR(VBETA, VRVAL, VIPX, VIPY, VISIG, NV)
 1.
                     THIS SUBROUTINE CALCULATES THE SIGNATURE BY
                     DIRECT SUBSTITUTION
                     INPUT-
                                  VBETA--BETA PARAMETER
                                 VIPX--LATERAL DISTANCE OF GAGE FROM OUTER
EDGE OF TIRE PRINT
VIPY--NV VALUES OF MEASURED DEFLECTION
AT VIPX FROM OUTER EDGE OF TIRE PRINT
                                  VRVAL--R PARAMETER
    CC
                     DUTPUT-
                                 VISIG--NV VALUES OF THE SIGNATURE
            DIMENSION VIPY(200), VISIG(200)
            YTOP=VBETA*(VIPX**VRVAL)
 4.5.6.7.
            DO 40 I=1, NV
            VISIG(I)=VIPY(I)/EXP(VTOP)
        40 CONTINUE
            RETURN
 8.
            END
            SUBROUTINE NUCONV(N, V, G, D, DT)
 1.
                     THIS SUBROUTINE IMPLICITLY CONVOLUTES V( ) AND O( )TO
                     YIELD G( ) AT N POINTS
                                 V--N VALUES OF INPUT
                     INPUT-
                                 DT--TIME INCREMENT
                     DUTPUT-
                                 G--N VALUES OF THE IMPLICIT CONVOLUTION RESULT
```

```
DIMENSION V(200), G(200), D(200)
 2345678
            S=0.
Q=0.0
            G(1)=O(1)/(Y(1)*DT)
DO 984 I=2,N
            K=I-1
            DO 985 M=1.K
 9.
            Q=Q+V(K+2-M)*G(M)
10.
       985 CONTINUE
11.
            2=0
            G(I)=((D(I)/DT)-S)/V(1)
12.
13.
            Q=0.0
14.
15.
            S=0.0
CONTINUE
16.
17.
            RETURN
            END
 1.
            SUBROUTINE EXACTI (MM, DT, EQVK, EQMASS, T, GG, EQVC)
                      THIS SUBROUTINE CALCULATES THE REDUCED TRANSFER FUNCTION
                      BY DIRECT SUBSTITUTION
     00000000
                                  EQVK--EQUIVALENT K PARAMETER
EQMASS--EQUIVALENT MASS
EQVC--EQUIVALENT C PARAMETER
                      INPUT-
                                   DT--TIME INCREMENT
                                  MM--NUMBER OF DATA POINTS
                      DUTPUT-
                                  GG--MM VALUES OF THE TIME DEPENDENT TRANSFER FUN
 2.
            DIMENSION T(200), GG(200)
            TRACE ARITHMETIC
 з.
            DD 630 I=1,MM
            T(1)=FLOAT(1)*DT
 5.
            TIME=T(I)
     C
            BB=SQRT(EQVK/EQMASS)
            AA=EQYC/(2. *EQMASS)
 8.
9.
            TRI=BB*BB-AA*AA
            IF(TRI.LT. 0. 0)GD TD 3
    С
10.
            GE1=1./SQRT(TRI)
            GE2=1./EXP(AA*T(I))
G3=SIN(SQRT(TRI)*T(I))
11.
12.
13.
            GG(I)=GE1*GE2*G3
     C
14.
            GD TD 630
    C
15.
16.
          3 CONTINUE
            PP=SQRT(ABS(TRI))
P1=EXP(PP*T(I))
17.
            P2=1./(EXP(PP*T(I)))
18.
19.
            P3=1./EXP(AA*T(I))
20.
            P5=(P1-P2)/2.
            GG(I)=P3*P5/PP
21.
22.
       630 CONTINUE
23.
            RETURN
            END
```

NEW RECORD

NUMBER OF GAGES																•												
TIME INCREMENT	.20		GAGE 5	00000	.00000	. 00001	00005	£0000.	. 00004 40000	90000	60000.	. 00012	.00014	200017	.00018	. 00020	00000	. 00021	. 00021	. 00000	. 00019	.00015	.00010	20000	00003	. 00001	- 00001	- 00002
WHEEL LOAD TI	25000.	IN FT	GAGE 4	.00001	. 00001	. 00003	. 00005	80000	.00010	.00015	. 00019	00005	. 00033	. 00044	000031	00026	. 00058	. 00062	. 00062	. 00059	. 00056	00043	. 00033	00000	00014	.00010	00000	.00002
MAX.		DEFLECTIONS I	GAGE 3	.00001	. 00003	. 00005	60000	. 00011	. 00018	. 00028	. 00037	09000	. 00086	26000	. 00112	. 00140	. 00150	. 00166	. 00166	. 00153	. 00145	. 00111	06000	5000	. 00047	. 00038	. 00026	.00021
RECORD NUMBER			GAGE 2	00000	. 00003	. 00006		.00019	. 00023	. 00040	. 00051	28000	. 00108	. 00156	. 00189	.00261	. 00299	. 00343	. 00349	.00310	.00290	.00218	. 00178	00197	. 00101	68000.	.00065	. 00057
NUMBER OF DATA POINTS	\$		GAGE 1	.00001	.00004	. 00008	. 00013	.00017	000030	. 00048	. 00063	00108	. 00134	. 00195	. 00238	. 00341	. 00384	. 00434	. 00436	. 00386	. 00344	. 00257	00210	. 00175	. 00122	. 00106	.00081	. 00073 . 00067

000000000000000000000000000000000000000		. 00021				
	SHDI	. 00063	EXPONENT R	1.31790E+00	LATERAL DIST.	2.50000E-01 5.83330E-01 1.50000E+00 2.50000E+00 3.50000E+00
	PEAK DEFLECTIONS	, 00168	ВЕТА	-6.02783E-01	MEASURED Y	4.35510E-03 3.49750E-03 1.68158E-03 6.30420E-04 2.12980E-04
		09200	A PARAMETER	4.79864E-03	CALCULATED Y	4.35510E-03 3.56827E-03 1.71561E-03 6.38773E-04 2.07310E-04
.00063 .00059 .00059 .00058 .00056		. 00436				

ACCELERATION	2.93968E-05 9.79888E-05 1.95988E-05 1.95978E-04 2.93966E-04 2.93966E-04 7.83931E-04 7.83911E-03 1.07788E-03 1.07788E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17788E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17587E-03 1.17788E-03 1.17788E-03 1.17788E-03	
VELOCITY	7.83911E-05 7.83911E-05 7.83911E-05 1.17587E-04 1.95978E-04 3.13548E-04 3.13548E-04 7.0578E-04 7.0578E-04 1.21508E-04 1.821508E-03 1.60708E-03 1.68541E-03 2.07736E-03 1.68541E-03 2.3736E-03 2.3736E-03 1.68541E-03 2.3736E-03 2.3736E-03 2.3736E-03 2.3736E-03 1.68541E-03 2.3736E-03	
SIGNATURE	0.3.13564E-055 6.827128E-055 1.41104E-04 1.81391E-04 2.150851E-04 3.13564E-04 4.07633E-04 5.33059E-04 5.33059E-04 1.17387E-03 1.77164E-03 1.77164E-03 2.53987E-03 3.93183E-03 4.60939E-03 3.93183E-03 3.93183E-03 3.93183E-03 3.93183E-03 3.93183E-03 3.93183E-03 3.93183E-03 3.93183E-03 3.93586E-03 4.17046E-03	
INCREMENT	ก๛ ุล ณ _ิ กาล อ _ั นามามามามามามามามามามามามามามามามามามาม	331 35 35 35

NCREMENT	M-COMP.	C-COMP.	K-COMP.	EXACT	CONVOLUTION	TOT FUNC	SIGNATURE
444	72,36842		0	514.61988	617,71209	04052	6
a	24, 12281		95, 83333	783, 33333	675.91968	03672	E0 000.
· (*)	24.12281		143,75000	610.12427	618,69802	03287	.00003
4	36.18421		191.66667	670, 10234	788,48276	02941	90000
IO.	48.24561		239, 58333	951.20614	1087,28563	02632	90000
9	+8.84561		335,41667	1268,16520	1409.61497	02355	.00011
~	72.36842	1105.62865	431.25000	1609.24708	1875, 76839	02108	.00014
100	72,36842		573, 66 vol	2495, 24854	2377,82005	.01886	.00019
0	72,36842		766.66667	2608, 04094	2916,98704	01688	00005
0	144.73684		958, 33333	3314,32749	3850,51099	01511	. 00031
_	192,98246		1245, 83333	4534,57602	5092, 77541	01352	. 00041
a	265,35088		1629,16667	5874,78070	6788.82147	.01210	. 00053
	313,59649		2108,33333	7950.07310	8855.46687	. 01083	69000
14	217,10526		2827, 08333	9899, 08626	10604.18900	69600	. 00093
	168,85965		3593,75000	11280.88450	12230.96440	29800	.00118
	168.85965		4456.25000	13027.88743	13946.77641	92200	.00146
	265,35088		5414.58333	14746. 08918	16225.51305	00694	. 00177
	289,47368		6420,83333	17545,46784	18754.55138	00621	.00210
	313, 59649		7762.50000	19795.76023	21549.91447	00556	. 00254
	265,35088		8960.41667	22935, 56287	24266.73247	00498	. 00293
	-217.10526		10733,33333	24668.27485	24740,46330	. 00445	. 00351
	-506.57895		12027, 08333	23240.16813	23698.22351	00399	. 00394
	-747.80702		13272,91667	22033.51608	21219.97611	00357	.00434
	-892, 54386		14087.50000	18059.72222	17655.89134	.00319	. 00461
•	1326, 75439	-	14327, 08333	14327. 08333	11497.90363	00286	. 00469
1	1375,00000		14375,00000	5702.85n88	4443.21485	00256	.00420
_	-506.57895		12745.83333	961.84211	972,66864	00229	. 00417
œ			11931.25000	-274.89035	-1610.69484	00202	06200
σ.	-120,61404	-13709.79532	10158,33333	-3672, 07602	-3627,83553	00183	. 00332

EQUIV. DAMPING LBS-SEC/FT EQUIV. STIFFNESS LBS/FT 3056258.49 EQUIV. MASS SLUGS 246179.17

5641606.1

FDRCE RATIDS/FDRCE DVER 25000)

SECTION(1) SECTION(2) SECTION(3)

.417 .460 .981

TIME RATIOS(TIME OVER TIME TO MAX. DEF.) SECTION(1) SECTION(2) SECTION(3)

1.000 . 962 692.

C-PARAMETER DIFFERENCE RATIO

FORCE DIFFERENCE RATIO AT THE CROSSING POINT

MAXIMUM FORCE DIFFERENCE

259.537

```
PROGRAM PREDCT (INPUT: DUTPUT: TAPE5=INPUT: TAPE6=DUTPUT: PLDT)
  1.
                    THIS PROGRAM PREDICTS THE SIGNATURE AT AN UNTESTED SITE DUE TO VEHICLE A, IF THE SIGNATURE OF A STANDARD VEHICLE IS KNOWN AT THE UNTESTED SITE, AND THE SIGNATURE OF BOTH THE STANDARD VEHICLE AND VEHICLE A ARE KNOWN AT A
      CCCC
                    STANDARD SITE
                    STANDARD VEHICLE AND SITE
                    C 131
                                     SITE 1
                                                                   SIGNATURE
      Ċ
                    C 131
                                    SITE 3
                                                                   BOYER
                                                                                  SIGNATURE
                                             47 BOYER
                    C-135 SITE 1
                                                                   SIGNATURE
                    INCLUDED DNLY FOR PURPOSES OF THIS EXAMPLE-THIS SIGNATURE
                     IS THE PREDICTED SIGNATURE
                           SITE 3
                                                           SIGNATURE
                                              BOYER
                                                           SIGNATURE DUE TO STANDARD VEHICLE
                    INPUT-
                                           SCR()
                                                           AT STANDARD SITE
                                                          SIGNATURE DUE TO STANDARD VEHICLE
AT OTHER SITE
SIGNATURE DUE TO OTHER VEHICLE
      CCCCC
                                           XC >
                                           DFCE( )
                                                           AT STANDARD SITE
                                           COMPY() SIGNATURE DUE TO OTHER VEHICLE
AT OTHER SITE(FOR PURPOSES OF
THIS EXAMPLE ONLY)
ALL INPUT SIGNATURES ARE IN INCHES
DO() PREDICTED SIGNATURE IN FT.
      CCC
                    DUTPUT
      CCC
                    SUBROUTINES REQUIRED
                                                           (1)
                                                                  EXACT1 (
                                                                  HUCDHY
                                                           (2)
                                                                  ERECT2(
                                                           (3)
                                                           (4)
                                                                  MODPRM
                DIMENSION SOUTI(200), SOUTE(200), YOUTI(200)
  3.
               DIMENSION ETINP(200), EQFNC(200)
               DIMENSION ET(200)
DIMENSION VISIG(200)
 4. 5. 6. 7. 8. 9.
               DIMENSION XT(6,200),XF(6,200),DTRK1(200),DF4(200),SFNC(200)
               DIMENSION PP(800),XG(800),DD(800)
DIMENSION DRVO(100),DRV1(100),DRV2(100)
DIMENSION DF(200),X(100),DFCE(100),DDEF(100)
DIMENSION SCR(100),T(200),V(200),D(200),G(200),GG(200)
10.
               DIMENSION COMPY(200), VOUT2(200)
                TRACE ARITHMETIC
           50 FORMAT(//60X,20H-
12.
         498 FORMAT(29X,54HEQUIV INPUT IN LBS
                                                                        TOT FUNCTION
                                                                                                   RESPONSE IN
              1FT//)
         398 FORMAT(31X,E15.5,5X,E15.5,2X,E15.5)
14.
      C
15.
16.
17.
18.
               READ(5,100)(SCR(I), I=1,47)
               READ(5,100)(X(I),I=1,47)
READ(5,100)(DFCE(I),I=1,47)
READ(5,100)(CDMPV(I),I=1,47)
         100 FORMAT(7F10.6)
20.
21.
22.
23.
               DO 496 I=1,47
SOUT1(I)=SCR(I)/12.
                VOUT1(1)=DFCE(1)/12.
                SOUT2(1)=X(1)/12.
                VDUT2(I)=COMPV(I)/12.
         496 CONTINUE
     CC
```

```
C
26.
27.
28.
            JP=47
            PMAX =20000.
            DT=0.04
29.
            MM=47
            LL=47
            NT=47
31.
    C
           DO 74 I=1,NT
VISIG(I)=SOUT1(I)
32.
33.
        74 CONTINUE
           WRITE(6,497)
       497 FORMATC//2X, 23HSITE ONE
                                           STD VEHICLE//)
36.
37.
            CALL MODPRM(PMAX, JP, DT, VISIG, EQMASS, DF, GG)
               COMPUTE EQUIV.
                                   INPUT FOR VEHICLE 2
                                                               SITE 1
     C
           DO 721 I=1,MM
T(I)=FLOAT(I)*DT
38.
39.
           D(I)=VDUT1(I)
40.
            V(I)=GG(I)
42.
       721 CONTINUE
43.
           N=47
    C
           CALL NUCONY(N, V, G, D, DT)
44.
45.
           DO 722 I=1,MM
ETINP(I)=G(I)*EQMASS
46.
       722 CONTINUE
    C
48.
            WRITE(6,50)
49.
            WRITE(6, 495)
50.
       495 FORMAT(//2X, 21HSITE ONE
                                           VEHICLE 2//)
51.
52.
           WRITE(6,498)
           WRITE(6,398)(ETINP(I),GG(I),U(I),I=1,MM)
    C
                 COMPUTE EQUIVALENCY
                                           FUNCTION
           DO 723 I=1,MM
EQFNC(I)=ETINP(I)/DF(I)
53.
       723 CONTINUE
               COMPUTE TOT FUNCTION , EQUIV. INPUT FOR NEXTSITE STD. VEHICLE
    C
           WRITE(6,397)
       397 FORMAT(//2X, 23HNEXT SITE
                                           STD VEHICLE//>
           DO 724 I=1,MM
VISIG(I)=SOUT2(I)
58.
59.
      724 CONTINUE
60.
61.
           CALL MODPRM(PMAX, JP, DT, VISIG, EQMASS, DF, GG)
    č
               COMPUTE EQUIV. INPUT NEXT SITE
                                                      VEHICLE 2
62.
           DO 725 I=1,MM
63.
64.
           ET(I)=DF(I)*EQFNC(I)
       725 CONTINUE
               COMPUTE OUTPUT FOR NEXT SITE VEHICLE 2
65.
66.
67.
           DO 206 I=1,MM
           PP(I)=ET(I)/EQMASS
           XG(I)=GG(I)
       206 CONTINUE
```

```
69.
             CALL ERECT2(MM, PP, XG, DT, DD)
70.
71.
72.
73.
             WRITE(6,50)
             WRITE(6,494)
        494 FORMAT (//2X, 21HNEXT SITE
                                                VEHICLE 2//)
             WRITE(6, 498)
             WRITE(6,398)(ET(I),XG(I),DD(I),I=1,MM)
             WRITE(6,50)
WRITE(6,2000)
      2000 FORMAT(//60X, *PREDICTED*, 10X, *MEASURED*//>
78.
79.
      WRITE(6,1000)(DD(I),VDUT2(I),I=1,47)
1000 FORMAT(52X,2F15.5)
     C
80.
             WRITE(6,50)
     CCC
81.
             STOP
82.
             END
     C
             SUBROUTINE NUCONV(N, V, G, O, DT)
DIMENSION V(200), G(200), D(200)
 2.
             S=0.
 4.
             Q=0.0
 5.
             G(1)=O(1)/(V(1)*DT)
             DO 984 I=2,N
             K=I-1
            DD 985 M=1,K
Q=Q+V(K+2-M)*G(M)
 8.
 9.
       985 CONTINUE
10.
12.
             G(I)=((D(I)/DT)-S)/V(1)
     C
13.
             Q=0.0
14.
15.
16.
17.
             S=0.0
       984 CONTINUE
            RETURN
END
     C
             SUBROUTINE ERECTE (MM, PP, XG, BT, DD)
             DIMENSION PP(800), XG(800), 00(800)
            PS2=0.0
 з.
            DO 331 I=2,MM
K=I-1
5.6.7.8.9.
            DD 332 M=1,K
PS2=PP(K+1-M)*XG(M)+PS2
       332 CONTINUE
            00(I-1)=PS2*DT
10.
            PS2=0.0
       331 CONTINUE
11.
            RETURN
12.
13.
            END
            SUBROUTINE EXACT (MM,DT,EQVK,EQMASS,T,GG,EQVC)
DIMENSION T(200)+GG(200)
2. •
```

```
COMPUTING EXACT TRANSFER FUNCTION
 3.
            DO 630 I=1,MM
            T(I)=FLOAT(I)*DT
 4.
 5.
            TIME=T(I)
            BB=SQRT(EQVK/EQMASS)
            AA=EQVC/(2. *EQMASS)
 7.
     C
 8.
            TRI=BB#BB-AA#AA
            IF(TRI.LT.0.0)GO TO 3
 9.
10.
            GE1=1./SQRT(TRI)
            GE2=1./EXP(AA*T(I))
11.
            G3=SIN(SQRT(TRI)*T(I))
12.
            GG(I)=GE1#GE2#G3
13.
     C
            GO TO 630
14.
    C
15.
          3 CONTINUE
16.
            PP=SQRT(ABS(TRI))
            P1=EXP(PP*T(I))
17.
            P2=1./(EXP(PP#T(I)))
18.
            P3=1./EXP(AA*T(I))
19.
20.
            P5=(P1-P2)/2.
            GG(1)=P3*P5/PP
21.
       630 CONTINUE
22.
     С
23.
            RETURN
            END
24.
     C
            SUBROUTINE MODPRM(PMAX, JP, DT, VISIG, EQMASS, DF, GG)
 2.
            DIMENSION QC(100)
 3.
            DIMENSION DF(200)
            DIMENSION T(200)
DIMENSION U(200), GG(200)
DIMENSION V(200), G(200)
DIMENSION IVEL(11,5)
 4.
 5.
 8.
            DIMENSION DRY1(200), DRY2(200), CPMXX(50), YISIG(200), DRY0(200)
            DO 74 I=1, JP
DRVO(I)=VISIG(I)
 9.
10.
            TRM1=(VISIG(I)+VISIG(I+1))/2.
11.
            IF(I.GT.1)GO TO 744
TRM2=VISIG(I)/2.
12.
13.
14.
15.
16.
17.
18.
       GO TO 474
744 CONTINUE
            TRM2=(VISIG(I)+VISIG(I-1))/2.
       474 CONTINUE
            DRV1(I)=(TRM1-TRM2)/DT
19.
        74 CONTINUE
            DO 75 I=1, JP
TRN1=(DRV1(I+1)+DRV1(I))/2.
20.
21.
            IF(I.GT.1)G0 T0 755
TRN2=DRV1(I)/2.
22.
23.
24.
25.
26.
            GO TO 575
       755 CONTINUE
            TRN2=(DRV1(I)+DRV1(I-1))/2.
27.
28.
       575 CONTINUE
            DRY2(I)=(TRN1-TRN2)/DT
        75 CONTINUE
29.
    BDIM=0.00000000001
30.
            10 726 I=1,JP
HUMP1=DRV0(I+1>-DRV0(I)
31.
32.
33.
            IF (HUMP1.GE. 0. 0)GO TO 726
            JPP=I+6 __
```

```
GD TD 727
    35.
36.
37.
          726 CONTINUE
         727 CONTINUE
       Симинининия
       C
   38.
39.
              IMAXV=0
              DV1MX=0.0
    40.
              DO 315 I=1,JPP
46.
              DV1MX=DRV1(I)
    47.
              IMAXY=I
    48.
         315 CONTINUE
   49.
              DMAX=DRYO(1)
             DO 53 I=2,JPP
IF(DRYO(I).LT.DMAX)GO TO 53
   50.
   51.
   52.
53.
             DMAX=DRYO(I)
              IMAXD=I
          53 CONTINUE
   55.
56.
57.
              IMAXV1=IMAXV-1
              DO 57 I=2, IMAXV1
              IF(DRY2(I).LE.0.0)DRY2(I)=(DRY2(I-1)+DRY2(I+1))/2.
IF(DRY2(I).LE.0.0)DRY2(I)=(DRY2(I-2)+DRY2(I+2))/2.
    58.
    59.
          57 CONTINUE
              IMAXVA=IMAXV+1
   60.
             DD 109 I=IMAXVA,JPP
IF(DRV2(I).GE.0.0)DRV2(I)=(DRV2(I-1)+DRV2(I+1))/2.
   61.
    62.
   63.
              IF(DRV2(I).GE.0.0)DRV2(I)=(DRV2(I-2)+DRV2(I+2))/2.
         109 CONTINUE
    64.
65.
                                                         斯斯斯斯米斯米斯米斯米斯斯斯米米斯斯米米斯
                                                         *****
   68.
         107 CONTINUE
   69.
             DD 789 IA=1,JPP
DERDTS=DRV1(IA+1)/DRV1(IA)
   70.
    71.
              IF(DEROTS.GT.0.0)GD TD 789
   72.
73.
             DZERO=DRV1(IA)-DRV1(IA+1)
             DDTZ=DRV1(IA)/DZERD
   74.
75.
76.
             NAFNL=IA
         GD TD 788
789 CONTINUE
         788 CONTINUE
             DO 889 IA=1, JPP
   78.
   79.
              ZEROTS=DRV2(IA+1)/DRV2(IA)
   80.
              IF (ZERDTS.GT. 0. 0)GD TD 889
              TZERD=DRY2(IA)-DRY2(IA+1)
   81.
             FDTZ=DRY2(IA)/TZERD
   82.
              IAFNL=IA
   83.
              DRORNG=DRYO(IA+1>-DRYO(IA)
   85.
              ZVALU0=DRV0(IA)+FDTZ*DR0RNG
             DRIRNG=DRV1(IA+1)-DRV1(IA)
   86.
             ZVALU1=DRV1(IA)+FDTZ#DR1RNG
   87.
         GD TD 888
889 CONTINUE
   88.
   89.
         888 CONTINUE
   90.
       Сжижжижжи
             WRITE(6,99)
   92.
          99 FORMAT(/30X, *FOR SIGNATURE*/)
             WRITE(6,76)
```

```
94.
         76 FURMAT (//1X,*
                                  INCREMENT
                                                 DEFLECTION
                                                                      YELDCITY
                                                                                      ACCEL
            1ERATION#/>
         WRITE(6,77)(I,DRV0(I),DRV1(I),DRV2(I),I=1,JPP)
77 FORMAT(1X,I14,3E15.5)
 95.
 96.
97.
      78 FORMAT(/15X,2I15/)
      Сжижжижжи
 98.
99.
             IDIFF=IMAXD-IMAXV
             IMAXF=0
100.
             AA1=0.05
101.
             ICC=0
      Сжжжжжжжжж
102.
            DO 104 ILP=1,40
      Сжжжжжжж
103.
             EQKA=AA1*PMAX/DMAX
104.
             AA3=-0.91
      Сжжж
105.
             DO 103 JJ=1,20
             IT=IMAXV
106.
107.
             EQMA=AA3*PMAX/DRY2(IMAXD)
108.
             CPMAX=0.0
             AA2=AA1+AA3
IF(AA2.LT.0.0)G0 T0 1021
109.
110.
      Сжжжж
111.
             IMAXC=IMAXD
112.
             IT=IT+1
113.
             DO 102 II=IT, IMAXC
EQCA=EQMA*DRV2(II)/DRV1(II)
             EQCA=-EQCA
115.
116.
117.
             EQVK=EQKA
             EQMASS=EQMA
             EQVC=EQCA
118.
119.
             MM=JPP
120.
             CALL EXACTI (MM, DT, EQYK, EQMASS, T, GG, EQYC)
121.
             DO 73 I=1, JPP
             D(I)=DRYO(I)
122.
123.
             V(1)=GG(1)
124.
         73 CONTINUE
125.
             N=JPP
             CALL NUCONY(N, Y,G, D, DT)
126.
     C
127.
             DO 508 MP=1,JPP
             DF(MP)=G(MP)#EQMASS
128.
129.
        508 CONTINUE
            DO 509 MA=1, JPP
DF(MA)=(DF(MA)+DF(MA+1))/2.
130.
131.
132.
        509 CONTINUE
133.
             CPMAX=0.
134.
             DO 501 M=1, JPP
135.
             DF(M)=(DF(M)+DF(M+1))/2.
136.
137.
            IF(DF(M).GT.CPMAX)GO TO 20
GO TO 502
138.
         20 CPMAX=DF(M)
139.
             ICF=M
140.
        502 CONTINUE
141.
        501 CONTINUE
142.
             ERFCE=CPMAX-PMAX
143.
             ABERFG=ABS(ERFCE)
             IF(ABERFG.GT.500.0)GD TD 102
144.
145.
             ICC=ICC+1
     C
146.
             DFRNG=DF(IAFNL+1)-DF(IAFNL)
             ZVALUF=DF(IAFNL)+FDTZ*DFRNG
CMAXV=(ZVALUF-EQKA*ZVALU0)/ZVALU1
147.
148.
149.
             CDIFFR=(CMAXV-EQCA)/CMAXV
```

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```
150.
              ABCD=CDIFFR
151.
              QC(ICC)=CDIFFR
              IF(ICC.EQ.1)GD TD 102.
CRATID=QC(ICC)/QC(ICC-1)
152.
153.
154.
              IF(CRATID.GT.0.0)GD TD 102
155.
              FCHK=EQKA*DRV0(II)
             DIFCHK=FCHK-DF(II)
DFCHKR=DIFCHK/FCHK
156.
157.
158.
              FPFRAC=DF(II)/PMAX
159.
              ICDPT=II
160.
              IBBB=IMAXD-ICDPT
161.
162.
163.
              BB1=FLOAT(IBBB/IDIFF)
TRMM=EQMA*DRY2(II)
TRMC=EQCA*DRY1(II)
              TRMK=EQKA*DRV0(II)
164.
165.
              CPMXX(II)=TRMM+TRMC+TRMK
166.
167.
              WRITE(6,847)
        847 FORMAT(//10X,*GDDD RUN*//)
GD TD 846
102 CONTINUE
168.
169.
       1021 CONTINUE
170.
              AA3=AA3+0.045
171.
      Сжжжжжжжжжжж
172.
173.
174.
175.
         103 CONTINUE
         AA1=AA1+0.025
       1000 FORMAT(/1X, *000*)
        844 FORMAT(1X,7F10.2,3E15.5)
845 FORMAT(6F15.4,10X,2I5,F10.3)
176.
              WRITE(6,1002)
178.
179.
       1002 FORMAT(/10X, *OUTER LOOP COMPLETED*/)
180.
        846 CONTINUE
181.
      C
182.
              WRITE(6, 1000)
183.
              AFRNG=DF(NAFNL+1)-DF(NAFNL)
              DVALUF=DF(NAFNL)+DDTZ*AFRNG
184.
      ZVALUR=ZVALUF/PMAX
C******
185.
186.
             DVALZD=DVALUF/PMAX
      Сжжжжж
187.
             FDRTID=ABERFG/PMAX
188.
             DENOM=FLOAT (IMAXD)
189.
              DENOM1=1.
190.
              TCRSPT=FLOAT(II)/DENOM
              TZROA=FLOAT(IAFNL)/DENOM
191.
192.
       1199 FORMAT(/10X,11F11.3/)
193.
194.
195.
196.
197.
198.
       1001 FORMAT(10X, 15, 7F15.5)
              CALL EXACTI (JP, DT, EQKA, EQMA, T, GG, EQCA)
              DO 733 I=1, JP
              D(I)=DRVO(I)
        V(I)=GG(I)
733 CONTINUE
              N=JP
200.
              CALL NUCONV(N, V, G, D, DT)
201.
              DO 518 MP=1, JP
202.
              DF(MP)=G(MP)*EQMA
203.
         518 CONTINUE
204.
      C
205.
          94 CUNTINUE
              RETURN
206.
207.
              END
```

FOR SIGNATURE

INCREMENT	DEFLECTION	VELOCITY	ACCELERATION
1	1.27667E-04	1.92708E-03	9.93490E-03
2	1.54167E-04	7.94792E-04	8.31380E-03
3	1.91250E-04	1.03125E-03	6.69271E-03
4	2.36667E-04	1.33021E-03	1.03906E-02
5	2.97667E-04	1.86250E-03	1'.57552E-02
6	3.85667E-04	2.59063E-03	2.10677E-02
7	5. 04917E-04	3.54792E-03	2.91276E-02
2345678	6.69500E-04	4.92083E-03	3.62500E-02
9	8.98583E-04	6.44792E-03	4.09245E-02
10	1.18533E-03	8.19479E-03	4.38151E-02
īi	1.55417E-03	9.95312E-03	3.51432E-02
iż	1.98158E-03	1.10062E-02	9.81771E-03
13	2.43467E-03	1.07385E-02	-3,29948E-02
14	2.84067E-03	8.36667E-03	-8.32943E-02
15	3.10400E-03	4.07500E-03	-1.15703E-01
16	3.16667E-03	-8.89583E-04	-1.14401E-01
17	3.03283E-03	-5.07708E-03	-8.50130E-02
18	2.76050E-03	-7.69062E-03	-4.40234E-02
19	2.41758E-03	-8.59896E-03	-4.75260E-03
έó	2.07258E-03	-8.07083E-03	-9.53125E-03
21	1.77192E-03	-6.89896E-03	7.39583E-03
55	1.52067E-03	-5.80104E-03	2.50000E-03
22	1.050015-03	-3.001046-03	L. 30000E-03

GODD RUN

00

SITE ONE VEHICLE 2

EQUIV INPUT IN LBS	TDT FUNCTION	RESPONSE IN FT
7.26622E+03	2.30853E-02	2.02000E-04
9.67190E+02 3.38946E+03	2.76124E-02 2.56125E-02	2.68500E-04 3.50500E-04
6.28019E+03	2.17664E-02	5.07583E-04
1.20998E+04	1.78084E-02	8.30917E-04
2.26201E+04	1.43096E-02	1.45967E-03
3.73522E+04	1.13959E-02	2.51742E-03
5.36091E+04 6.66077E+04	9.03463E-03 7.14611E-03	4.03258E-03 5.86650E-03
5.49146E+04	5.64567E-03	7.24833E-03
-4.40640E+03	4.45756E-03	6.66300E-03
-1.88870E+04	3.51838E-03	5.11650E-03
1.76576E+04	2.77663E-03	4.47067E-03
4.69750E+04	2.19108E-03 1.72893E-03	5.00992E-03 6.88358E-03
8.37103E+04 6.44724E+04	1.36424E-03	8.41667E-03
-1.57651E+04	1.07645E-03	7.41800E-03
-2.57654E+04	8.49373E-04	5.45308E-03
-5.79904E+03	6.70194E-04	3.97867E-03
-1.63126E+03	5.28813E-04	2.96208E-03
1.61962E+03	4.17257E-04 3.29234E-04	2.31008E-03 1.93375E-03
4.38986E+03 4.45170E+03	2.59780E-04	1.69475E-03

5.24889E+03	2.04977E-04	1.55192E-03
5.77634E+03	1.61736E-04	1.47250E-03
4.82121E+03	1.27617E-04	1.39683E-03
5.39067E+03	1.00695E-04	1.34767E-03
6.05333E+03	7.94527E-05	1.33158E-03
4.47763E+03	6.26916E-05	1.28433E-03
4.63410E+03	4.94664E-05	1.23733E-03
5.15415E+03	3.90311E-05	1.21075E-03
5.09242E+03	3.07972E-05	1.19233E-03
3.93199E+03	2.43003E-05	1.14658E-03
5.04313E+03	1.91740E-05	1.12867E-03
5.26145E+03	1.51291E-05	1.12800E-03
4.01299E+03	1.19375E-05	1.09825E-03
4.74872E+03	9.41922E-06	1.08333E-03
6.32678E+03	7.43217E-06	1.11892E-03
2.59107E+03	5.86430E-06	1.06242E-03
5.12136E+03	4.62719E-06	1.05375E-03
5.51294E+03	3.65105E-06	1.07242E-03
4.25295E+03	2.88083E-06	1.06250E-03
4.31075E+03	2.27310E-06	1.04625E-03
5.95350E+03	1.79358E-06	1.07567E-03
4.67396E+03	1.41521E-06	1.08050E-03
3.94255E+03	1.11666E-06	1.05650E-03
4.68328E+03	8.81093E-07	1.04683E-03

NEXT SITE STD VEHICLE

FOR SIGNATURE

DEFLECTION	VELOCITY	ACCELERATION
9.95833E-05	1.52187E-03	7.77344E-03
1.21750E-04	6.21875E-04	6.71224E-03
1.49333E-04	8.09375E-04	5.65104E-03
1.86500E-04	1.07396E-03	6.19792E-03
2.35250E-04	1.30521E-03	5.92448E-03
2.90917E-04	1.54792E-03	7.12240E-03
3.59083E-04	1.87500E-03	8.54167E-03
4.40917E-04	2.23125E-03	9.60937E-03
'5.37583E-04	2.64375E-03	1.25000E-02
6.52417E-04	3.23125E-03	1.47917E-02
7.96083E-04	3.82708E-03	1.32161E-02
	4.28854E-03	7.79948E-03
1.13917E-03	4.45104E-03	-6.17187E-03
	3.79479E-03	-2.66797E-02
		-4.29948E-02
		-4.88021E-02
		-3.80990E-02
		-1.61849E-02
		-1.82292E-03
		-5.88542E-03
		-7.35677E-04
9.22417E-04	-2.56146E-03	-7.16146E-04
	9.95833E-05 1.21750E-04 1.49333E-04 1.86500E-04 2.35250E-04 2.90917E-04 3.59083E-04 4.40917E-04 5.37583E-04 6.52417E-04 7.96083E-04 9.58583E-04	9.95833E-05 1.52187E-03 1.21750E-04 6.21875E-04 1.49333E-04 8.09375E-04 1.86500E-04 1.07396E-03 2.35250E-04 1.30521E-03 2.90917E-04 1.54792E-03 3.59083E-04 1.87500E-03 4.40917E-04 2.23125E-03 5.37583E-04 2.64375E-03 7.96083E-04 3.23125E-03 7.96083E-04 4.28854E-03 1.13917E-03 4.45104E-03 1.31467E-03 3.79479E-03 1.44275E-03 2.31667E-03 1.50000E-03 3.55208E-04 1.47117E-03 -1.58750E-03 1.37300E-03 -2.69271E-03 1.25575E-03 -2.88229E-03 1.02867E-03 -2.75000E-03

GOOD RUN

EQUIV INFUT IN LBS	TDT FUNCTION	RESPONSE IN FT
2.39378E+04	.2.79870E-02	1.57565E-04
-5.99763E+04	4.02600E-02	-1.68119E-04
6.45929E+03	4.45935E-02	-2.74326E-04
1.36858E+04	4.49856E-02	-2.24518E-04
2.55768E+04	4.34868E-02	-2.40447E-05
3.73268E+04	4.11442E-02	3.17975E-04
5.95647E+04	3.84877E-02	8.60932E-04
7.51046E+04	3.57813E-02	1.58148E-03
8.03293E+04	3.31525E-02	2.39395E-03
6.33737E+04	3.06592E-02	3.10814E-03
-4.96779E+03	2.83238E-02	3.29894E-03
-1.96412E+04	2.61510E-02	3.13688E-03
1.84098E+04	2.41370E-02	3.06349E-03
4.64276E+04	2.22741E-02	3.21925E-03
6.79205E+04	2.05529E-02	3.61990E-03
3.74124E+04	1.89636E-02	3.92125E-03
-3.11110E+01	1.74966E-02 1.61429E-02	3.91757E-03 3.90907E-03
2.12723E+04 5.20213E+03	1.48937E-02	3.79282E-03
1.06867E+03	1.37411E-02	3.60242E-03
-1.67619E+03	1.26777E-02	3.36579E-03
-3.75658E+03	1.16966E-02	3.10234E-03
-4.38155E+03	1.07913E-02	2.83187E-03
-6.42824E+03	9.95617E-03	2.55472E-03
-9.40418E+03	9. 18563E-03	2.26518E-03
-6.84531E+03	8.47472E-03	1.99745E-03
-6.04811E+03	7.81883E-03	1.75536E-03
-7.78484E+03	7.21371E-03	1.52313E-03
~4.78390E+03	6.65541E-03	1.32403E-03
-2.34767E+03	6.14033E-03	1.16421E-03
~3.53550E+03	5.66511E-03	1.02124E-03
-8.24185E+02	5.22666E-03	9.09508E-04
3.66539E+03	4.82215E-03	8.46376E-04
5.42162E+03	4.44895E-03	8.20303E-04
-7.56887E+03	4.10463E-03	7.27339E-04
7.55546E+03	3.78696E-03	7.05572E-04
1.67329E+03	3.49387E-03	6.79791E-04
4.12036E+03	3.22347E-03	6.69173E-04
2.13777E+03	2.97399E-03	6.53120E-04
1.94469E+03	2.74383F-03	6.33810E-04
3.01993E+03	2.53147E-03	6.20752E-04
6.73974E+03	2.33555E-03	6.35642E-04
2.8388E+03	2.15480E-03	6.37603E-04
4.24396E+03	1.98303E-03	6.42583E-04
3.23174E+03	1.83417E-03	6.42152E-04
6.8666E+03	1.69222E-03	6.63086E-04
2.46450E+03	1.56125E-03	0

PREDICTED	MEASURED
.00016 00017 00027 00022 00002 .00032 .00086 .00158 .00239	.00018 .00028 .00044 .00068 .00107 .00165 .00252 .00357 .00404
.00330	.00318

00314	.00289
.00306	.00284
. 00322	.00314
.00362	.00382
. 00392	.00433
.00392	.00399
.00391	.00320
.00379	.00244
.00360	.00178
.00337	.00128
.00310	.00091
.00283	. 00067
.00255	. 00051
.00227	.00041
	.00035
.00200	
.00176	. 00032
.00152	.00029
.00132	. 00027
.00116	. 00025
.00102	.00014
.00091	.00012
.00085	.00012
.00082	.00012
.00073	.00012
.00071	.00012
.00068	.00011
.00067	.00012
.00065	.00011
.00063	.00011
.00062	.00011
. 00064	.00011
.00064	. 00011
.00064	.00011
.00064	.00011
.00066	.00011
0	.00011
U	. 00011

APPENDIX D

DATA DIGITIZED

FROM LIGHT

SENSITIVE PAPER

The data listing has the following format

- Card 1 A comment card giving the base name, site number, grid location of measurement point, Load Cart data of test, distance from closest measurement gage to edge of tire track in inches, time increment at which data was digitized and estimated number of cards.
- Card 2 Distance (ft) of measurement point from edge of tire print where Gage 1 is the closest.
- Card 3 Number of cards actually read, time increment used, and weight of load cart (25 kips).
- Card 4 to (n-1) Data digitized and listed (from left to right) as,

 Gage 1, Gage 2, Gage 3, Gage 4, and Gage 5.
- Card n (last card) Calibration data for Gages 1 to 5 also listed from from left (Gage 1) to right (Gage 5).

```
.9375
.200 25000.0
0071 0102
0074 0
                                       LOC A-1
1.6875
      LECIN
                                                         LD. CT.
                                                                          3.11.76
                                                                                            7.25 .2
                                                                                                             48
       -542
                                                        2.6875
                                                                         3.6875
     48
                                         0134
0027
          0045
0030
0034
          0048
0051
                                         0134
                              0110
0117
                                         0136
0040
0049
          0058
0065
                    0085
                                         0137
                                        0141
0145
0151
                    0094
                               0126
                    0107
0122
0146
0172
                              0138
0154
0175
0198
0223
          0078
0092
0061
0077
0103
          0115
                                         0158
0130
          0141
                                         0165
0167
          0174
                    0203
                                         0173
          0211
0254
0297
                    0234
0272
0306
                              0247
0277
0301
0210
0259
                                        0181
0190
0307
                                         0195
                              0324
0344
0358
0366
0367
                    0340
0371
                                         0206
0360
          0340
0414
0456
0487
0490
          0382
          0420
0447
0456
0459
                    0394
0411
0415
0419
0418
                                         9020
                                         0209
                                         0209
0483
                               0371
                                         9050
0461
          0451
                               0375
                                        0213
          0429
0402
0374
                    0403
0385
0363
                              0367
0354
0338
0427
0393
                                         0212
                                         0209
0359
                                         0204
          0349
                    0344
                               0324
                                         0202
0331
                    0320
0299
0273
0249
0302
          0324
                               0305
                                         0194
          0301
0273
0250
0223
                              0288
0267
0247
0222
0277
0248
0225
                                         0189
                                        0182
                                         0175
0198
0171
                    0223
                                         0167
                              0196
0167
0143
          0196
                    0196
                                         0156
0141
0115
          0163
0137
                    0163
0137
                                        0147
0140
                    0117
                              0126
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0096
0085
          0117
                                         0136
                                        0134
0133
          0105
                    0104
0076
          0095
                    0095
                               0109
0071
0067
0064
                                        0131
          0089
                    0089
                               0103
          0084
0080
                              0099
0096
                                        0131
                    0084
                    0080
                                         0130
                    0078
0074
0073
0072
0072
          0078
0074
0060
                               0094
                                         0129
                                         0128
0058
                               0092
          0073
0072
0057
                               0091
                                         0128
0055
0055
                                        0128
                              0090
          0071
0070
                                        0128
0127
                              0090
0053
0053
                    0070
                               0089
          0069
                    0069
                               8800
                                         0128
0053
          0069
                    0069
                              0087
                                         0127
0051
          0069
                    0069
                              0087
                                        0128
0051
                    0069
                              0087
                                         0127
          0069
                      .00006995
                                          .00004052
  .00008065
                                                               .00002212
                                                                                   .00002778
```

画物後の単元のできぬ三次の次を画の形式の音のの次数

```
EGLIN
                   SITE 1
                                 LOC A-1
                                              LD. CT.
                                                            3.11.76
C
                                                                         .7.00 . _
                                                                                       23
                   .91666
                               1.83333
                                             2.83333
                                                          3.83333
     .58333
    53
             .200 25000.0
0023
        0052
                0075
                         0107
        0057
                 0079
0028
                         0108
                                 0135
        0067
                                 0136
0034
                0090
                        0117
                        0127
                                 0139
0042
        0077
                0102
0052
        0088
                0115
                         0140
                                 0143
                                 0149
0153
0158
0065
        0104
                0130
                         0156
        0121
0080
                0150
                         0173
                0167
0096
        0138
0158
                         0189
                        0203
0113
                0185
                                 0164
0135
        0180
                8050
                         0555
                                 0169
                0229
0160
        0206
                         1450
                                 0174
                0251
0271
0299
0321
                        0259
0272
0291
0184
0211
0249
        0231
                                 0179
        0256
0292
0325
                                 0183
                                 0188
0288
                         0305
                                 0191
0325
        0357
                0342
                         0316
                                 0193
0364
        0388
                0359
                         0326
                                 0195
0401
0433
0453
0473
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0447
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0388
                        0333
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0458
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0458
        0446
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                        0334
0323
                0384
0370
        0430
0437
        0411
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        0389
                0351
                        0308
                                0192
                        0292
0273
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        0367
                0331
                                0187
0386
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0267
        0344
                0309
                                0182
        0316
                0284
                                 0176
                0257
0232
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                                 0170
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        0260
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                0207
        1150
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                0115
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                0100
                                 0134
0100
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                        0094
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0089
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                0080
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0059
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0070
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                                 0125
                        0064
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        0073
0069
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                        0063
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                                  .00004052
                                                  .00002212
                                                                  .00002778
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.45833
38
                                                                                     3.11.76
                           SITE 1
                                               LOC A-2
                                                                   LD. CT.
                                                                                                           5.50 .2
                                                                                                                                38
           15833 .79167
.200 25000.0
0070 0100 0131
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                                                                  2.70833
0042
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0044
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                        0101
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0286
            0301
                        0307
           0263
0225
0190
                       0281
0250
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0195
                                    0446
0395
                                    0342
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            0087
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                                                                       .00002212
 .0001613
                                                 .00002778
```

SCHOOL STATE OF STATE STATES

```
.58333
45
        08333 .91666
.200 25000.0
                   SITE 1
                                LUC A-2
                                              LD. CT.
                                                            3.11.76
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                                                                                       45
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                                             2.83333
                                                           3.83333
9200
                                 0145
        0055
9500
                 0085
                         0124
                                 0147
        0057
0059
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0067
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                        0131
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0153
0027
0030
                 0091
0032
                 0096
                         0148
                                 0156
                        0159
0175
0035
                 0101
                                 0160
0041
        0074
                0109
                                 0166
0050
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0261
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        0266
0279
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                         0403
                0270
                         0412
                0273
0273
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0232
0286
        0280
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0281
        0275
                         0418
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                0245
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                0205
                                 0190
        0167
                0187
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の大学の対象を見られている。とのできないなどは

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0183 0180 0171 0204 0214

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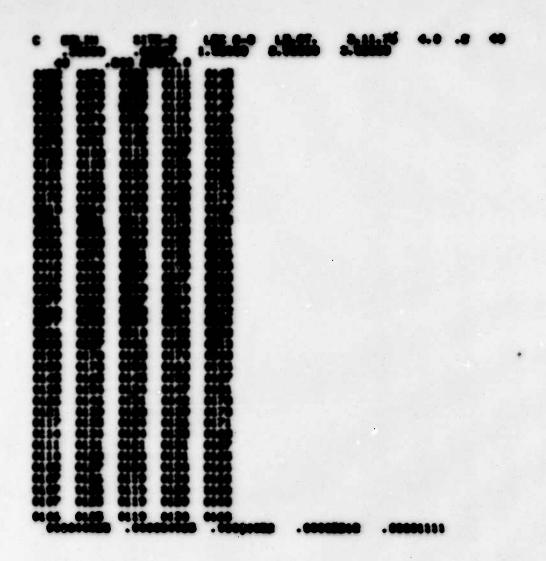
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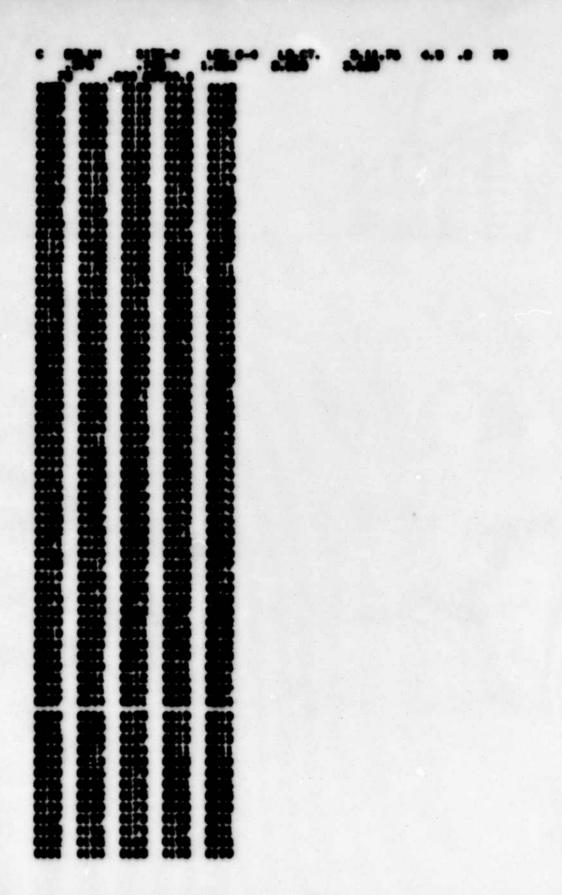
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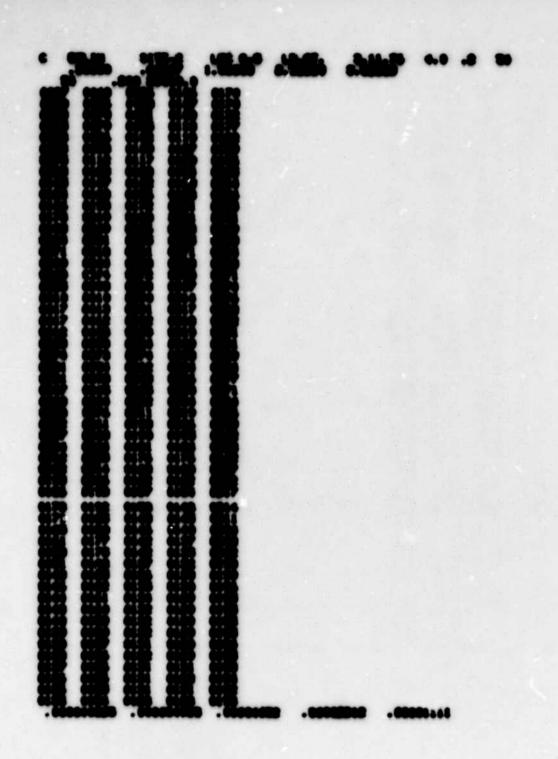
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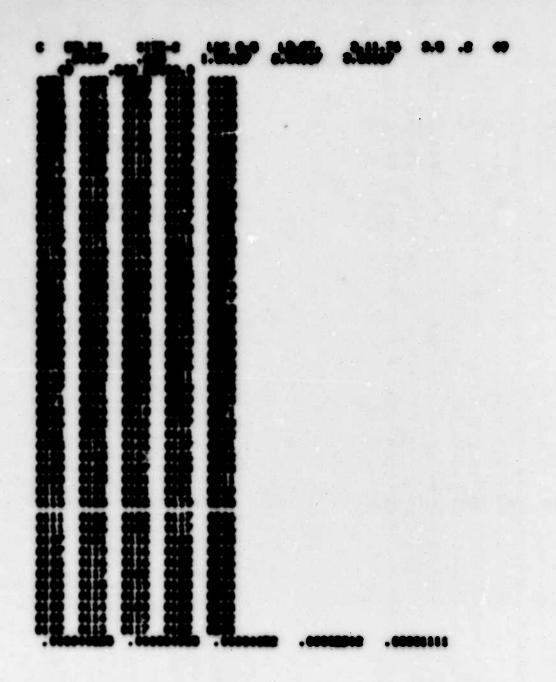
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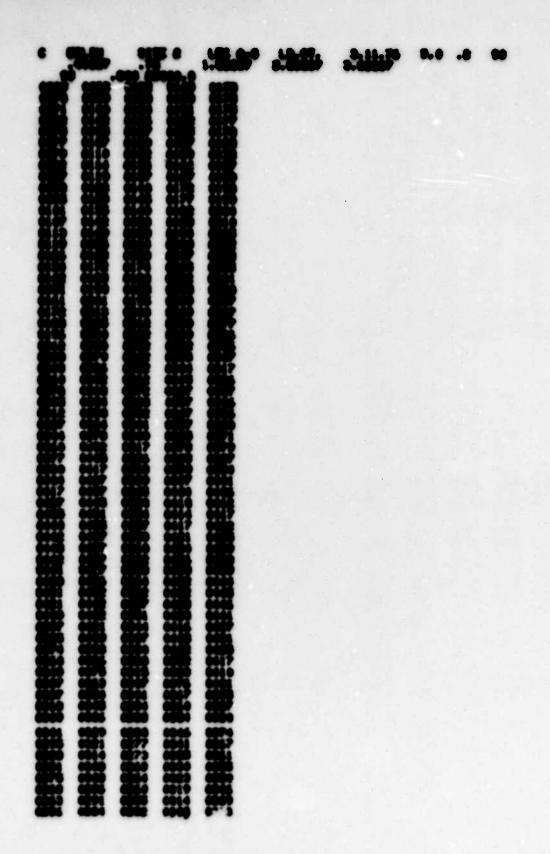
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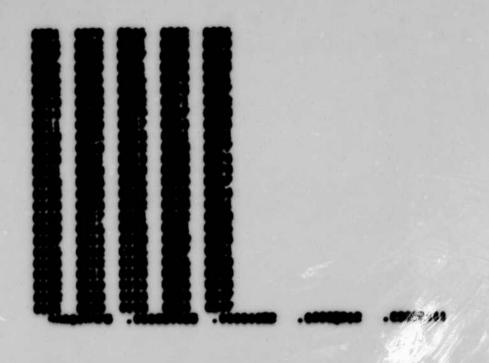


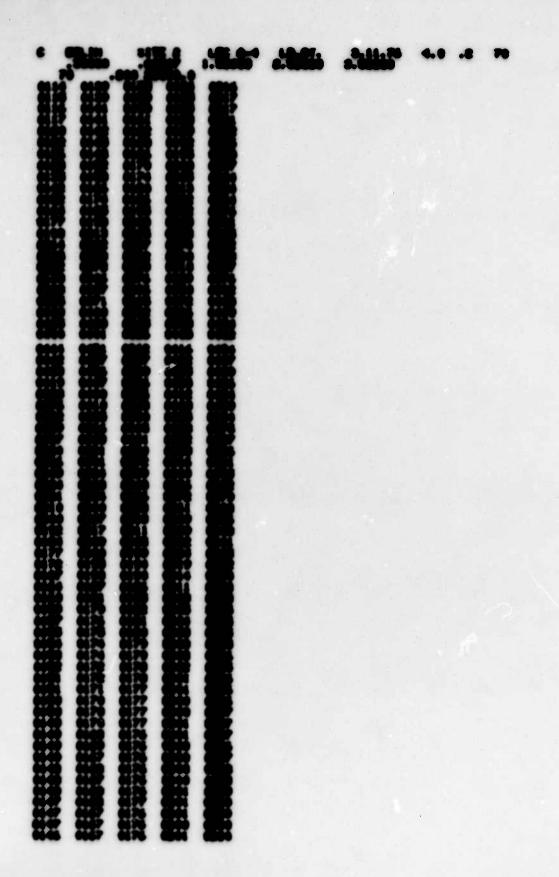




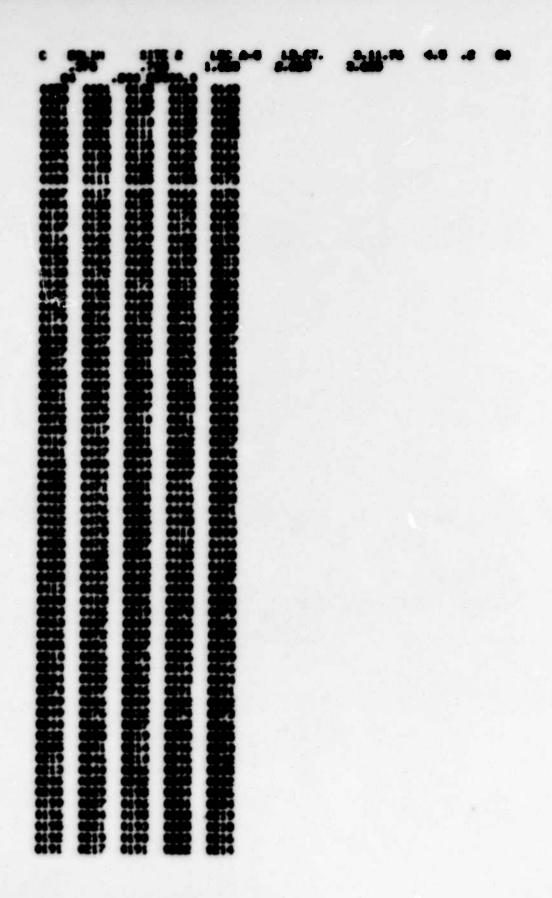


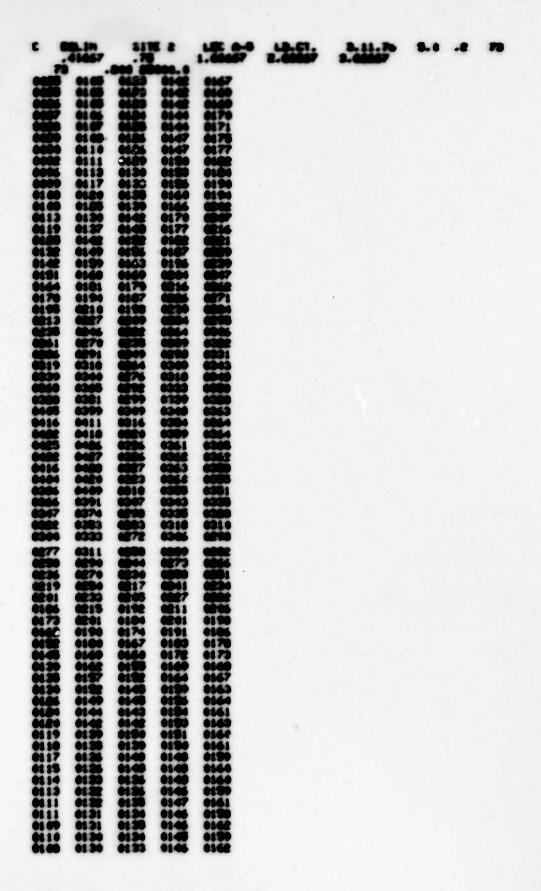


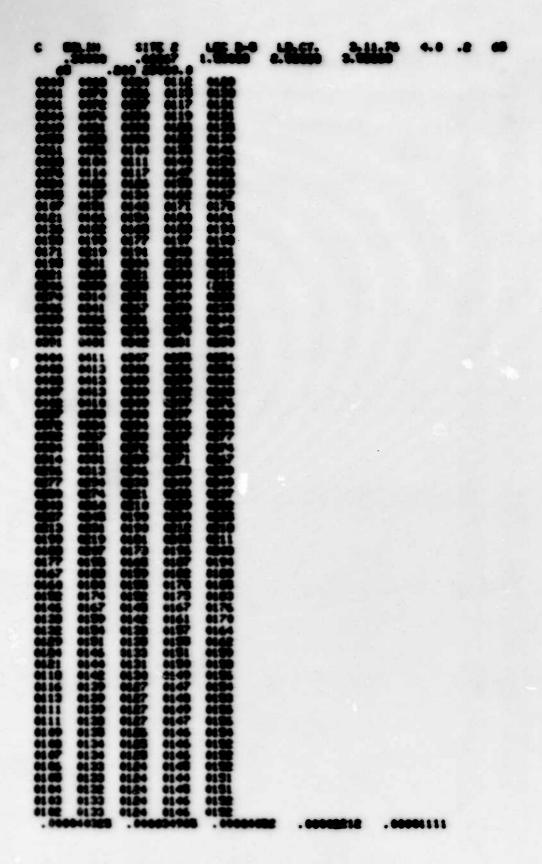


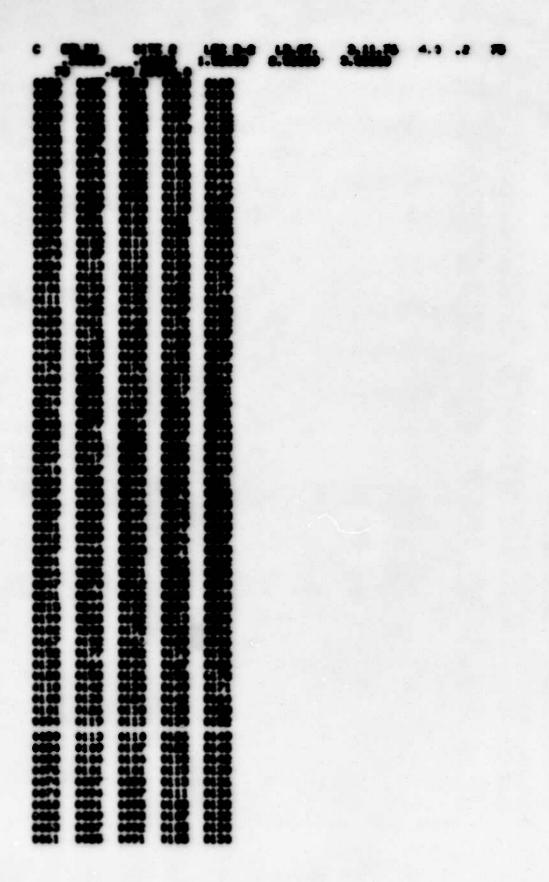


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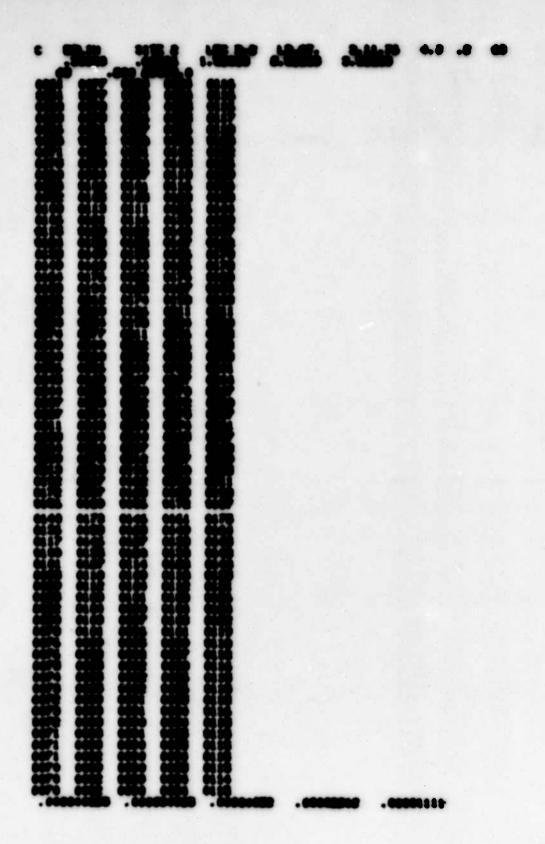


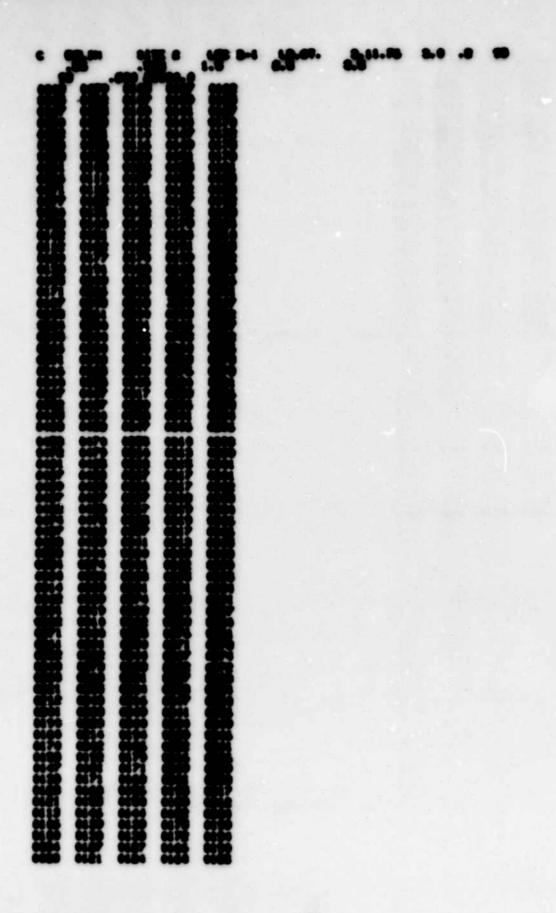


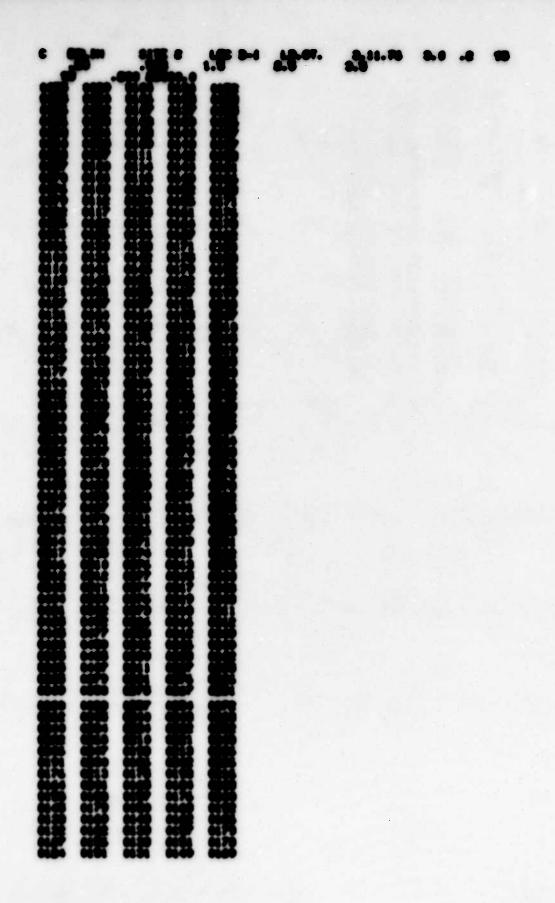


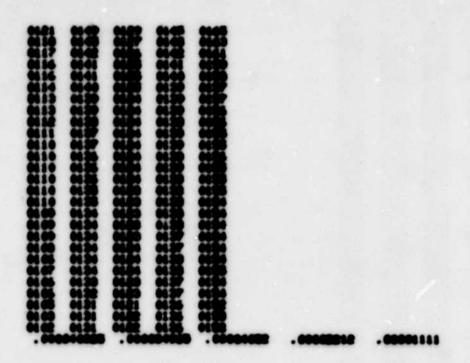


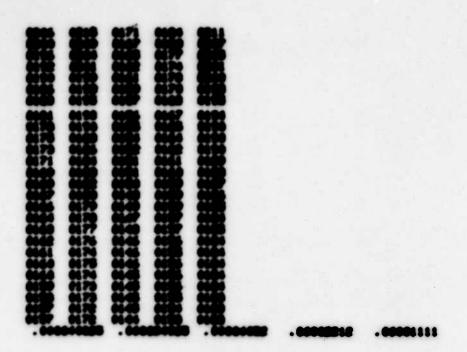
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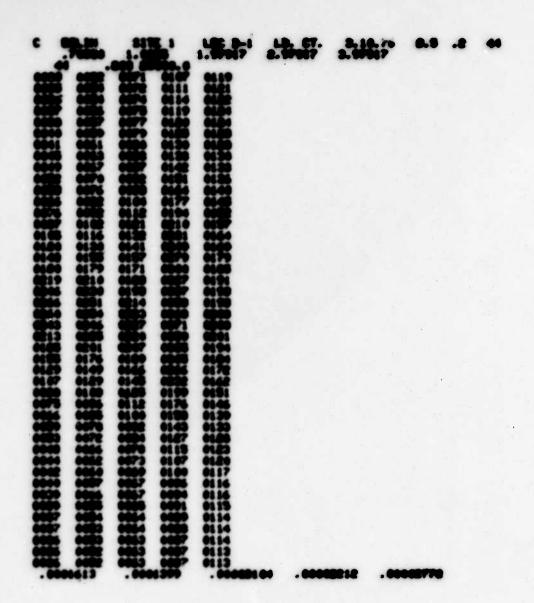


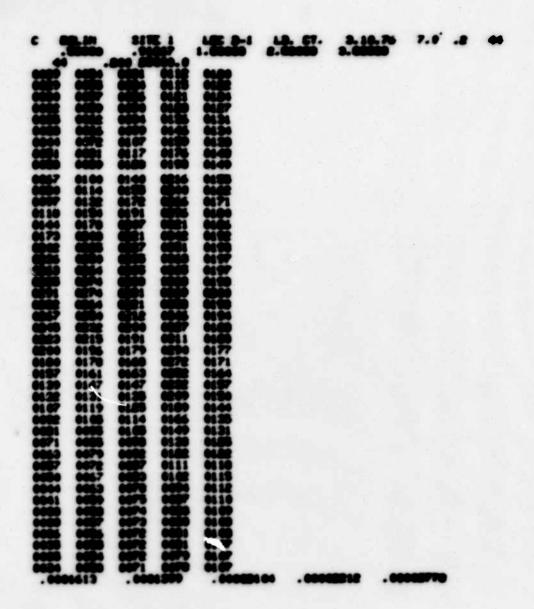


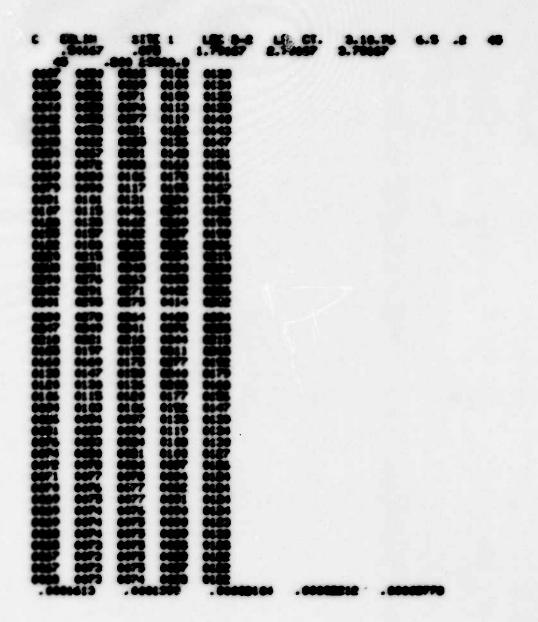


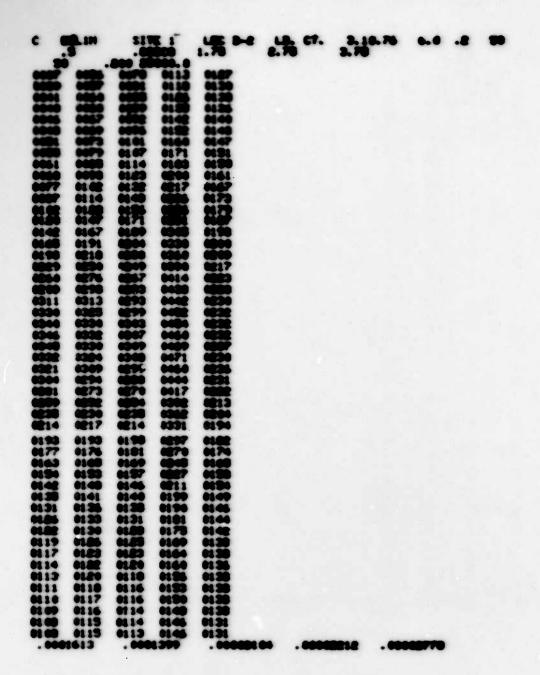




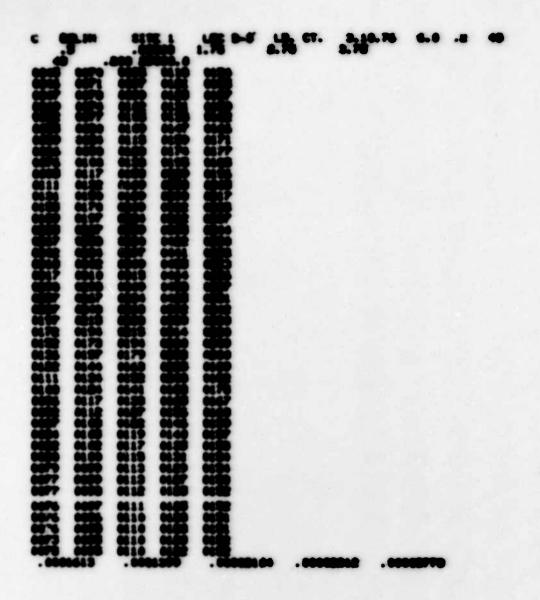


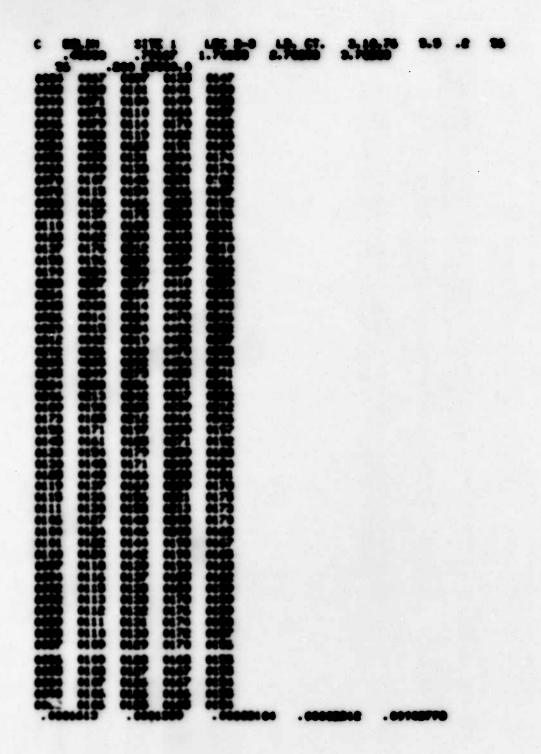


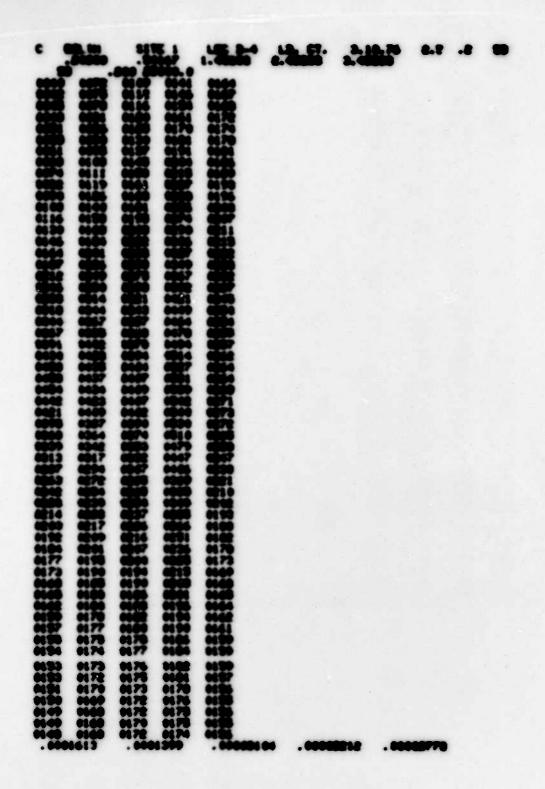


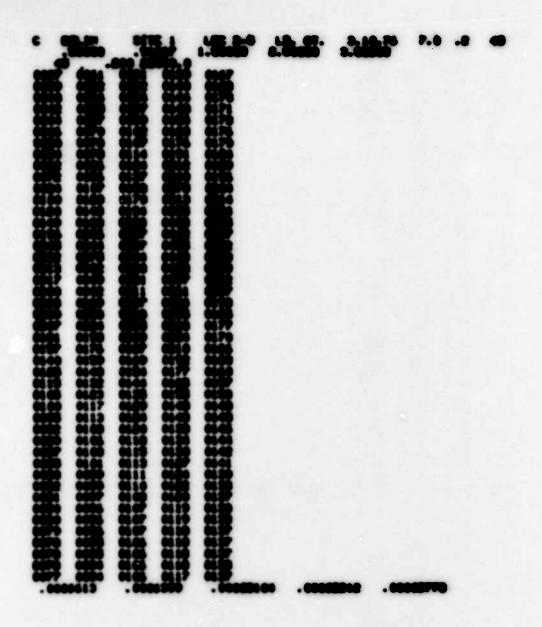


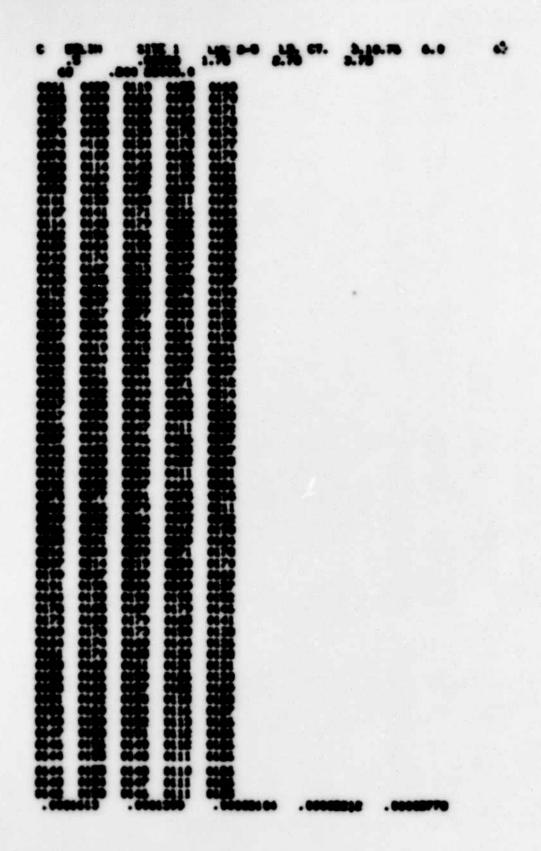
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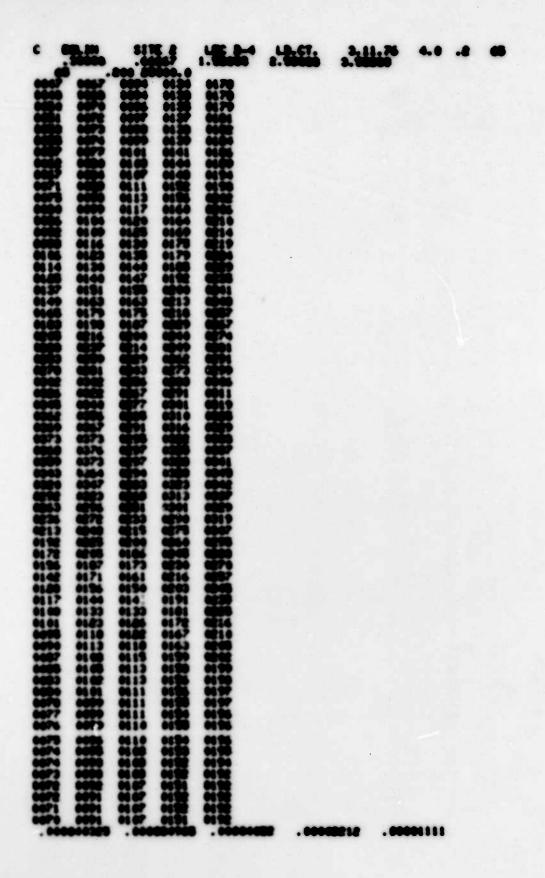


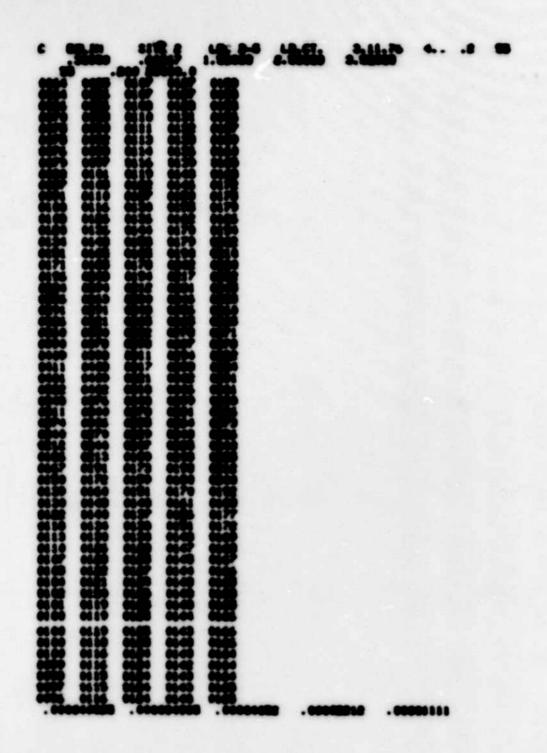


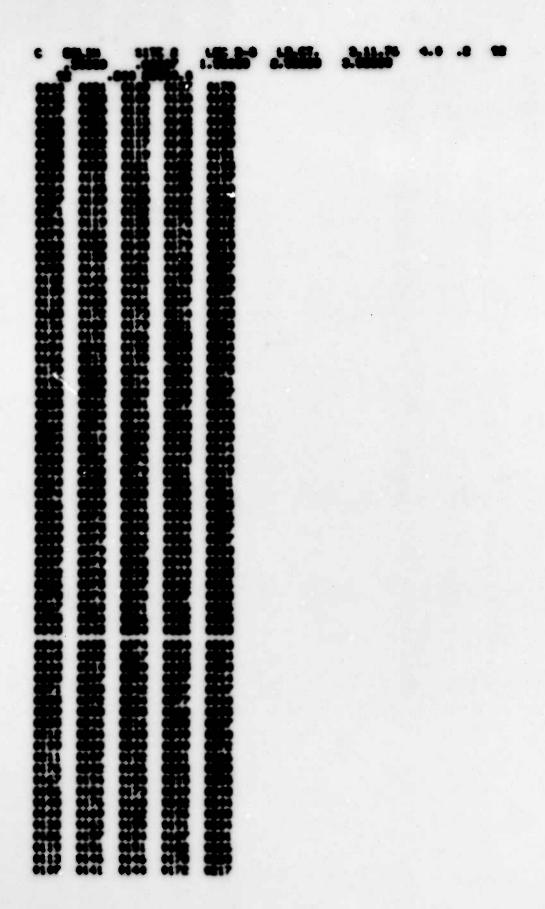


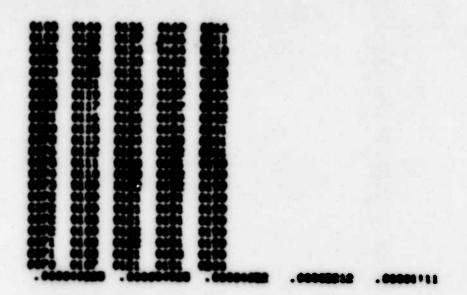


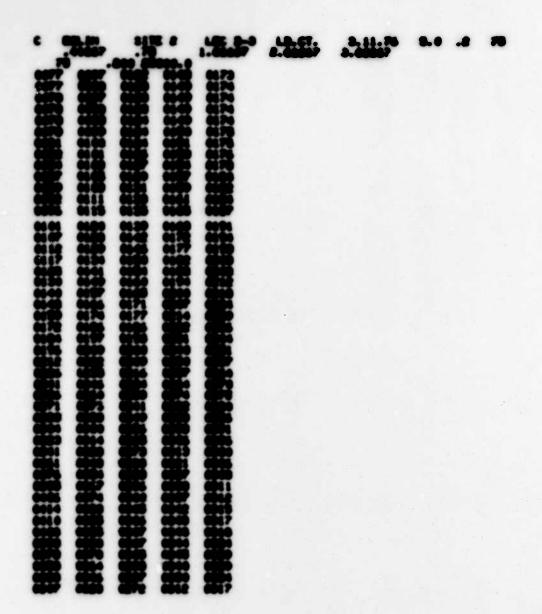


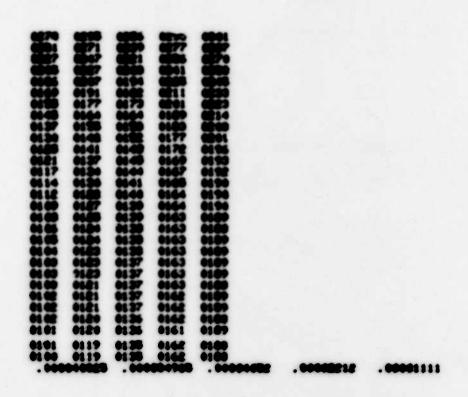


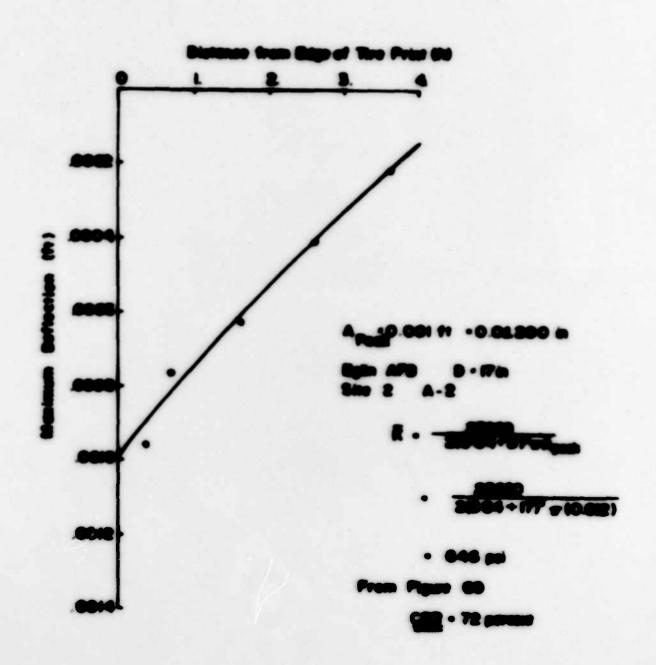












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